

SPACE SHUTTLE RUDDER/SPEEDBRAKE
SUBSYSTEM ANALYSIS

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CONTROL SYSTEMS DEVELOPMENT DIVISION



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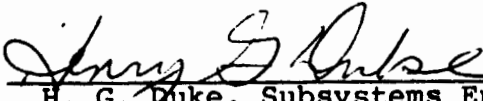
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TECHNICAL REPORT INDEX/ABSTRACT (See instructions on reverse side.)

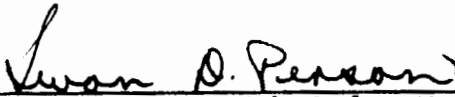
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13. ABSTRACT This report describes the CSMP program, its uses, some limitations and its application to the R/SB subsystem model. This report also analyzes the Space Shuttle R/SB subsystem using the CSMP to evaluate the Rockwell math model contained in Rockwell publication SD 74-SH-0324.			
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SPACE SHUTTLE RUDDER/SPEEDBRAKE
SUBSYSTEM ANALYSIS

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LIST OF ABBREVIATIONS AND ACRONYMS

ASA	Aerosurface Servo Amplifier
CPU	Central Processor Unit
CSMP	Continuous System Modeling Program
DSBRHL	Degrees Speedbrake in Rudder Hinge Line
FCS	Flight Control System
GPM	Gallons Per Minute
HM	Panel Hinge Moment
HMR	Panel Hinge Moment of Rudder
HMSB	Panel Hinge Moment of Speedbrake
HP	Hewlett Packard
JCL	Job Control Language
KHST	Servo Value Torque Motor Hysteresis
LPDEG	Left Panel Degrees
MDM	Multiplexer/Demultiplexer
PDU	Power Drive Unit
RHL	Rudder Hinge Line Reference System
RIN	Rudder Input Command
RPDEG	Right Panel Degrees
R/SB	Rudder and/or Speedbrake
RPM	Revolutions Per Minute
SIN	Input Speedbrake Command

1. SUMMARY

This report presents the analysis of the Space Shuttle Rudder/Speedbrake (R/SB) subsystem using the Continuous System Modeling Program (CSMP) to evaluate the Rockwell math model contained in Rockwell publication SD 74-SH-0324 (reference 1). The R/SB subsystem fits into the overall avionics system as depicted in figure 1-1.

The report describes the CSMP program, its uses, some limitations and its application to the R/SB subsystem model. The appendices contain definitions of the constants and variables used in the program. The report highlights three (3) areas of analysis: 1) step response, 2) ramp response, and 3) the delay time or deadspace observed in system response. Data obtained using the CSMP program was further processed in a continuous format in a manner similiar to that shown in figure 1-2.

The step response is used to evaluate three (3) factors: 1) the linearity of the output response, 2) the accuracy of the output response, and 3) the response of the system to a step command. Various step commands were separately addressed first to the Rudder, then to the Speedbrake. When one channel was driven the other was set to zero or null.

The Rudder displayed a 1 percent accurate output response for all three commands, thus demonstrating acceptable linearity and accuracy. The Speedbrake did not, however, meet reasonable standards since accuracy and linearity were in error in excess of 6 percent. The slew rate of both systems was fast enough to meet maximum software speed requirements of 12 deg/sec.

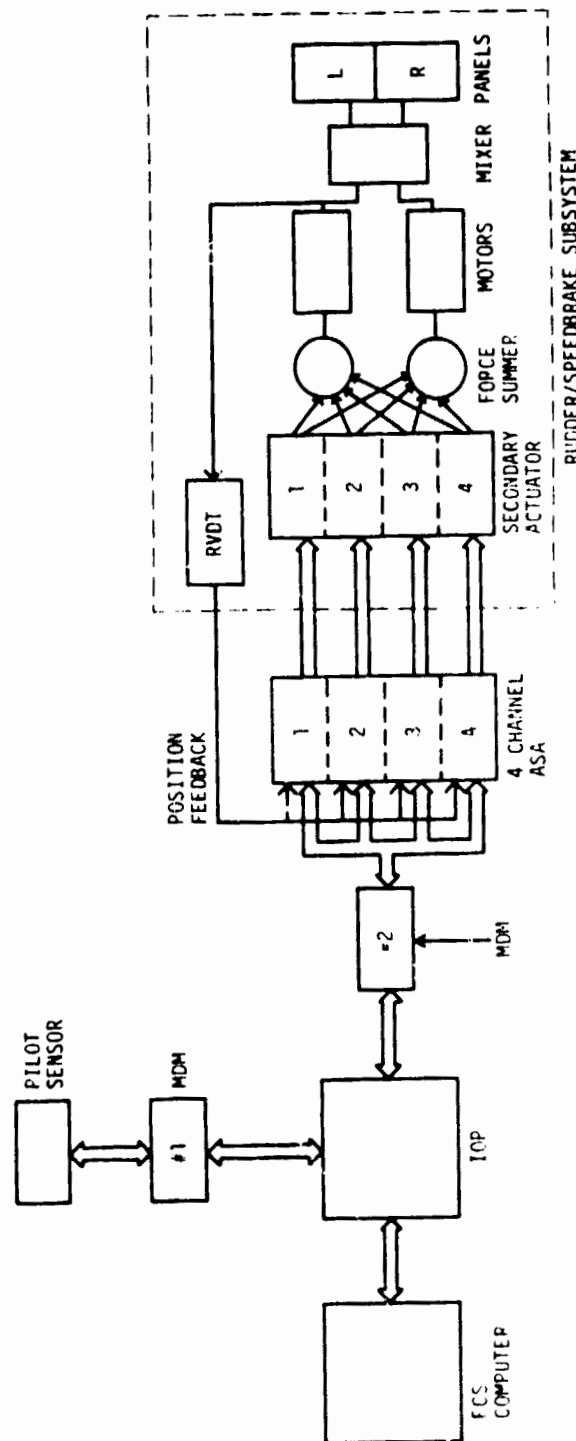
The ramp command was used to evaluate two factors: 1) the tracking ability of both the Rudder and Speedbrake, and 2) the

ability of the hydraulic system to meet worst case specifications. It was found that after initial system dynamics had settled out, the system output followed the input with an undetectable graphical error in both Rudder and Speedbrake response. It was also found that the R/SB hydraulic motors were indeed capable of meeting the specification supplied from Rockwell and that their performance would not overtax the hydraulic system.

The deadspace or system delay time to input command was found to possibly be longer than expected but since there does not appear to be any specification on this parameter, it was difficult to judge.

Of the four contributors to the deadspace (the Servo valve flapper motor, the Summer and Mixer gear trains, and the Power Drive Unit (PDU) gear train), it was found that the PDU gear train contributed the most to the deadspace. The PDU gear train was responsible for over 57 percent of the delay.

It was established during the analysis that the deadspace was command-dependent, i.e., as the command rate increased, the deadspace decreased. A 10-deg/sec Rudder command exhibits a deadspace of 75 milliseconds (ms) whereas a 5-deg/sec Rudder command has a deadspace of 125 ms. This report contains methods for the determination of both Rudder and Speedbrake deadspace for various input ramp commands.



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Figure 1-1. - Rudder/Speedbrake subsystem in avionics system.

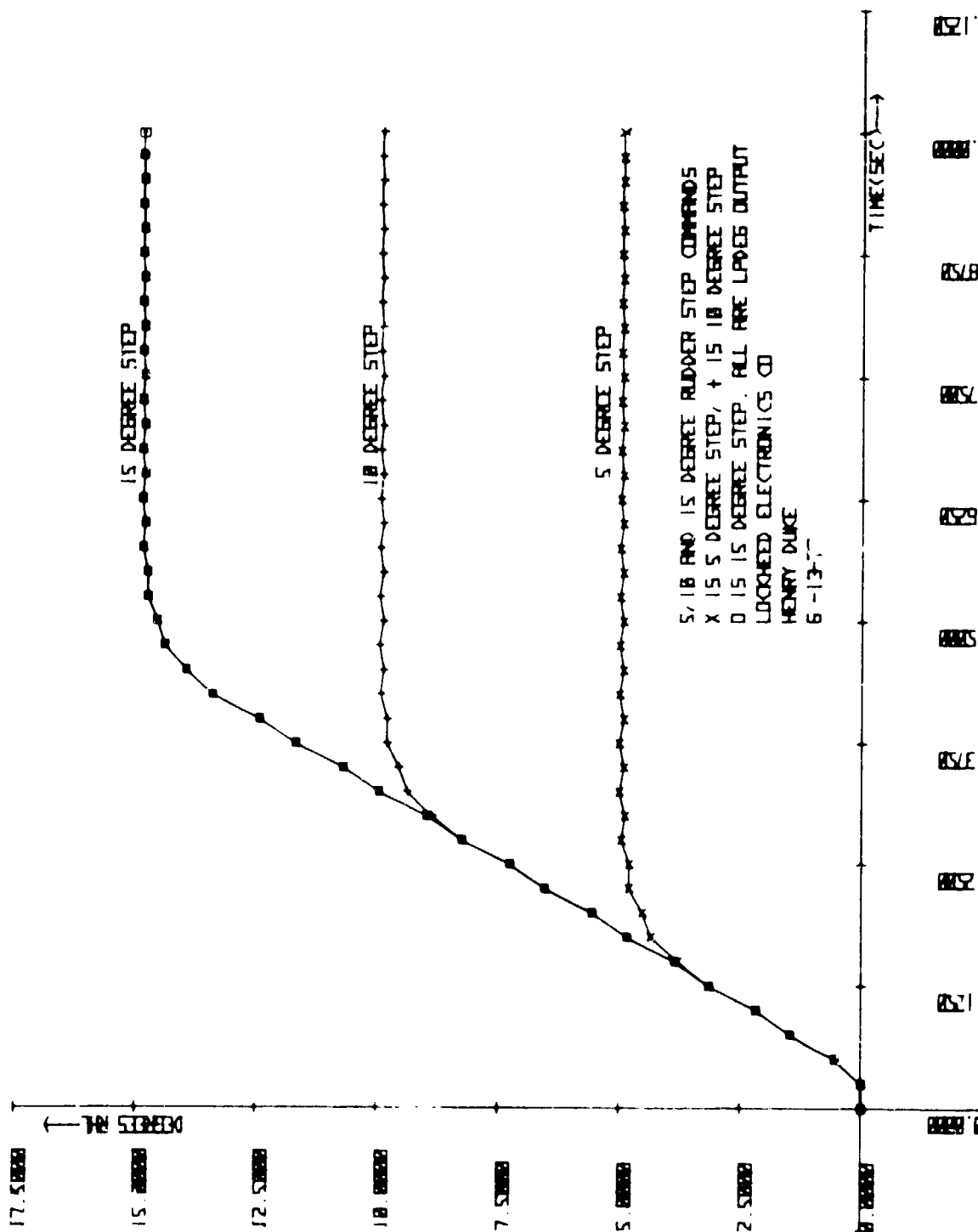


Figure 1-2. - Example of plot format.

2. INTRODUCTION

The R/SB subsystem is one of the major flight control systems developed for use as primary flight control in the Space Shuttle. This report presents a basic analysis of the R/SB using a 4-channel software model developed by LEC engineers from inputs by Rockwell. Rockwell developed a mathematical description of the R/SB (ref. 1) which was used in conjunction with CSMP III (ref. 2) to develop the software model used in this analysis. This report makes use of the CSMP model to investigate some basic system response characteristics such as linearity and response time.

3. CSMP PROGRAM

The CSMP program is an extremely versatile program with a variety of useful capabilities. It features a debug subroutine which, for a predetermined number of points, will list each of the variables and their value for each integration sampling time.

One of the major advantages of CSMP is its sort capability. The computer will sort the equations as necessary, place them in order and operate on them without the programmer being concerned with the order of occurrence of the equations.

Subroutines are also available to allow for the handling of 1st and 2nd order functions without the necessity of programming separate integration subroutines. Outputs can also be shown in logarithmic readout, standard readout, and other useful possibilities.

A listing of the CSMP program used is given in appendix A. The CSMP program is broken into five separate sections:

1. INPUT JCL - initial setup instructions to computer.
2. INITIAL - system constants and initial conditions.
3. DYNAMIC - system equations.
4. TERMINAL - integration method and time, start time, finish time, plot time, etc., necessary to produce outputs and load in outputs to be plotted or listed.
5. OUTPUT JCL - sort and perform necessary operations to produce output.

During the "reading in" of the data described in sections 2, 3 and 4 above, the basic CSMP program can be broken into with basic FORTRAN as a subroutine to perform decision-making.

There are some limitations, however, in the performance of the CSMP model. Some of the more serious limitations are:

1. Since the computer performs as a sampling data system, a limitation exists on the size of the sampling time for integration purposes. These limits depend upon the integration routine used and internal system oscillations encountered. Some recent experimentation with those limits resulted in the determination that the 50 μ s integration time used in the program shown in appendix A was in fact approaching maximum. Using a larger integration time resulted in numerical instability. The method used in the present CSMP integration process is the Runge-Kutta fixed step which, in this program, works quite well with a 50 μ s integration period.
2. The achievement of reasonable results in R/SB performance requires large amounts of CPU time. As an example, an average 1-second run requires almost 20 minutes CPU time.
3. There is a limitation on the total possible number of statements which may be used in the model. The present model utilizes over 580 statements which approaches the 600 statement limit. This borderline condition prohibits any possible large expansions to accomodate system changes. Elimination of portions of the program will be necessary to accomplish large changes which require additional cards.

Except for its few limitations, the CSMP program is a versatile tool in the analysis of control systems.

Appendix F contains a list of the variables and constants used in the CSMP program and their definitions or functional descriptions.

4. STEP RESPONSE

The plots obtained for 5-, 10- and 15-degree R/SB steps are given in appendix B. The first six (6) plots contained therein are the original CSMP computer printout plots whereas, the others were obtained by plotting the CSMP values using a HP 9820 calculator and a HP 9862A calculator plotter. For a further discussion of this technique, see appendix G.

Of major concern was the accuracy of both Rudder and Speedbrake deflections. Rudder input commands were based upon a 5-volt-input level causing a 27.1-degree Rudder Hinge Line (RHL) deflection of each panel in the same direction. Speedbrake input commands were based upon a 5-volt input command causing a panel separation angle of 49.3 degrees (RHL).

After panel movement settled down to an approximate steady-state value, a number of final points were averaged and this was used as the steady-state value of panel deflection (or separation). The resulting plots are shown in appendix B (B-7 through B-12). Table I gives the results of this analysis. An obvious discrepancy exists here since the +15 degree Speedbrake step has not reached its steady-state value. The 5- and 10-degree steps have, however, and the error achieved here will be used in this analysis. From this, it can be observed that the Rudder response exhibits a reasonable error, but the Speedbrake is excessive. Magnifying the 10-degree steps in both Rudder and Speedbrake steady-state graphs (B-9 and B-10) reveals that a 20 Hertz oscillation exists on the output waveshape. This is quite low level, however, with the peak-to-peak value being down by over 23 dB from the steady-state value of 9.9003 volts for the Rudder and 9.306 volts for the Speedbrake. The blowup reveals that the Speedbrake is still rising at a rate of 0.5 deg/sec. At this rate, it would take 20 seconds to arrive at

TABLE I.- STEP RESPONSE STEADY-STATE VALUES

Type*	Input Command Step (Deg)	Average	No. of Points	Error (%)
R	5	4.9495	50	1.01
R	10	9.9003	50	1.00
R	15	14.842	50	1.05
SB	5	4.555	50	8.9
SB	10	9.306	50	6.94
SB	15	No final ss** value achieved	50	Undetermined

*R is Rudder

SB is Speedbrake

**steady state

the 10-degree steady-state position. As shown in section 5, this increase will become steady-state in well under 20 seconds.

The delay time from input stimulus to output response is also of importance. The major contribution to the delay time comes from deadspace contained in the PDU gear train and in the Servo valve. Some other contributions exist in the summer and mixer gears, but, as shown in section 6, they are negligible. The delay times for the Rudder and Speedbrake step commands are given in table II with respective plots contained in appendix C.

Values shown in table II are based upon the first usable point given in the readout. The actual starting point will be occurring in the 5-ms period preceeding the first usable point as shown in figure 4-1. Linear interpolation could be used to obtain a more accurate figure but since the system response is not linear, the improvement in accuracy would, at best, be small. Therefore, work in this area is found in section 5 which contains a general discussion of delay for any command.

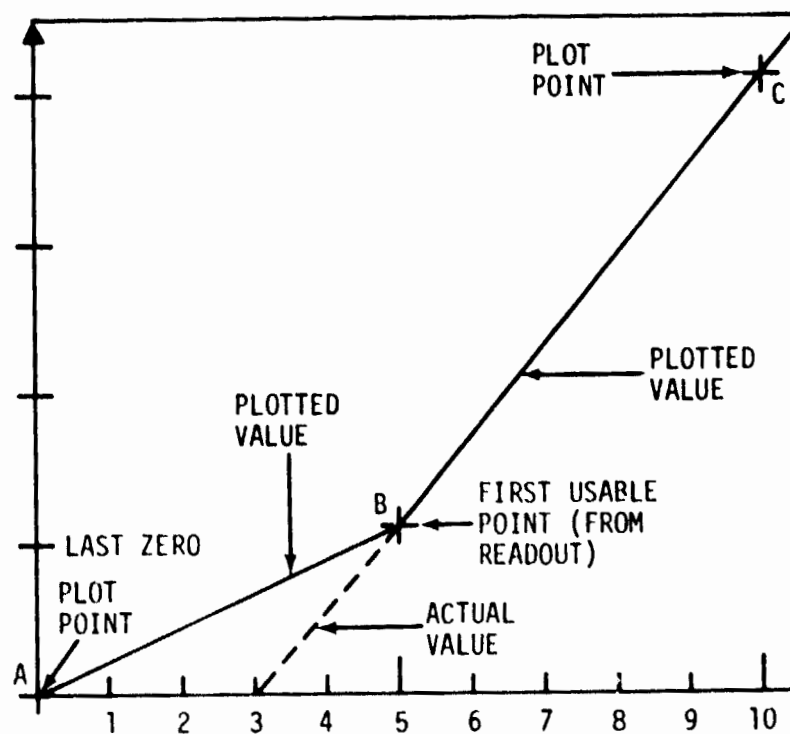
The slew rate of the response curves in figure 4-1 corresponds to the maximum possible system response to an input signal. This is the hardware limit of performance. The hardware limit must exceed the software limit to allow the system to respond to maximum possible system change commands.

A step command from the pilot will result in a ramp output from the ASA to the input of the Servo valves as shown in figure 4-2. (For a more detailed discussion of the input command, see appendix I).

At the present time, the Autopilot rate limits the MDM output to 12.1 deg/sec from the Rudder, 6.1 deg/sec (opening) and 10.85 deg/sec (closing) from the Speedbrake. The output panel

TABLE II.- DELAY TIME TO STEP COMMAND

Input	Delay Time		
	Start Command (Sec)	Start Response (Sec)	Delay (Sec)
RIN (Rudder)	0.005	0.025	0.020
SIN (Speedbrake)	0.005	0.040	0.035



4.1 - Graphical method of determining starting point of plot.

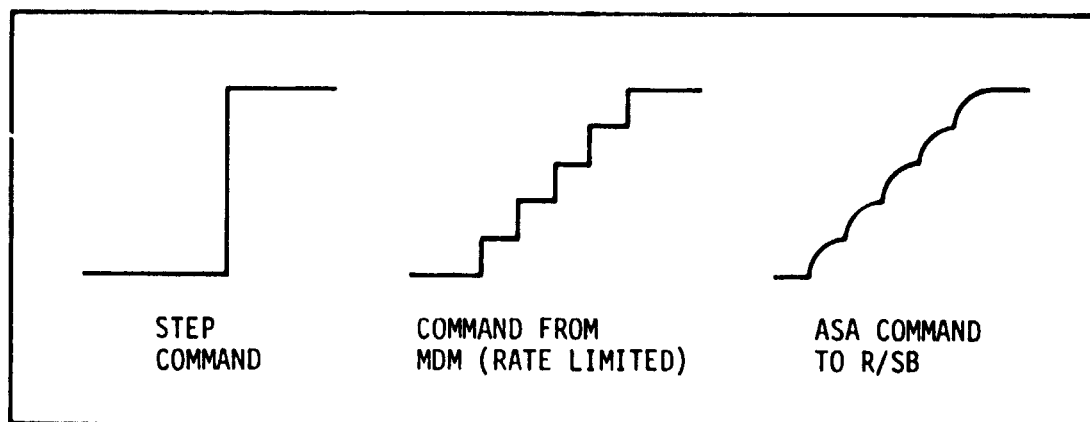


Figure 4.2 - Input command waveshapes.

speed for an input step must exceed these values for proper performance. The slew rates for Rudder and Speedbrake are given in table III. Since 5-, 10- and 15-degree step slew rates are the same for both Rudder and Speedbrake (see appendix B), only the values for a 10-degree step are listed.

The Rudder slew rate corresponds to the change in the left panel position (LPDEG) and the Speedbrake corresponds to the change in the angular separation of the Speedbrake (DSBRHL). Both values are RHL and fall well outside of the minimum opening panel deflection rates of 12.1 and 6.1 for the Rudder and Speedbrake.

TABLE III.— RUDDER AND SPEEDBRAKE MAXIMUM SLEW RATE

Command	Slew Rate (deg/sec)
Rudder 10-deg step	34.07
Speedbrake 10-deg step	21.20

5. RAMP RESPONSE

This section reviews the response of the R/SB systems to ramp input stimuli. Each of the four input channels in one of the systems (Rudder or Speedbrake) is addressed with a ramp while the four channels of the other system are addressed to the null (zero) position. All components are taken as operational and at a temperature of 100° F.

Figures 5-1, 5-2, 5-3 and 5-4 give the output (RIN1 vs LPDEG or SIN1 vs DSBRL) for Rudder and Speedbrake input commands. Figures 5-2 and 5-4 demonstrate that the R/SB outputs follow the inputs so close that an error is undetectable. To obtain meaningful results, it was necessary to observe Rudder performance for over 2 seconds and Speedbrake performance for over 6 seconds. As can be seen, the delay on these final segments is constant, being 78 ms for the Rudder and 450 ms for the Speedbrake.

Initially, at the beginning of response, the situation is quite different as shown in figures 5-5 and 5-6. The initial delay in the action of the panels is caused wholly by the hysteresis dead-space. This is discussed and shown in more detail in section 6. After initial response of the panels to the stimulus, the system exhibits an additional slowness in coming up to speed. After running speed is reached, the system overshoots and then, after a period of time, returns to following the input command. This is characteristic of the ramp response of a working hardware model where the motor speed comes up to rated speed then overshoots and dampens out to follow the input.

In May, some information was received from Rockwell in a letter concerning their model. Graphs were sent of the R/SB performance which included the hydraulic motor speed for a single motor with three (3) operating, output panel position and output panel rates for both Rudder and Speedbrake. The information received corresponded to a 10.03-deg/sec Rudder command and Speedbrake commands

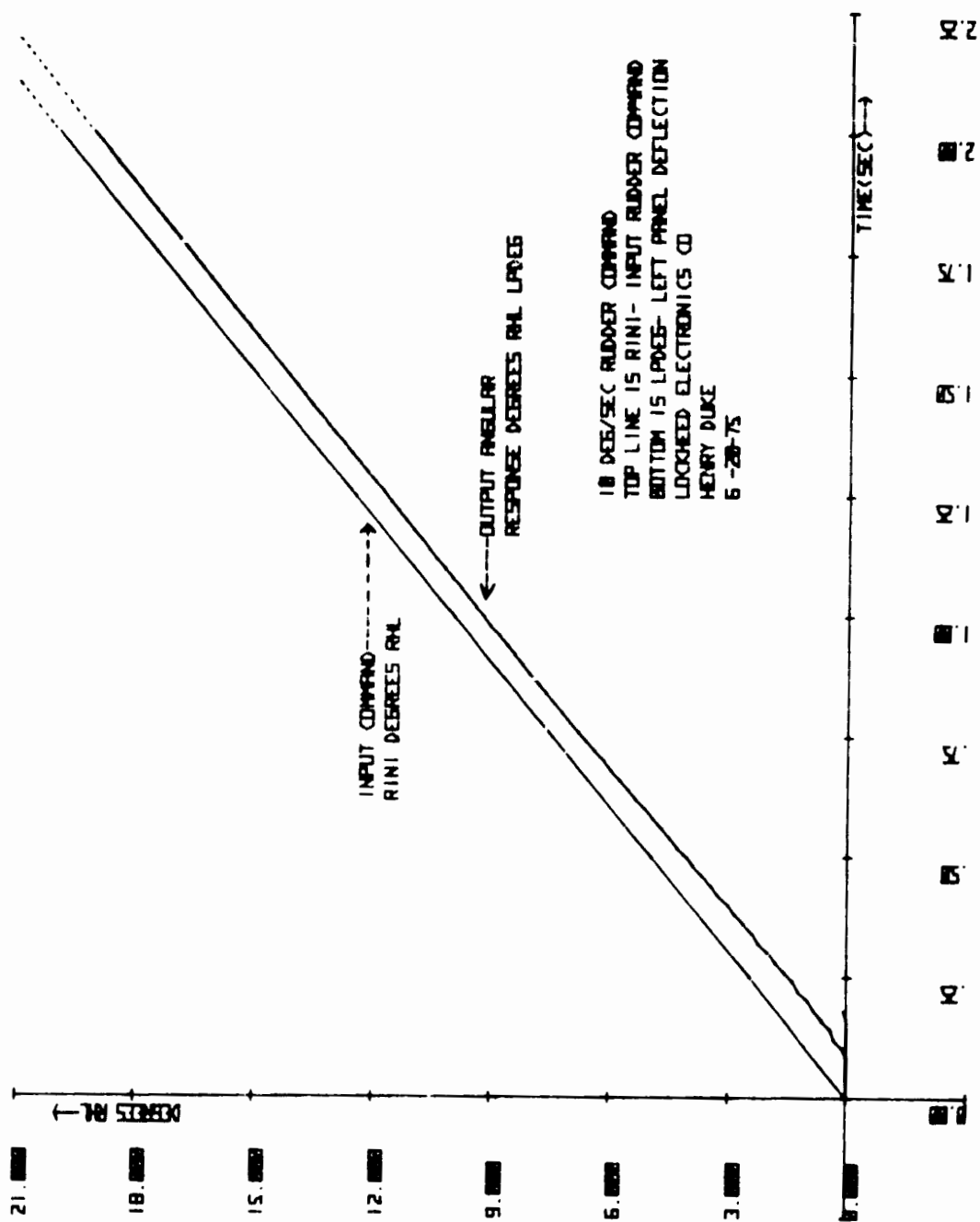


Figure 5-1. - Rudder input stimulus and output response.

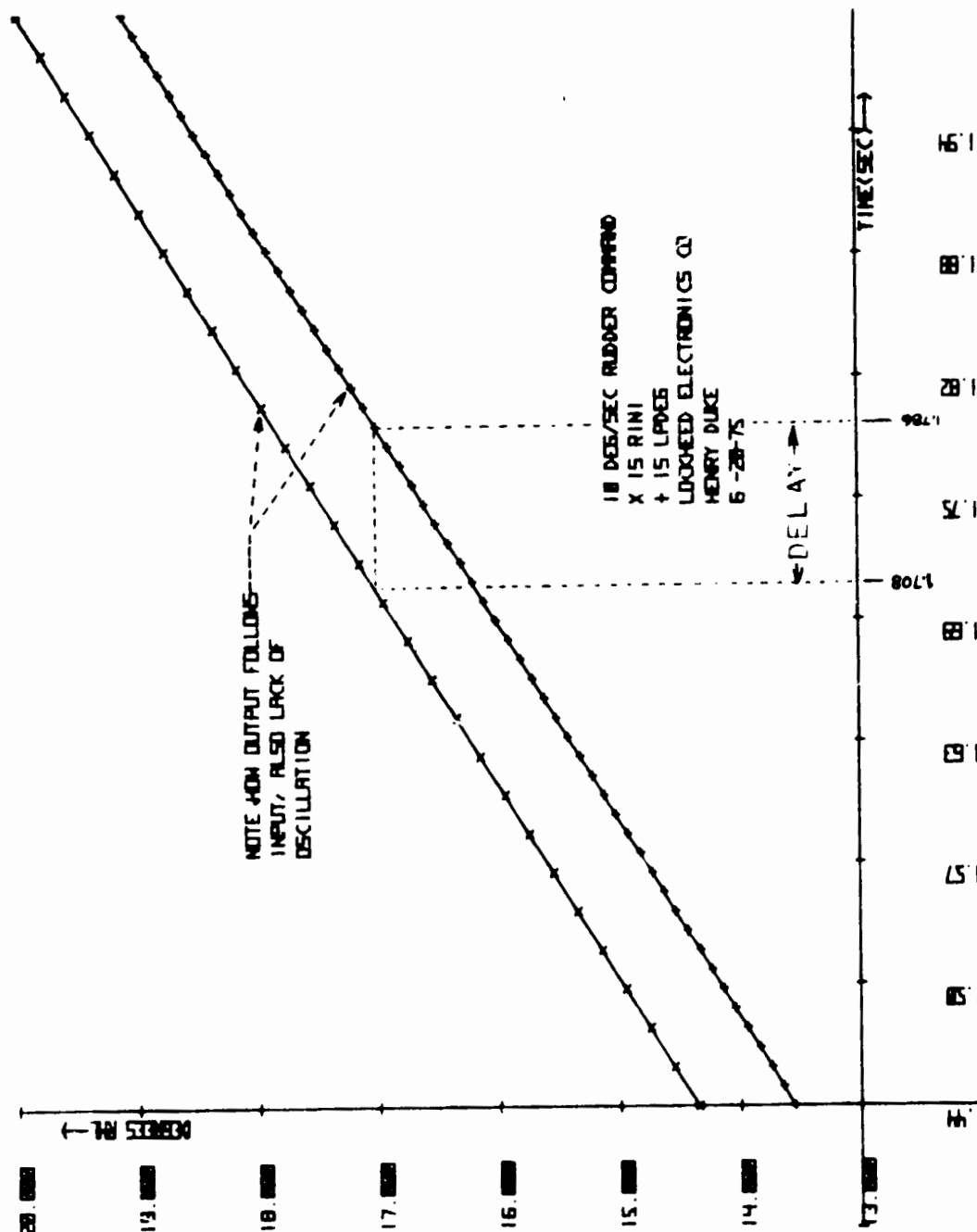


Figure 5-2. - Rudder input stimulus and output response magnified.

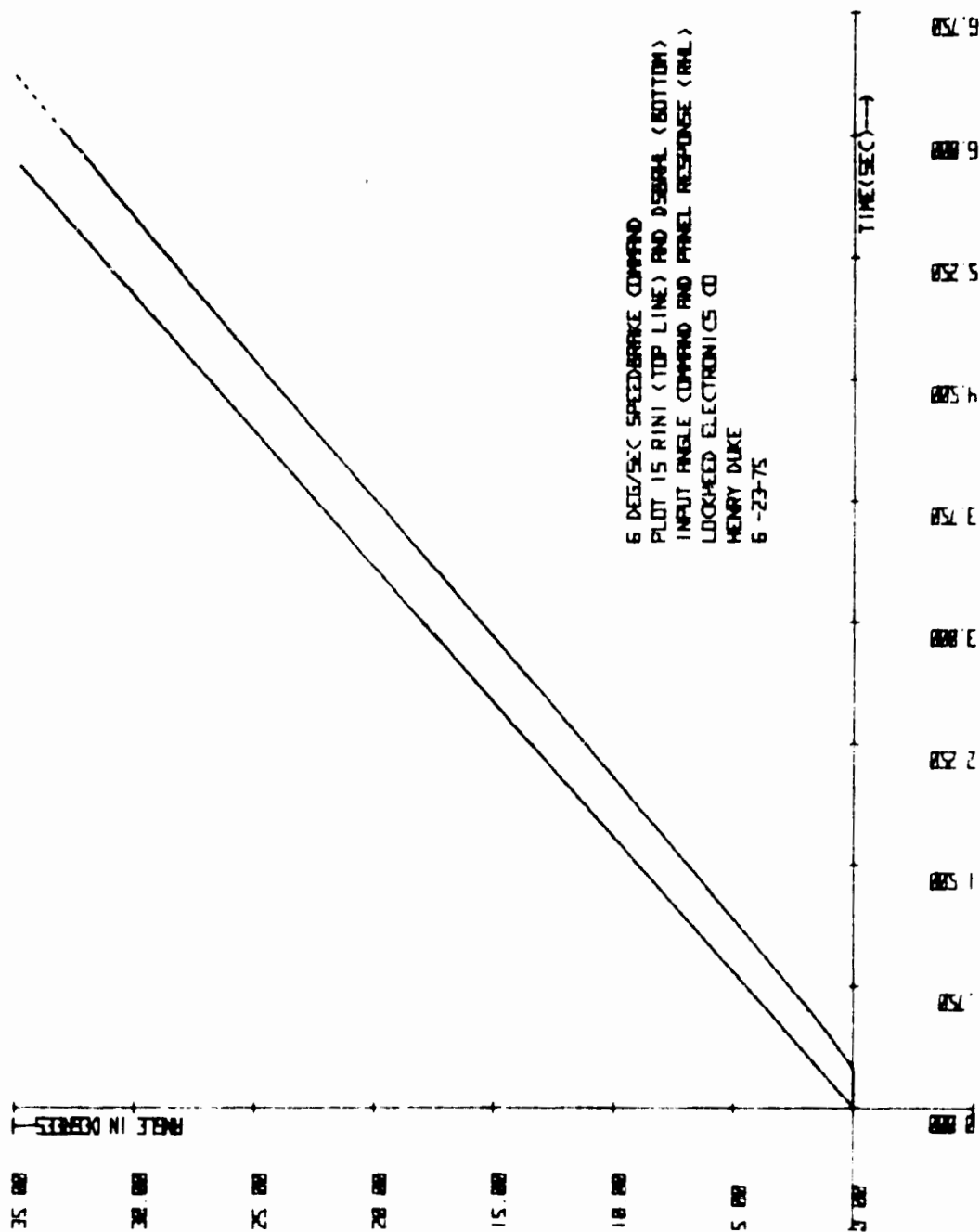


Figure 5-3. - Speedbrake input stimulus and output response.

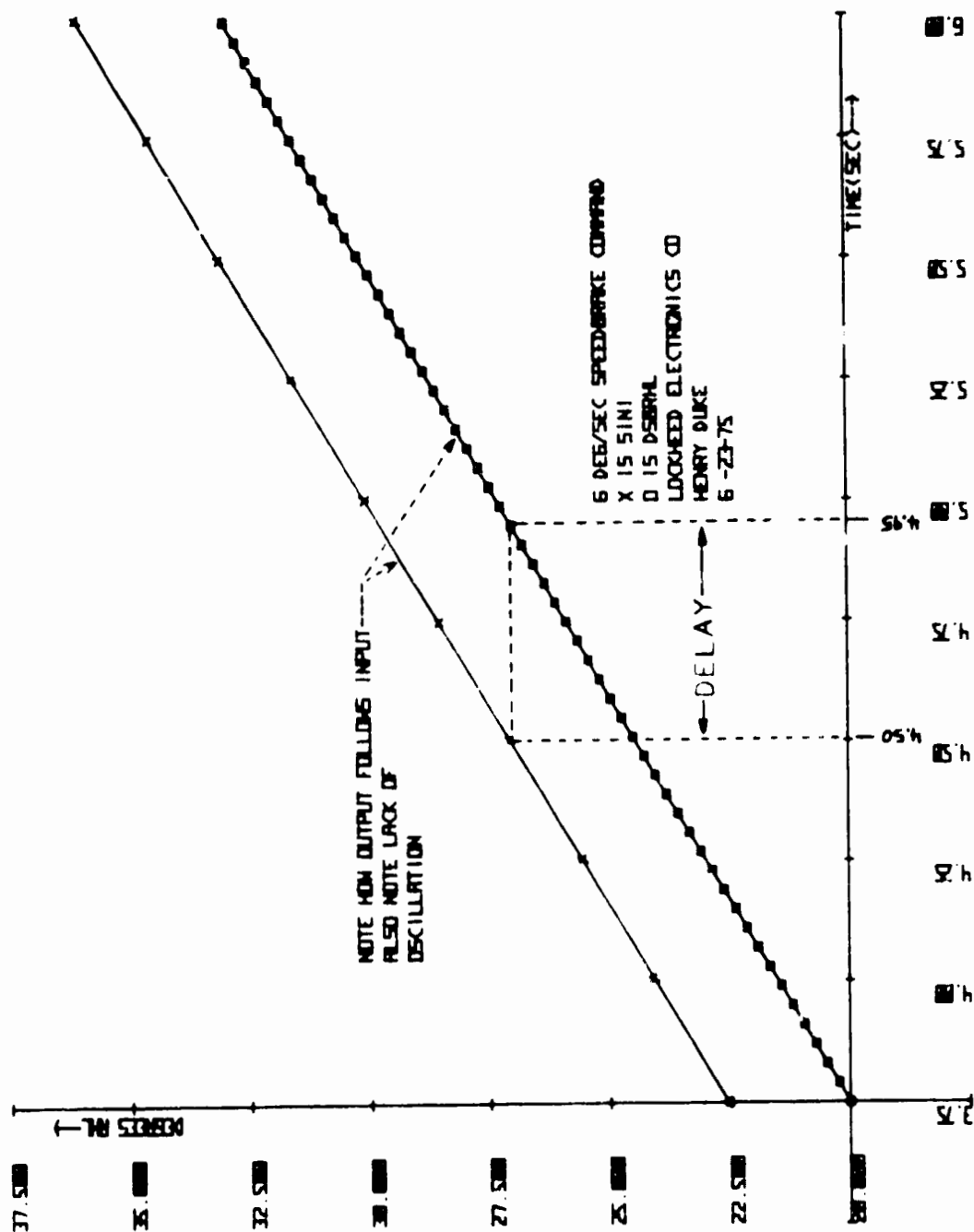


Figure 5-4. - Speedbrake input stimulus and output response magnified.

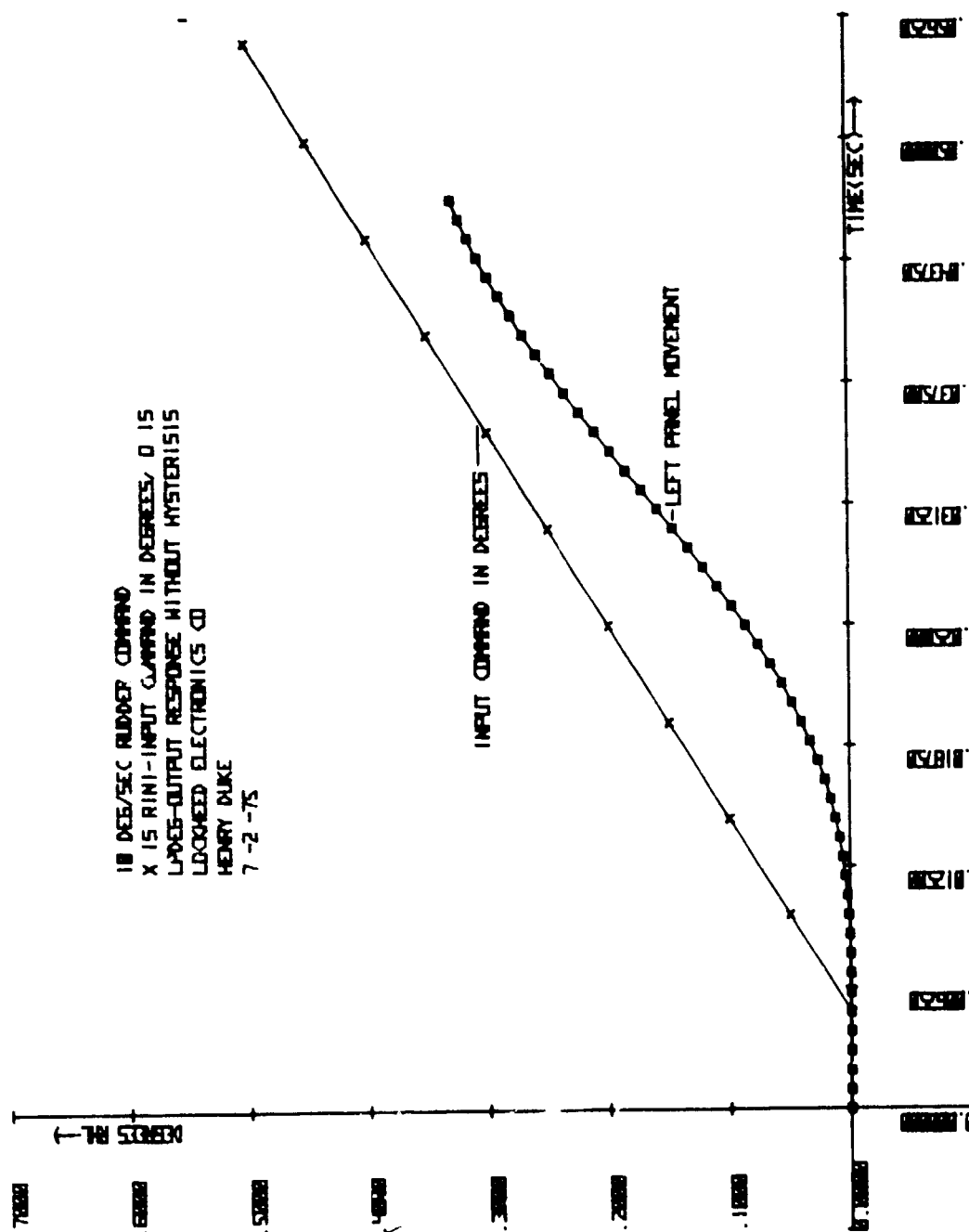


Figure 5-5. - Rudder initial response.

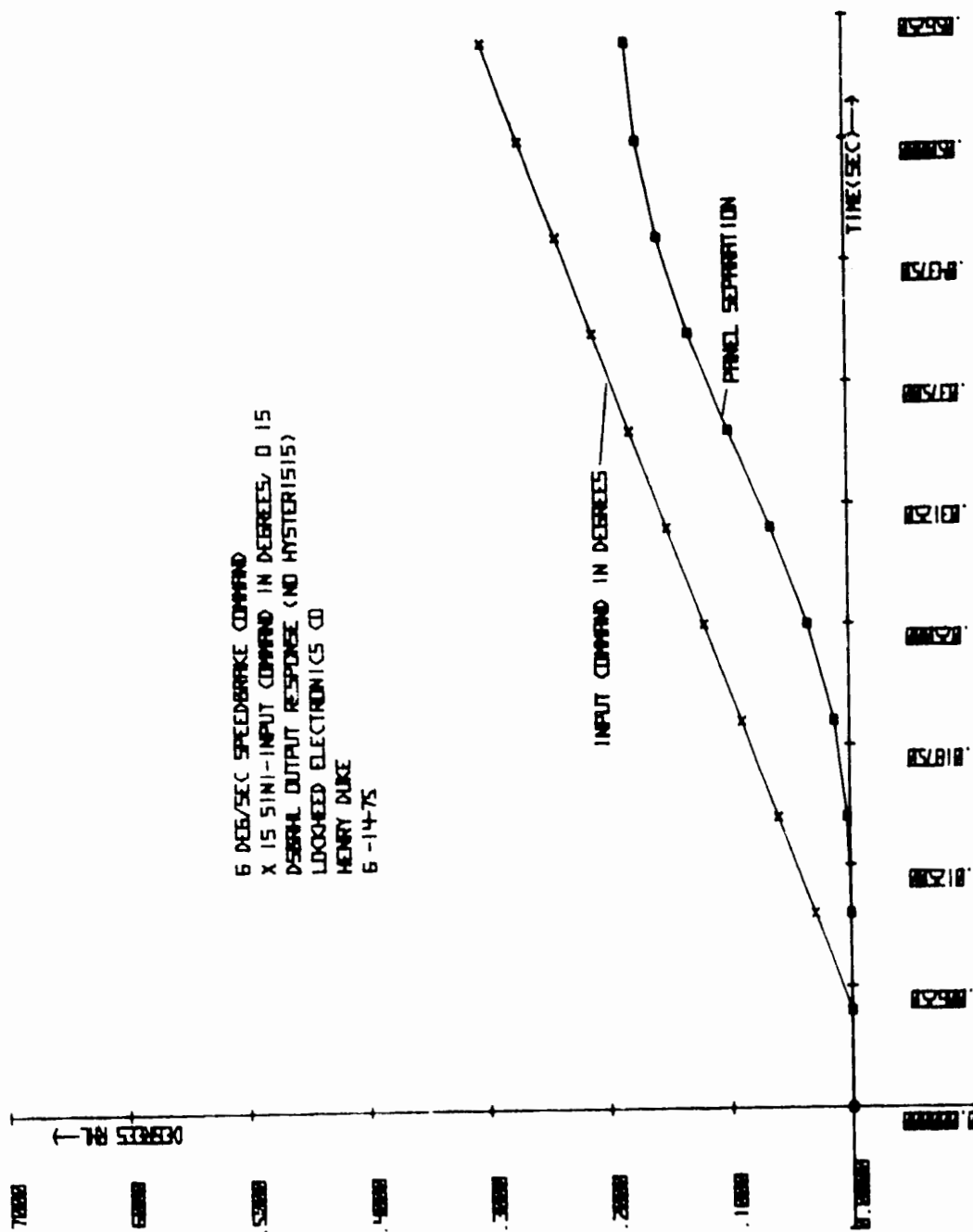


Figure 5-6. -- Speedbrake initial response.

respectively. Graphs contained within this letter are contained in appendix D for reference. Also contained in appendix D are corresponding CSMP plots. Prior to receipt of this data, a telephone conversation with personnel at Rockwell revealed that for 3-motor operation, the Rudder motor speed-per-motor should be 2850 rpm and the Speedbrake motor speed for a single motor should be 2550 rpm. The plots contained in appendix D show the results found in table IV.

The Rockwell model had given the motor speed for 10.03 deg/sec. This has been interpolated to 6 deg/sec which is the value given:

$$\begin{aligned}\frac{10 \text{ deg/sec}}{6 \text{ deg/sec}} &= \frac{450}{X} \text{ rad/sec} \\ X &= 270 \frac{\text{rad}}{\text{sec}} \times \frac{60 \text{ sec}}{1 \text{ min}} \times \frac{1 \text{ rev}}{2 \pi \text{ rad}} \\ &= 2578 \text{ rpm}\end{aligned}$$

Speed for the Lockheed values were taken as an average over a number of points, as shown in figures D-3 and D-8, using methods in appendix G. Note that the Rudder error in Lockheed's model is very small whereas the Speedbrake error is over 2 percent. It has also been determined that the stated motor speeds are well within the capabilities of the motors as their capability is dependent mainly upon the maximum possible flow. Appendix E features a writeup of the necessary fluid input to produce a given output rotational speed. Maximum output speed for 3-motor operation is that for the 12.1-deg/sec Speedbrake closing. Using interpolation:

$$\text{Speed} = \frac{(12.1)(2850) \text{ rpm}}{10} = 3449 \text{ rpm}$$

For a speed of 3449 rpm, the motor input will have to be about 7.8 gallons per minute (GPM), well within the 22.3-GPM maximum (ref. 1 para. 3.1.2.4.4).

Since the 12.1-deg/sec input command is the software limit, it represents worst case and it is easily seen that both Rudder and Speedbrake hydraulic sources will be capable of producing the necessary input flow.

TABLE IV.— COMPARISON OF MOTOR SPEEDS

Type Of Input	Rate (deg/sec)	Motor Speed		
		Rockwell telephone (RPM)	Rockwell model (RPM)	Lockheed (RPM)
Rudder	10	2850	2836	2848.8
Speedbrake	6	2550	2578	2611.2

6. DEADSPACE

For this analysis, the deadspace¹, or "deadband" as it is sometimes called, is the time elapsed from the input stimulus to one or more hysteresis loops to the time when the output responds. Observed in this section are the effects of each individual loop upon deadspace and the total cumulative effects of all loops.

Appendix H plots give the deadspace measured in all four hysteresis loops for both Rudder and Speedbrake performance for a 10-deg/sec Rudder and a 6-deg/sec Speedbrake command. This section, however, deals with their effects for various commands and how to calculate the deadspace for any input command (see figure 6-1).

There are fourteen (14) hysteresis loops located in the R/SB subsystem. Four of these are common to both Rudder and Speedbrake command responses, whereas the remaining ten are peculiar to either Rudder or Speedbrake command chains. Table V lists the location of each hysteresis loop, the value in arc-min, and approximate time contribution for a 10-deg/sec or 6-deg/sec Rudder or Speedbrake command. Note that a Rudder 10-deg/sec command will not be felt equally at the beginning of the first loop as a Speedbrake 10-deg/sec command. This is caused by a variation of input factors called KRV and KSBV. KRV is shown in figure 6.2 with KSBV in a similar position for the Speedbrake. For a further discussion of the differences, refer to appendix I.

A series of various input Rudder commands were input and the cumulative effects of hysteresis deadspace were plotted as shown in figure 6-1. Each plot point is the cumulative time from input stimulus to the first observed output response.

As observed, the largest contributor to deadspace is the PDU gear train. Although the value for the PDU gear train hysteresis is

¹Deadspace pertains to the special DEADSPACE function utilized in CSMP programming (see reference 2).

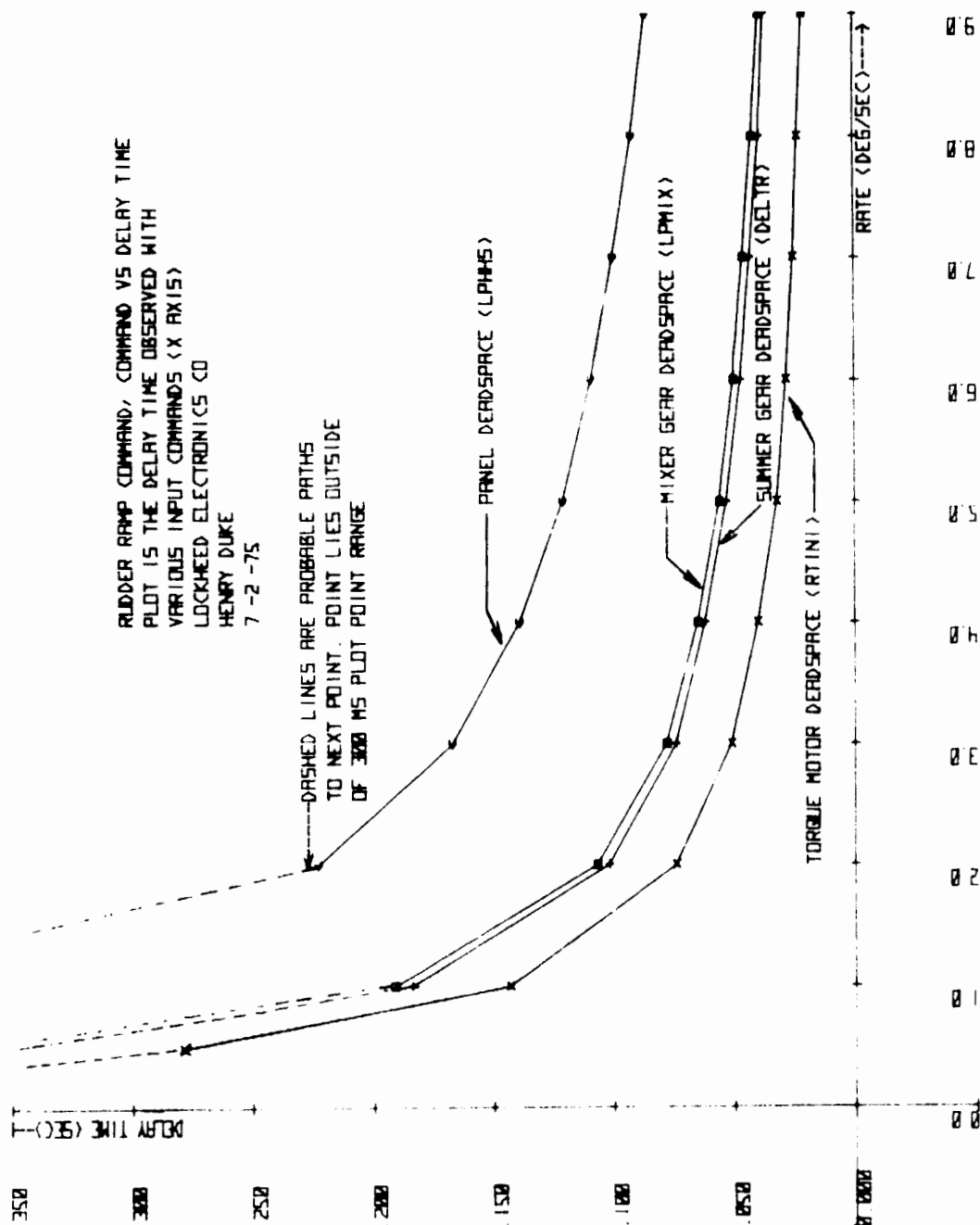
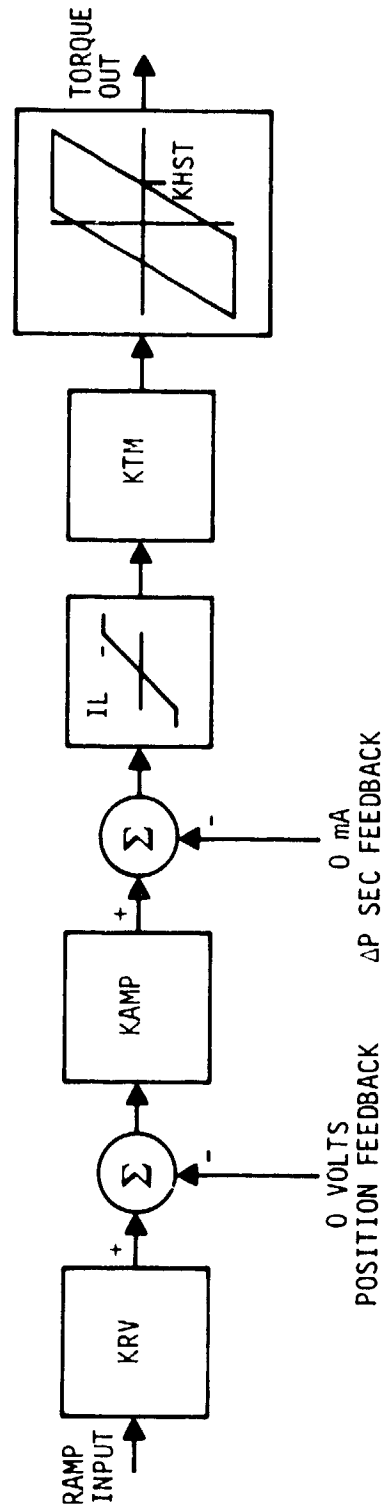


Figure 6-1. - Cumulative Rudder deadspace for various input commands.



KRV = Rudder conversion factor = $0.1845 \frac{\text{Volts}}{\text{Degree}}$

KAMP = ASA gain = $17.5 \frac{\text{mA}}{\text{Volt}}$

IL = Flapper valve torque motor = $\pm 8 \text{ mA}$
current limit

KTM = Current to torque gain = $0.045 \frac{\text{In-lb}}{\text{mA}}$

KHST = Flapper torque hysteresis = 0.02 In-lb

Figure 6-2. - R/SB subsystem from input to flapper output.

TABLE V.— HYSTERESIS CONTRIBUTION TO DEADSPACE

Hysteresis Loop	Description	Value (arc-min)	Time For Command (sec) (deg/sec)	
1R	Flapper torque motor	0.4	0.014	10 R
2R	Flapper torque motor	0.4	0.014	10 R
3R	Flapper torque motor	0.4	0.014	10 R
4R	Flapper torque motor	0.4	0.014	10 R
5R	Summer gears	163.8	0.015	10 R
1SB	Flapper torque motor	0.4	0.036	6 S
2SB	Flapper torque motor	0.4	0.036	6 S
3SB	Flapper torque motor	0.4	0.036	6 S
4SB	Flapper torque motor	0.4	0.036	6 S
5SB	Summer gears	163.8	0.020	6 S
*1R/SB	Mixer gears left panel	71.8	0.003	10 R
*2R/SB	Mixer gears right panel	71.8	0.003	10 R
*3R/SB	PDV gears left panel	10.0	0.043	10 R
*4R/SB	PDV gears right panel	10.0	0.043	10 R

*Time shown is for 10-deg/sec Rudder command only.

apparently much less than some of the other values shown in table V, its deadspace is disproportionately greater. This is caused by the large gear ratio that exists from the motor outputs to the panel. This gear ratio is 5,099:1 for the Rudder and 15,472:1 for the Speedbrake.

From figure 6-1, it can be ascertained that the output delay is approximately 4 times the input delay. This means that the 0.074 sec delay for the 2 deg/sec run will take 0.29 seconds to be felt on the panels.

By knowing the delay involved in the first loop, the remaining hysteresis deadspace can be found easily by approximation. This factor of 4 applies to the Rudder only. The Speedbrake was not run for various inputs as was the Rudder, but from the 6 deg/sec command, it is easily found that the factor is 11.

1. Rudder deadspace output panel movement is:

$$\text{Deadspace} \approx \frac{(4)(0.145)}{\text{slope of input}} = \frac{0.58}{\text{slope}}$$

i.e., 10 deg/sec \approx 0.058 sec deadspace

2. Speedbrake deadspace output panel movement is:

$$\text{Deadspace} \approx \frac{(11)(0.08)}{\text{slope of input}} = \frac{0.88}{\text{slope}}$$

i.e., 6 deg/sec \approx 0.146 sec deadspace.

A block diagram of the first hysteresis loop is shown in figure 6-2. This shows one channel (of a 4-channel servo valve) from the input command to the flapper torque output.

Figures 6-3 and 6-4 show the times for various input commands to reach threshold. These are further listed in tables VI and VII.

A single 10-deg/sec run was made without the hysteresis loops present to determine if there were other unknown factors contributing to the deadspace. This is given in figure 6-5. Although figure

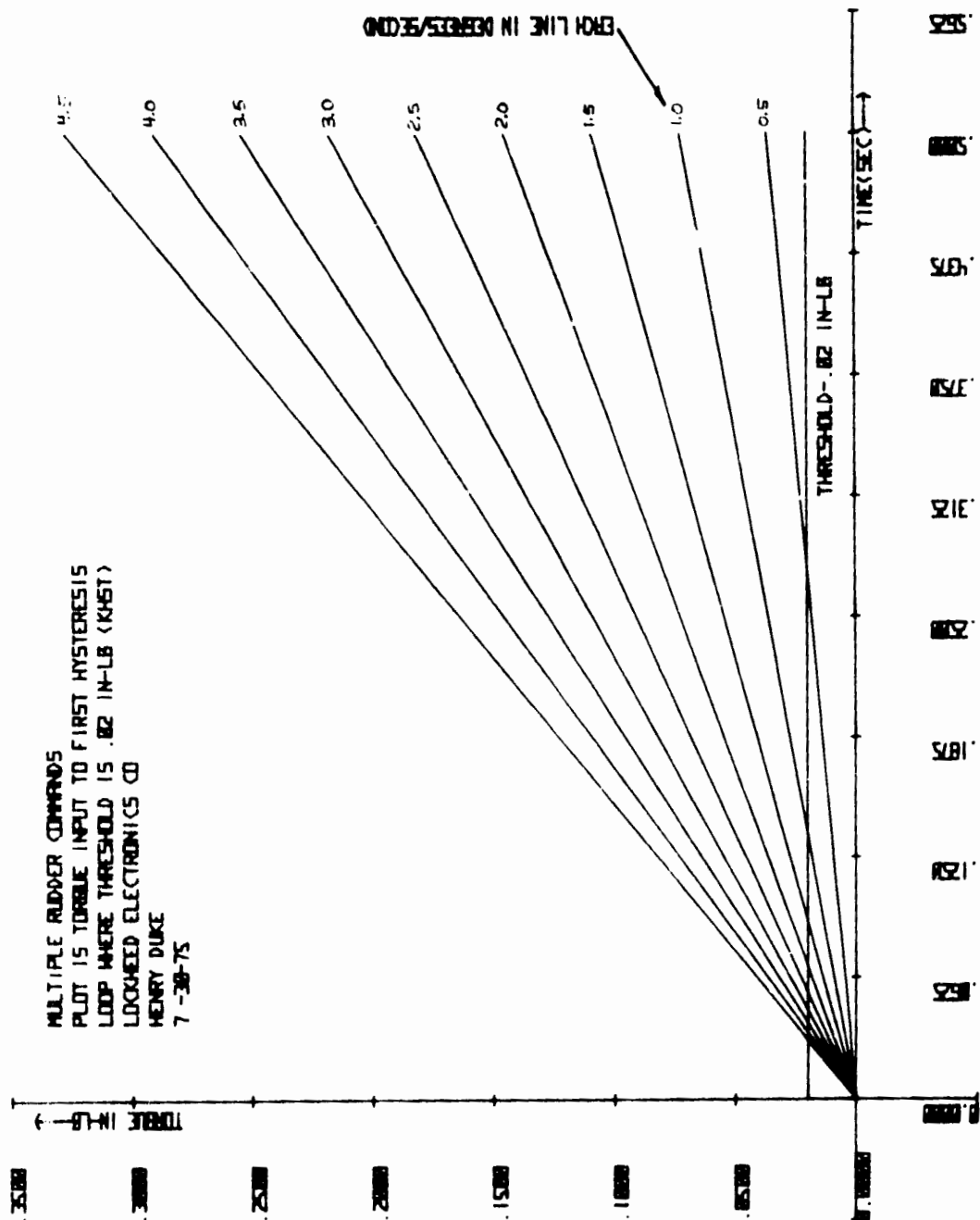


Figure 6-3. - Various input ramps seen at first hysteresis loop for Rudder.

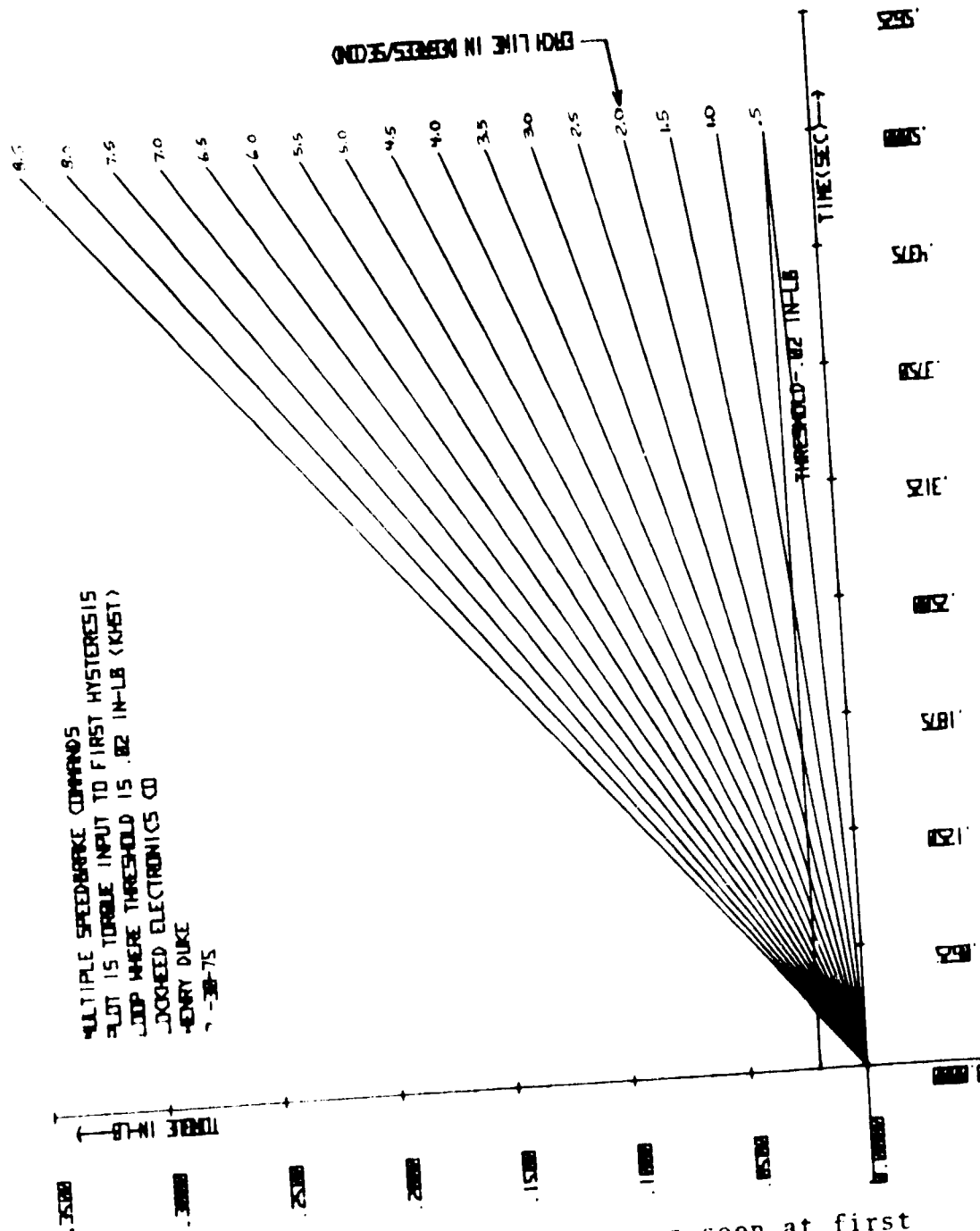


Figure 6-4. Various input ramps seen at first hysteresis loop for Speedbrake.

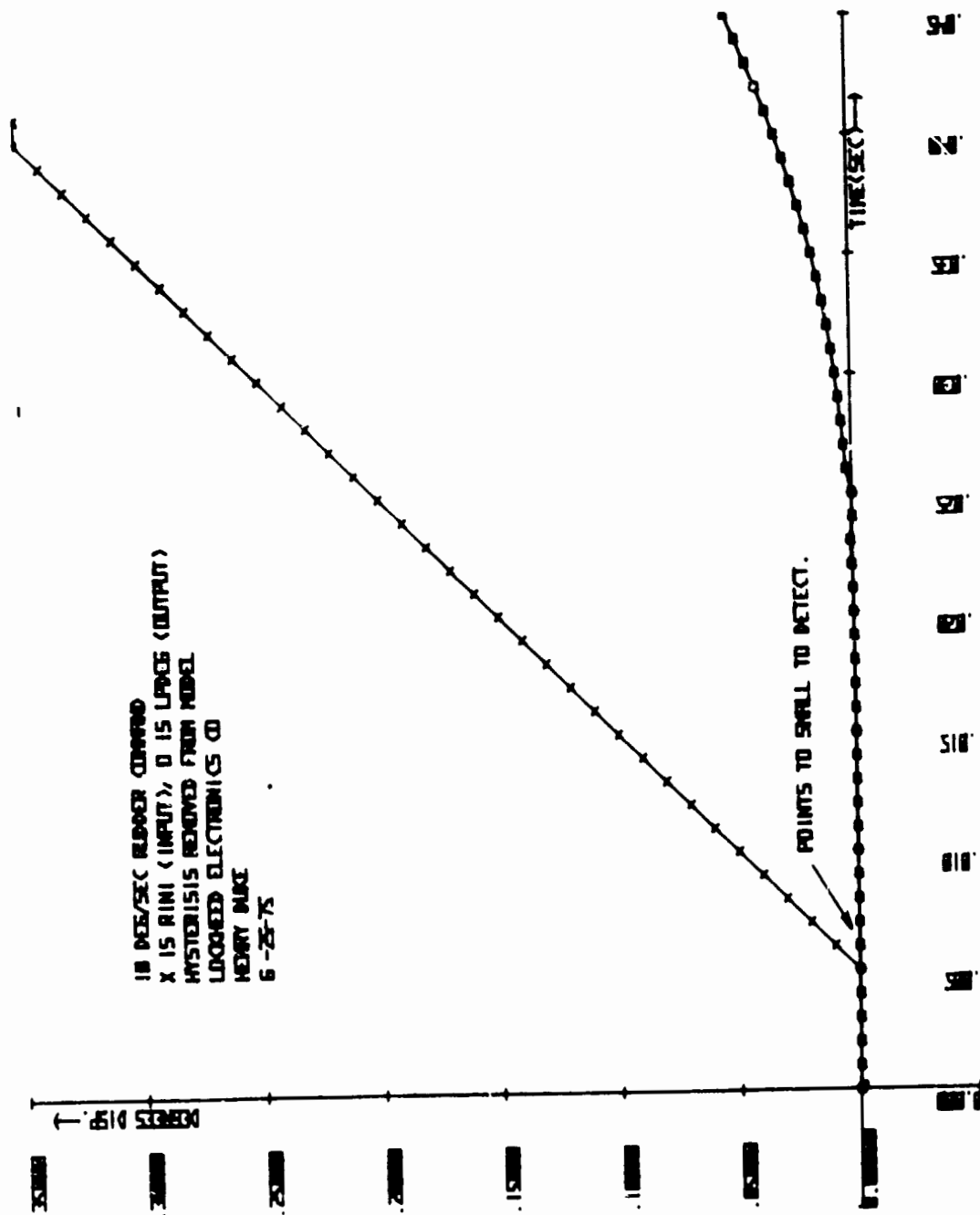


Figure 6-5. - Input stimulus and output response without hysteresis.

TABLE VI.— RUDDER DEADSPACE FOR SERVO FLAPPER VALVE

Ramp Input Command (deg/sec)	Start Time for Output of Flapper (sec)	Delay in Action through Servo Flapper (sec)
0.1	1.3815	1.3765
0.2	0.6932	0.6882
0.3	0.4638	0.4588
0.5	0.2803	0.2753
1.0	0.1427	0.1377
2.0	0.0738	0.0688
3.0	0.0509	0.0459
4.0	0.0394	0.0344
5.0	0.0325	0.0275
6.0	0.0279	0.0229
7.0	0.0247	0.1967
8.0	0.0222	0.0172
9.0	0.0202	0.0152
10.0	0.0187	0.0137
12.1	0.0164	0.0114

TABLE VII.- SPEEDBRAKE DEADSPACE FOR SERVO FLAPPER VALVE

Ramp Input	Delay in Servo Performance (Sec)	Actual Delay
0.1	2.5099	2.5049
0.2	1.2570	1.2520
0.3	0.8397	0.9835
0.5	0.5058	0.5008
0.7	0.3627	0.3577
1.0	0.2554	0.2504
2.0	0.1302	0.1252
3.0	0.0885	0.0835
4.0	0.0676	0.0626
5.0	0.0551	0.0501
6.1	0.0461	0.0411
8.0	0.0363	0.0313
10.84	0.0281	0.0231
12.0	0.0259	0.0209

6-5 does not fully demonstrate this, the CSMP readout began listing points for the output LPDEG at $t = 5$ ms, the start time for the input.

Some discrepancies do exist in the application of figures 6-3 and 6-4 beyond the 0.02 in-lb threshold. At this point, the secondary delta P feedback is no longer zero and therefore, begins to subtract from the input. The graph will therefore begin to change slope as the error begins to go to zero.

The values given in tables VI and VII were not graphically obtained but were mathematically calculated using the HP 9820. Consequently, they are more accurate than any values obtained from figures 6-3 and 6-4.

It must, therefore, be concluded from the information given that the hysteresis, and only the hysteresis, is the sole contributor to the deadspace.

7. CONCLUSIONS AND RECOMMENDATIONS

The contents of this report were concerned with the analysis of the Space Shuttle R/SB subsystem using the CSMP program to test the Rockwell math model contained in Rockwell publication SD 74-SH-0324.

It was first shown how the program was implemented using the CSMP technique and some performance limitations were given in using CSMP. A brief overlay of the CSMP program technique was made with a sample program and glossary appearing in the appendix.

The report discussed three (3) separate areas of system performance: 1) the Step Response, 2) the Ramp Response, and 3) the delay time obtained in system response.

The Step Response was used to determine three (3) factors:

1) the linearity of the system, 2) the accuracy of the system, and 3) the speed of the system in responding to a step command. A 5-, 10- and 15-degree step command was separately addressed to first the Rudder and then to the Speedbrake. When one channel was exercised, the other was set to zero degree step input or null.

It was found that the Rudder displayed a 1-percent accurate output response for all three commands, whereas the Speedbrake response was in excess of 6 percent. The cause of this radical variation in accuracy was undetermined. It possibly exists in a definition of either the expected panel response to a 5-volt command not being 49.3 degrees RHL or in some factor in the summing and mixing gear trains. Further investigation will be necessary in this area to pinpoint the possible cause of trouble. The linearity of the Rudder was acceptable but the Speedbrake once again remained unacceptable. This was, however, probably caused by the run being too short to produce a true steady-state output response condition.

The 5-degree run was successful, but the 10-degree run may require more time. Since both Rudder and Speedbrake channels are identical except for constants, this appears to be the only reasonable solution. It was also found that both the Rudder and Speedbrake would be hardware capable of meeting maximum software input commands.

It was found that a low level oscillation exists in step command response. But it was such a low level that it is thought to be insignificant for further study. Further testing will require an observation of the oscillation but, unless it becomes significant, it will probably be neglected in future reports.

The ramp response was used to determine two factors: 1) the tracking ability and 2) the ability of the hydraulic system to meet specifications. The system was found to track very close to input command, in fact, with an undetectable error in both Rudder and Speedbrake performance. Oscillation was found to be virtually nonexistent and undetectable in output response. The speed of the hydraulic motors was compared with Rockwell-furnished data and found to be extremely accurate for the Rudder and within 3 percent for the Speedbrake. The hydraulic flow into the motors was compared with the maximum possible flow from the hydraulic pumps and it was found that the motors would not overstress the system for normal operation. Abnormal operating characteristics, such as the failure of one or two motors, will be the subject of a Failure Modes and Effects Analysis to be commenced soon.

The final section of the report dealt with one of the nonlinearities of the system; namely, the results of the system hysteresis. It was found that the system response was delayed from the input by a large period of time. An investigation was launched into the causes of this phenomena and the severity of its occurrence. It was found that the cause was solely the hysteresis loops contained within the model. The severity of the phenomena remains unknown since there appears to be no gauge as to its magnitude. The

Rudder was found to exhibit a delay time (or deadspace as it is called in this report) of 75 milliseconds whereas the Speedbrake was in excess of 200 milliseconds. The Rudder input rate was 10 deg/sec whereas the Speedbrake was 6 deg/sec. It was further found that, because the hysteresis loops created a threshold which had to be surpassed, the deadspace was command-dependent. An approximate mathematical technique has been developed in this report to determine the deadspace for all possible input commands.

This report leaves several unanswered questions. The most prominent of these is the large steady-state error in Speedbrake performance. Since the Speedbrake involves the use of both panels separating in opposite directions, it is conceivable that the observed error could be twice that of the Rudder. This does not, however, appear to be the case since the error is in excess of 6 percent. This issue needs to be pursued in greater depth.

Another item of interest is the large deadspace observed in both Rudder and Speedbrake output responses. This, as was observed, was solely the contribution of the hysteresis contained in the system and not some other ambiguous factor. At present, there appears to be no specification to cover this problem.

Future runs need to focus on these two items in particular, especially linearity. Critical items such as changing ASA gain and panel load factors will continue to create new runs to determine response to design changes and/or modifications of parameters. A failure Modes and Effects Analysis is also scheduled. During this study, the output response caused by such failures as failing a single motor will be observed.

The results herein will be used as guidelines, when possible, in the design of a R/SB subsystem hardware model. However, since the system is very fluid at the present time, the information contained herein will probably change with the hardware model following the latest designs.

8. REFERENCES

1. Anon.: Descriptions and Mathematical Models for Aerosurface Actuators. Rockwell International, Space Division, SD 74-SH-0324, December 1974.
2. Anon.: Continuous System Modeling Program III (CSMP III). IBM, SH 19-7001-2, September 1972.
3. Anon.: Procurement Specification, Actuator Subsystem, Rudder/Speedbrake. Rockwell International, Space Division; MC621-0053, February 1975.
4. Merrit, Herbert E.: Hydraulic Control Systems. John Wiley & Sons, Inc., 1967.
5. Gibson, J. E., et.al.: A Set of Standard Specifications for Linear Automatic Control Systems. AIEE Transactions, May 1961.

APPENDIX A

SAMPLE PRINTOUT
OF
CSMP

```

*****
//H01NASA JOB (1234), 'H DUKE', TIME=40, MSGLEVEL=(0,0)
//CSMPT PROC DSYS=2314, LIBRARY='CSMP.LIBRARY', SYSTEM=
//      PRELOAD='CSMP.LOAD', STEPLIB=SYSTEM, VSYS=SYSAUX
//T EXEC PGM=DEJCSMP5, REGION=102K, TIME=10
//CSMPLIB DD UNIT=SDSYS, VOL=SER=EVSY, DISP=SHR, DSN=GLIBRARY
//FT01F001 DD UNIT=SYSDA, SPACE=(3500, (20, 20)), DSN=&&SYSIN,
//      DCB=(RECFM=F, LRECL=80, BLKSIZE=800)
//FT02F001 DD SYSOUT=B, DCB=(RECFM=F, BLKSIZE=80)
//FT05F001 DD UNIT=SYSDA, SPACE=(3500, (10, 10)),
//      DCB=(RECFM=F, BLKSIZE=80), DISP=(NEW, PASS)
//FT07F001 DD UNIT=SYSDA, SPACE=(3500, (20, 20)),
//      DCB=(RECFM=F, BLKSIZE=80), DISP=(NEW, PASS)
//FT13F001 DD UNIT=SYSDA, SPACE=(3500, (40, 40)),
//      DCB=(RECFM=VS, BLKSIZE=444)
//FT14F001 DD UNIT=SYSDA, SPACE=(3500, (10, 10)), DCB=(RECFM=F, BLKSIZE=80)
//STEPLIB DD UNIT=SDSYS, VOL=SER=EVSY, DISP=(SHR, PASS),
//      DSN=CPRELOAD&STEPLIB
//FT03F001 DD SYSOUT=A
//FT08F001 DD SYSOUT=A, DCB=(RECFM=FBA, LRECL=133, BLKSIZE=1330)
//FT06F001 DD SYSOUT=A, DCB=(RECFM=FBA, LRECL=133, BLKSIZE=1330)
//      PEND CSMPT
//TRANS EXEC CSMPT
++CSMPT PROC DSYS=2314, LIBRARY='CSMP.LIBRARY', SYSTEM=
++      PRELOAD='CSMP.LOAD', STEPLIB=SYSTEM, VSYS=SYSAUX
++T EXEC PGM=DEJCSMP5, REGION=102K, TIME=10
++CSMPLIB DD UNIT=SDSYS, VOL=SER=EVSY, DISP=SHR, DSN=GLIBRARY
IEF6531 SUBSTITUTION JCL - UNIT=2314, VOL=SER=SYSAUX, DISP=SHR, DSN=CSMP.LIBRARY.SYSTEM
++FT01F001 DD UNIT=SYSDA, SPACE=(3500, (20, 20)), DSN=&&SYSIN,
++      DCB=(RECFM=F, LRECL=80, BLKSIZE=800)
++FT02F001 DD SYSOUT=B, DCB=(RECFM=F, BLKSIZE=80)
++FT05F001 DD UNIT=SYSDA, SPACE=(3500, (10, 10)),
++      DCB=(RECFM=F, BLKSIZE=80), DISP=(NEW, PASS)
++FT07F001 DD UNIT=SYSDA, SPACE=(3500, (20, 20)),
++      DCB=(RECFM=F, BLKSIZE=80), DISP=(NEW, PASS)
++FT13F001 DD UNIT=SYSDA, SPACE=(3500, (40, 40)),
++      DCB=(RECFM=VS, BLKSIZE=444)
++FT14F001 DD UNIT=SYSDA, SPACE=(3500, (10, 10)), DCB=(RECFM=F, BLKSIZE=80)
++STEPLIB DD UNIT=SDSYS, VOL=SER=EVSY, DISP=(SHR, PASS),
IEF6531 SUBSTITUTION JCL - UNIT=2314, VOL=SER=SYSAUX, DISP=(SHR, PASS),
++      DSN=CPRELOAD&STEPLIB
IEF5531 SUBSTITUTION JCL - DSN=CSMP.LOAD.SYSTEM
++FT03F001 DD SYSOUT=A
++FT08F001 DD SYSOUT=A, DCB=(RECFM=FBA, LRECL=133, BLKSIZE=1330)
++FT06F001 DD SYSOUT=A, DCB=(RECFM=FBA, LRECL=133, BLKSIZE=1330)
//T.SYSIN DD *
IEF2361 ALLLOC. FOR H01NASA T TRANS
IEF2371 134 ALLOCATED TO CSMPLIB
IEF2371 133 ALLOCATED TO FT01F001
IEF2371 133 ALLOCATED TO FT02F001
IEF2371 134 ALLOCATED TO FT05F001
IEF2371 133 ALLOCATED TO FT07F001
IEF2371 134 ALLOCATED TO FT13F001
IEF2371 133 ALLOCATED TO FT14F001
IEF2371 134 ALLOCATED TO STEPLIB
IEF2371 134 ALLOCATED TO FT03F001
IEF2371 133 ALLOCATED TO FT08F001
IEF2371 134 ALLOCATED TO FT06F001
IEF2371 133 ALLOCATED TO SYSIN

```

REPRODUCIBILITY OF THE
ORIGINAL PAGE IS POOR

\$\$\$CONTINUOUS SYSTEM MODELING PROGRAM III VIM2 TRANSLATOR OUTPUT\$\$\$

 TITLE RUDDER RUN 10 DEG/SEC COMMAND TP 6.2C.75
 *THE FOLLOWING LISTING CORRESPONDS TO A SPECIAL LISTING FOR
 *RUDDER SPEEDBRAKE ANALYSIS

CARD INDEX

CARD NUMBER	REPRESENTS
1	JOB CARD
2	INPUT JCL
24	INDEX CARDS
25	INITIAL
71	DYNAMIC
581	TERMINAL
629	OUTPUT JCL

INITIAL

MACRO TF=RUDDOR(TOLP,RP,RATE,G1,G2,G3,EFF)

PROCEDURAL

```

X1=SIGN(1.0,TOLP)
X2=SIGN(1.0,RATE)
IF((X1.GT.0.0).AND.(X2.GT.0.0)) GO TO 10
IF((X1.LT.0.0).AND.(X2.LT.0.0)) GO TO 10
IF(X1.NE.X2) GO TO 40
10 TF=(TOLP+RP)*EFF/(3*G1*G2*G3)
GO TO 50
40 TF=(TCLP+RP)/(3*G1*G2*G3*EFF)
50 CONTINUE

```

END MACRO

MACRO SBTf=SBTOR(SLP,SRP,SRATE,SG1,SG2,SG3,SEFF)

PROCEDURAL

```

X1=SIGN(1.0,SLP)
X2=SIGN(1.0,SRATE)
IF((X1.GT.0.0).AND.(X2.GT.0.0)) GO TO 15
IF((X1.LT.0.0).AND.(X2.LT.0.0)) GO TO 15
IF(X1.NE.X2) GO TO 25
15 SBTf=(SLP-SRP)*SEFF/(3*SG1*SG2*SG3)
GO TO 35
25 SBTf=(SLP-SRP)/(3*SG1*SG2*SG3*SEFF)
35 CONTINUE

```

END MACRO

* 50

FUNCTION HINGE = (-49.3,1.025E6),(0,0),(49.3,-1.025E6) 50

*RSLP = SLOPE OF INPUT FUNCTION

CONST RSLP1=10.0,RSLP2=10.0,RSLP3=10.0,RSLP4=10.0

*S3SLP = SLOPE OF INPUT FUNCTION

CONST S3SLP1=0.0,S3SLP2=0.0,S3SLP3=0.0,S3SLP4=0.0

CONST IC=0.0

CONST MR=0.00207,BR=1.3860,XRM=0.065,DELTX3=0.0,DELTX2=0.0,...

KB=0.52,PS1=2800.0,PS2=2800.0,PS3=2800.0,KQP=33.2,...

BETA=1.8E5,M=3.88,KPO=0.004,TC=9.0,JMR=C.00636,JM=0.00565,...

BH=0.015,GS=7.32,GMR=1.47,GMS=4.46,...

GH=473.921,JP=2579.0,BPP=15500.0,TPC=0.0,IL=8.0,DELTX1=0.0

CONST AC=0.008975,AP=0.0276,AS=0.193,BP=0.0648,C1=185.2,C2=0.1056,...

C3=8.76E-5,CP=5.03E-5,CQ=4.5866,KFBL=6.22,KP=1200.0,...

KN=1.38F-4,KT=0.045,MP=6.83E-5,LN=0.01693,VT=0.0838,...

```

VT2=0.62,X0=0.00185,RPS=2800.0,SPS=2800.0,TAUC=0.1,KAMP=17.5,...
WD=314.0,LD=0.707,TDT=0.004,KPT=0.00167,KFB=10.571177,KC=1.91617
CONST Y1=1.0,Y2=1.0,Z1=1.0,Z2=1.0
CONST KSB=0.88438,KR=0.841328,RAD=57.29578,KFBSB=5.81093,KRV=0.1845,...
KSBV=0.10142
CONST HS=0.0476475,HMR=0.02088577,HMSB=0.02088577,HH=2.9088E-3
CONST (SRX3=0.0,DSB X2=0.0,DSBX1=0.0
CONST XSM=0.065,D=0.08276,EG=0.7872,FRC=13.0
CONST KHC=0.397E8,KHST=0.02,KFL=0.0011
CONST CCNV=0.2527
*CONVERT FROM RADIANS/SEC TO RPM
CONST CRPM = 9.5492966
CONST C1=0.0, IC2=0.0

```

DYNAMIC

*RJDDER INPUT EQUATIONS FROM ASA ELECTRONICS

RIN1 = RSLP1*RAMP(0.005)

RIN2 = RSLP2*RAMP(0.005)

RIN3 = RSLP3*RAMP(0.005)

RIN4 = RSLP4*RAMP(0.005)

*SPEF DBRAKE INPUT EQUATIONS FROM ASA ELECTRONICS

SIN1 = SBSLP1*RAMP(0.005)

SIN2 = SBSLP2*RAMP(0.005)

SIN3 = SBSLP3*RAMP(0.005)

SIN4 = SBSLP4*RAMP(0.005)

*RJDDER EQUATIONS

RVIN1=RVIN1*KRV

RVIN2=RVIN2*KRV

RVIN3=RVIN3*KRV

RVIN4=RVIN4*KRV

RASAI1=(RVIN1-RVZ1)*KAMP-RVK1

RASAI2=(RVIN2-RVZ2)*KAMP-RVK2

RASAI3=(RVIN3-RVZ3)*KAMP-RVK3

RASAI4=(RVIN4-RVZ4)*KAMP-RVK4

RIL1=LIMIT(-IL,IL,RASAI1)

*RJDDER SERVO(Secondary Actuator) NC. 1

RT1=RIL1*KTM

RTIN1=HSTRSS(IL,-KHST,KHST,RT1)

RTT1=RTIN1-RFP1*KN-XRL*KFBL

PFX1=RFT1*LN

RFXML1=LIMIT(-KFL,KFL,RFX1)

RFIFLW=(2.0*C1*RFXML1)-(2.0*(AP-AC)*DERIV(IC,RXSS1))

DR1X1=RFIFLW-(R1X1*G1/G2)

R1X1=INTGRL(IC,DR1X1)

RFP1=R1X1/G2

G1=C3+C2*X0

G2=VT2/(2.0*BETA)

RFIFT=RFP1*(AP-AC)-RSAP1*AC

*100-----100

RDDSP1=RFIFT-(RDDSP1*(BP/MP))-(RSP1*(KP/MP))

RDDSP1=INTGRL(IC,RDDSP1)

RSP1=INTGRL(IC,RDDSP1)

RXSS1=RSP1/MP

RPVS1=RPS-(RSAP1*SIGN(1.0,RXSS1))

RIQO=CQ*RXSS1*SIGN(1.0,RPVS1)*SQRT(ABS(RPVS1))

RQXSA1=RIQO+AP*DERIV(IC,RXSS1)

RDQSA1=(RQXSA1-(RSAP1*CP1))/AS

RXSA1=INTGRL(IC,RDQSA1)

RXA1=RXSA1-XRL

RSAP1=RXA1*((4.0*BETA*AS)/VT2)

```

RF1=RSAPT*AS
RIL2=LIMIT(-IL,IL,RASA12)
*RUDDER SERVO(SECONDARY ACTUATOR) NO. 2
RT2=RIL2*KTM
RTIN2=HSTRSS(IC,-KHST,KHST,RT2)
RFT2=RTIN2-RFP2*KN-XRL*KPBL
RFX2=RFT2*LN
RFXLM2=LIMIT(-KFL,KFL,RFX2)
RF2FLW=(2.0*C1*RFXLM2)-(2.0*(AP-AC)*DERIV(IC,RXSS2))
DR2X1=RF2FLW-(R2X1*G1/G2)
R2X1=INTGRL(IC,DR2X1)
RFP2=R2X1/G2
RF2FT=RFP2*(AP-AC)-RSAP2*AC
RDDSP2=RF2FT-(RDS2*(BP/MP))-(RSP2*(KP/MP))
RDS2=INTGRL(IC,RDDSP2)
RSP2=INTGRL(IC,RDS2)
RXSS2=RSP2/MP
RPVS2=RPS-(RSAP2*SIGN(1.0,RXSS2))
R2Q0=CQ*RXSS2*SIGN(1.0,RPVS2)*SQRT(ABS(RPVS2))
RQXSA2=R2Q0+AP*DERIV(IC,RXSS2)
RDXSA2=(RQXSA2-(RSAP2*CP))/AS
RXSA2=INTGRL(IC,RDXSA2)
RXA2=RXSA2-XRL
RSAP2=RXA2*((4.0*BETA*AS)/VT2)
RF2=RSAP2*AS
RIL3=LIMIT(-IL,IL,RASA13)
*RUDDER SERVO(SECONDARY ACTUATOR) NO. 3
RT3=RIL3*KTM
RTIN3=HSTRSS(IC,-KHST,KHST,RT3)
RFT3=RTIN3-RFP3*KN-XRL*KPBL
RFX3=RFT3*LN
RFXLM3=LIMIT(-KFL,KFL,RFX3)
RF3FLW=(2.0*C1*RFXLM3)-(2.0*(AP-AC)*DERIV(IC,RXSS3))
DR3X1=RF3FLW-(R3X1*G1/G2)
R3X1=INTGRL(IC,DR3X1)
RFP3=R3X1/G2
RF3FT=RF3FLW*(AP-AC)-RSAP3*AC
RDDSP3=RF3FT-(RDS3*(BP/MP))-(RSP3*(KP/MP))
RDS3=INTGRL(IC,RDDSP3)
RSP3=INTGRL(IC,RDS3)
RXSS3=RSP3/MP
*150-----150
RPVS3=RPS-(RSAP3*SIGN(1.0,RXSS3))
R3Q0=CQ*RXSS3*SIGN(1.0,RPVS3)*SQRT(ABS(RPVS3))
RQXSA3=R3Q0+AP*DERIV(IC,RXSS3)
RDXSA3=(RQXSA3-(RSAP3*CP))/AS
RXSA3=INTGRL(IC,RDXSA3)
RXA3=RXSA3-XRL
RSAP3=RXA3*((4.0*BETA*AS)/VT2)
RF3=RSAP3*AS
RIL4=LIMIT(-IL,IL,RASA14)
RT4=RIL4*KTM
RTIN4=HSTRSS(IC,-KHST,KHST,RT4)
RFT4=RTIN4-RFP4*KN-XRL*KPBL
RFX4=RFT4*LN
RFXLM4=LIMIT(-KFL,KFL,RFX4)
RF4FLW=(2.0*C1*RFXLM4)-(2.0*(AP-AC)*DERIV(IC,RXSS4))
DR4X1=RF4FLW-(R4X1*G1/G2)
R4X1=INTGRL(IC,DR4X1)

```

```

RFP4=R4X1/AG2
RF4FT=RF4*(AP-AC)-RSAP4*AC
RDO5P4=RF4FT-(RDO5P4*(BP/MP))-(RSP4*(KP/MP))
RDO5P4=INTGRL(IC,RDO5P4)
RSP4=INTGRL(IC,RDO5P4)
RXSS4=RSP4/MP
RPVS4=RPS-(RSAP4*SIGN(1.0,RXSS4))
R4QO=CQ*RXSS4*SIGN(1.0,RPVS4)*SQRT(ABS(RPVS4))
RQXSA4=R4QO*AP*DERIV(IC,RXSS4)
RDXSA4=(RQXSA4-(RSAP4*CP))/AS
RXSA4=INTGRL(IC,RDXSA4)
RXA4=RXSA4-XR1
RSAP4=RXA4*((4.0*BETA*AS)/VT2)
RF4=RSAP4*AS
FRTOT=RF1+RF2+RF3+RF4
FR=FRTOT-RBF-RSVFF
RSVFF=FCNSM1 XROOT,-FRC,0.0,FRC)
XROOT=(FR-XROFB)*Y2
XROFB=XRD*(BR/MR)
XRD=INTGRL(IC,XROOT)
XROOT=(XRD/MR)*Y1
XR=INTGRL(IC,XROOT)
XRL=LIMIT(-XRM,XRM,XR)
PROCEDURE Y1,Y2=RRATE(XR,XRM)
YY1=COMPAR(XR,XRM)
IF (YY1.LT.1.0)GO TO 20
Y1=0.0
Y2=0.0
GO TO 90
20 Y1=1.0
Y2=1.0
YY2=COMPAR(XR,-XRM)
IF(YY2.GT.0.0)GO TO 80
----- 00
*200 Y1=0.0
Y2=0.0
GO TO 90
80 Y2=1.0
Y1=1.0
90 CONTINUE
END PROCEDURE
RX3=XRL+DELT X3
RX2=XRL+DELT X2
RX1=XRL+DELT X1
PV3=PS3-SIGN(1.0,RX3)*PL3LIM
PV2=PS2-SIGN(1.0,RX2)*PL2LIM
PV1=PS1-SIGN(1.0,RX1)*PL1LIM
RBF3=PV3*RX3*K8
RBF2=PV2*RX2*K8
RBF1=PV1*RX1*K8
RBF=RBF3+RBF2+RBF1
QL3=KQP*SIGN(1.0,PV3)*SQRT(ABS(PV3))*RX3
QL3GPM = CONV*QL3
QL2=KQP*SIGN(1.0,PV2)*SQRT(ABS(PV2))*RX2
QL2GPM = CONV*QL2
QL1=KQP*SIGN(1.0,PV1)*SQRT(ABS(PV1))*RX1
QL1GPM = CONV*QL1
RSVM13=QL3-(KQP*PL3LIM)-(D*THDOT3)
RSVM12=QL2-(KQP*PL2LIM)-(D*THDOT2)

```

```

RSVM1=Q11-(KPO*PL1LIM)-(D*THDOT1)
RSV13=INTGRL(IC,RSV13)
RSV12=INTGRL(IC,RSV12)
RSV11=INTGRL(IC,RSV11)
PL3=(4.0*BETA*RSV13)/VM
PL2=(4.0*BETA*RSV12)/VM
PL1=(4.0*BETA*RSV11)/VM
PL3LIM=LIMIT(-PS3,PS3,PL3)
PL2LIM=LIMIT(-PS2,PS2,PL2)
PL1LIM=LIMIT(-PS1,PS1,PL1)
*RUDDER MOTOR 3
RMIN3=PL3LIM*D
RTF3=RUDDTOR(TLP,TRP,THDOT3,GH,GMR,GS,EG)
RMSUM1=RTSW3-RTF3
RTSW3=FCNSW(THDOT3,-TC,0.0,TC)
RMSUM2=RMSUM1+(THDOT3*BM)
THD3=(RMIN3-RMSUM2)/JMR
THDOT3=INTGRL(IC,THD3)
RTHET3=INTGRL(IC,THDOT3)
*RUDDER MOTOR 2
RMIN2=PL2LIM*D
RTF2=RUDDTOR(TLP,TRP,THDOT2,GH,GMR,GS,FG)
RMSUM3=RTSW2-RTF2
RTSW2=FCNSW(THDOT2,-TC,0.0,TC)
RMSUM4=RMSUM3+(THDOT2*BM)
THD2=(RMIN2-RMSUM4)/JMR
THDOT2=INTGRL(IC,THD2)
RTHET2=INTGRL(IC,THDOT2)
*RUDDER MOTOR 1
RMIN1=PL1LIM*D
RTF1=RUDDTOR(TLP,TRP,THDOT1,GH,GMR,GS,EG)
*250-----250
RMSUM5=RTSW1-RTF1
RTSW1=FCNSW(THDOT1,-TC,0.0,TC)
RMSUM6=RMSUM5+(THDOT1*BM)
THD1=(RMIN1-RMSUM6)/JMR
THDOT1=INTGRL(IC,THD1)
RTHET1=INTGRL(IC,THDOT1)
*CONVERT FROM RADIAN/SEC TO RPM
RM1RPM = CRPM*THDOT1
RM2RPM = CRPM*THDOT2
RM3RPM = CRPM*THDOT3
*RUDDER MECHANICAL SUMMING DIFFERENTIALS
DELTR1=RTHET1/GS
DELTR2=RTHET2/GS
DELTR3=RTHET3/GS
RUDSUM=DELTR3+DELTR2+DELTR1
DELTR=HSTRSS(IC,-HS,HS,RUDSUM)
RUDDER=DELTR/(GMR*GH)
DRV11=RSAP1*(KPT/TDT)-RV11/TDT
RV11=INTGRL(IC,DRV11)
RODV11=RV11-(RODV11*(2.0*LD*WD))-RV11
RV11=INTGRL(IC,RODV11)
RV11=(WD*WD)*INTGRL(IC,RODV11)
RDVK1=KC*DERIV(IC,RV11)
RDVK11=RDVK1-(RVK1/TAUC)
RVK1=INTGRL(IC,RDVK11)
RLV1=RUDDER*(KFA/TDT)-RLV1/TDT
RLV1=INTGRL(IC,RLV1)

```

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RDDVZ1=RLVDT1-(RDVZ1*(2.0*LD*WD))-RVZ1
RDVZ1=INTGRL(IC,RDDVZ1)
RVZ1=(WD*WD)*INTGRL(IC,RDVZ1)
DRVL2=RSAP2*(KPT/TDT)-RVL2/TDT
RVL2=INTGRL(IC,DRVL2)
RDDVM2=RVL2-(RDVM2*(2.0*LD*WD))-RVW2
RDVM2=INTGRL(IC,RDDVM2)
RVW2=(WD*WD)*INTGRL(IC,RDVM2)
RDVK2=KC*DERIV(IC,RVW2)
RDVKI2=RDVK2-(RVK2/TAUC)
RVK2=INTGRL(IC,RDVKI2)
RLV2=RUOFB*(KFB/TDT)-RLVDT2/TDT
RLVDT2=INTGRL(IC,RLV2)
RDDVZ2=RLVDT2-(RDVZ2*(2.0*LD*WD))-RVZ2
RDVZ2=INTGRL(IC,RDDVZ2)
RVZ2=(WD*WD)*INTGRL(IC,RDVZ2)
DRVL3=RSAP3*(KPT/TDT)-RVL3/TDT
RVL3=INTGRL(IC,DRVL3)
RDDVM3=RVL3-(RDVM3*(2.0*LD*WD))-RVW3
RDVM3=INTGRL(IC,RDDVM3)
RVW3=(WD*WD)*INTGRL(IC,RDVM3)
RDVK3=KC*DERIV(IC,RVW3)
RDVKI3=RDVK3-(RVK3/TAUC)
RVK3=INTGRL(IC,RDVKI3)
RLV3=RUOFB*(KFB/TDT)-RLVDT3/TDT
RLVDT3=INTGRL(IC,RLV3)
RDDVZ3=RLVDT3-(RDVZ3*(2.0*LD*WD))-RVZ3
RDVZ3=INTGRL(IC,RDDVZ3)

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•330-

300

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RVZ3=(WD*WD)*INTGRL(IC,RDVZ3)
DRVL4=RSAP4*(KPT/TDT)-RVL4/TDT
RVL4=INTGRL(IC,DRVL4)
RDDVM4=RVL4-(RDVM4*(2.0*LD*WD))-RVW4
RDVM4=INTGRL(IC,RDDVM4)
RVW4=(WD*WD)*INTGRL(IC,RDVM4)
RDVK4=KC*DERIV(IC,RVW4)
RDVKI4=RDVK4-(RVK4/TAUC)
RVK4=INTGRL(IC,RDVKI4)
RLV4=RUOFB*(KFB/TDT)-RLVDT4/TDT
RLVDT4=INTGRL(IC,RLV4)
RDDVZ4=RLVDT4-(RDVZ4*(2.0*LD*WD))-RVZ4
RDVZ4=INTGRL(IC,RDDVZ4)
RVZ4=(WD*WD)*INTGRL(IC,RDVZ4)

```

•SPEEDBRAKE EQUATIONS

```

SVIN1=SVIN1*KS8V
SVIN2=SVIN2*KS8V
SVIN3=SVIN3*KS8V
SVIN4=SVIN4*KS8V
SASAI1=(SVIN1-SVZ1)*KAMP-SVK1
SASAI2=(SVIN2-SVZ2)*KAMP-SVK2
SASAI3=(SVIN3-SVZ3)*KAMP-SVK3
SASAI4=(SVIN4-SVZ4)*KAMP-SVK4
SIL1=LIMIT(-IL,IL,SASAI1)

```

•SPEEDBRAKE SERVO(SECONDARY ACTUATOR) NO. 1

```

ST1=SIL1*KTH
STIN1=HSTRSS(IC,-KHST,KHST,ST1)
SFT1=STIN1-SFPI*KAN-XSL*KFBL
SFX1=SFT1*LN
SFXLM=LIMIT(-KFL,KFL,SFX1)

```

```

SF1FLW=(2.0*C1*SF1LM1)-(2.0*(AP-AC)*DERIV(IC,SXSS1))
DS1X1=SF1FLW-(S1X1*G1/G2)
S1X1=INTGRL(IC,DS1X1)
SF1P1=S1X1/G2
SF1FT=SF1P1*(AP-AC)-SSAP1*AC
SDDSP1=SF1FT-(SDSP1*(BP/MP))-(SSP1*(KP/MP))
SDSP1=INTGRL(IC,SDDSP1)
SSP1=INTGRL(IC,SDSP1)
SXSS1=SSP1/MP
SPVS1=SPS-(SSAP1*SIGN(1.0,SXSS1))
S1Q0=CQ*SXSS1*SIGN(1.0,SPVS1)*SQRT(ABS(SPVS1))
SQXSA1=S1Q0*AP*DERIV(IC,SXSS1)
SDXSA1=(SQXSA1-(SSAP1*C1))/AS
SXSA1=INTGRL(IC,SDXSA1)
SXA1=SXSA1-XSL
SSAP1=SXA1*((4.0*BETA*AS)/VT2)
SF1=SSAP1*AS
SIL2=LIMIT(-IL,IL,SASAI2)
*SPED BRAKE SERVO(SECONDARY ACTUATOR) NO. 2
ST2=SIL2*KTM
STIN2=HSTRSS(IC,-KHST,KHST,ST2)
SFT2=STIN2-SFP2*KN-XSL*KFRL
SFX2=SFT2*LN
-----350
SF1LM2=LIMIT(-KFL,KFL,SFX2)
SF2FLW=(2.0*C1*SF1LM2)-(2.0*(AP-AC)*DERIV(IC,SXSS2))
DS2X1=SF2FLW-(S2X1*G1/G2)
S2X1=INTGRL(IC,DS2X1)
SF2P2=S2X1/G2
SF2FT=SF2P2*(AP-AC)-SSAP2*AC
SDDSP2=SF2FT-(SDSP2*(BP/MP))-(SSP2*(KP/MP))
SDSP2=INTGRL(IC,SDDSP2)
SSP2=INTGRL(IC,SDSP2)
SXSS2=SSP2/MP
SPVS2=SPS-(SSAP2*SIGN(1.0,SXSS2))
S2Q0=CQ*SXSS2*SIGN(1.0,SPVS2)*SQRT(ABS(SPVS2))
SQXSA2=S2Q0*AP*DERIV(IC,SXSS2)
SDXSA2=(SQXSA2-(SSAP2*C1))/AS
SXSA2=INTGRL(IC,SDXSA2)
SXA2=SXSA2-XSL
SSAP2=SXA2*((4.0*BETA*AS)/VT2)
SF2=SSAP2*AS
SIL3=LIMIT(-IL,IL,SASAI3)
*SPED BRAKE SERVO(SECONDARY ACTUATOR) NO. 3
ST3=SIL3*KTM
STIN3=HSTRSS(IC,-KHST,KHST,ST3)
SFT3=STIN3-SFP3*KN-XSL*KFRL
SFX3=SFT3*LN
SF1LM3=LIMIT(-KFL,KFL,SFX3)
SF3FLW=(2.0*C1*SF1LM3)-(2.0*(AP-AC)*DERIV(IC,SXSS3))
DS3X1=SF3FLW-(S3X1*G1/G2)
S3X1=INTGRL(IC,DS3X1)
SF3P3=S3X1/G2
SF3FT=SF3P3*(AP-AC)-SSAP3*AC
SDDSP3=SF3FT-(SDSP3*(BP/MP))-(SSP3*(KP/MP))
SDSP3=INTGRL(IC,SDDSP3)
SSP3=INTGRL(IC,SDSP3)
SXSS3=SSP3/MP
SPVS3=SPS-(SSAP3*SIGN(1.0,SXSS3))

```

```

S3Q0=CQ*SXSS3*SIGN(1.0,SPVS3)*SQRT(ABS(SPV3))
SQXSA3=S3Q0+AP*DERIV(IC,SXSS3)
SDXSA3=(SQXSA3-(SSAP3*CP))/AS
SXSA3=INTGRL(IC,SDXSA3)
SXA3=SXSA3-XSL
SSAP3=SXA3*((4.0*BETA*AS)/VT2)
SF3=SSAP3*AS
SIL4=LIMIT(-IL,IL,SASAI4)
ST4=SIL4*KTM
STIN4=HSTRSS(IC,-KHST,KHST,ST4)
SF4=STIN4-SFP4*KN-XSL*KFBL
SFX4=SF4*LN
SFXLM4=LIMIT(-KFL,KFL,SFX4)
SF4FLW=(2.0*CI*SFXLM4)-(2.0*(AP-AC)*DERIV(IC,SXSS4))
DS4X1=SF4FLW-(S4X1*GI/G2)
S4X1=INTGRL(IC,DS4X1)
-----400
*400 SFP4=S4X1/G2
SF4FT=SF4*((AP-AC)-SSAP4*AC)
SDDSP4=SF4FT-(SDSP4*(BP/MP))-(SFP4*(KP/MP))
SDSP4=INTGRL(IC,SDDSP4)
SSP4=INTGRL(IC,SDSP4)
SXSS4=SSP4/MP
SPVS4=SPS-(SSAP4*SIGN(1.0,SXSS4))
S4Q0=CQ*SXSS4*SIGN(1.0,SPVS4)*SQRT(ABS(SPV4))
SQXSA4=S4Q0+AP*DERIV(IC,SXSS4)
SDXSA4=(SQXSA4-(SSAP4*CP))/AS
SXSA4=INTGRL(IC,SDXSA4)
SXA4=SXSA4-XSL
SSAP4=SXA4*((4.0*BETA*AS)/VT2)
SF4=SSAP4*AS
FSTOT=SF1+SF2+SF3+SF4
FS=FSTOT-SRF-SSVFF
SSVFF=FCNSW(XSDOT,-FRC,0.0,FRC)
XSDOT=(FS-XSDFB)*Z2
XSDFB=XSD*(BR/MR)
XSD=INTGRL(IC,XSDOT)
XSDOT=(XSD/MR)*Z1
XS=INTGRL(IC,XSDOT)
XSL=LIMIT(-XSM,XSM,XS)
PROCEDURE Z1,Z2=SRATE(XS,XSM)
Z1=CCMPAR(XS,XSM)
IF (Z1.LT.0.0)GO TO 30
Z1=0.0
Z2=0.0
GO TO 60
30 Z1=1.0
Z2=1.0
Z2=CCMPAR(XS,-XSM)
IF (Z2.GT.0.0)GO TO 70
Z1=0.0
Z2=0.0
GO TO 60
70 Z2=1.0
Z1=1.0
60 CONTINUE
END PROCEDURE
SX3=XSL+DSBX3
SX2=XSL+DSBX2

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SX1=XSL+DS9X1
SPV3=PS3-SIGN(1.0,SX3)*PSL3LM
SPV2=PS2-SIGN(1.0,SX2)*PSL2LM
SPV1=PS1-SIGN(1.0,SX1)*PSL1LM
SBF3=SPV3*SX3*K8
SBF2=SPV2*SX2*K8
SBF1=SPV1*SX1*K8
SBF=SBF3+SBF2+SBF1
-----450-----
QSL3=KQP*SIGN(1.0,SPV3)*SQRT(ABS(SPV3))*SX3
QSL3GM=CONV*QSL3
QSL2=KQP*SIGN(1.0,SPV2)*SQRT(ABS(SPV2))*SX2
QSL2GM=CONV*QSL2
QSL1=KQP*SIGN(1.0,SPV1)*SQRT(ABS(SPV1))*SX1
QSL1GM=CONV*QSL1
SSVM13=QSL3-(KQP*PSL3LM)-(D*THDTS3)
SSVM12=QSL2-(KQP*PSL2LM)-(D*THDTS2)
SSVM11=QSL1-(KQP*PSL1LM)-(D*THDTS1)
SSV11=INTGRL(IC,SSVM11)
SSV12=INTGRL(IC,SSVM12)
SSV13=INTGRL(IC,SSVM13)
PSL3=(4.0*BETA*SSV13)/VM
PSL2=(4.0*BETA*SSV12)/VM
PSL1=(4.0*BETA*SSV11)/VM
PSL3LM=LIMIT(-PS3,PS3,PSL3)
PSL2LM=LIMIT(-PS2,PS2,PSL2)
PSL1LM=LIMIT(-PS1,PS1,PSL1)
* SPEED BRAKE MOTOR 3
SMIN3=PSL3LM*D
STF3=SBTOR(TLP,TRP,THDTS3,GH,GMS,GS,EG)
SMSUM1=STSW3-STF3
STSW3=FCNSW(THDTS3,-TC,0.0,TC)
SMSUM2=SMSUM1+(THDTS3*BM)
STHD3=(SMIN3-SMSUM2)/JMS
THDTS3=INTGRL(IC,STHD3)
STHET3=INTGRL(IC,THDTS3)
* SPEED BRAKE MOTOR 2
SMIN2=PSL2LM*D
STF2=SBTOR(TLP,TRP,THDTS2,GH,GMS,GS,EG)
SMSUM3=STSW2-STF2
STSW2=FCNSW(THDTS2,-TC,0.0,TC)
SMSUM4=SMSUM3+(THDTS2*BM)
STHD2=(SMIN2-SMSUM4)/JMS
THDTS2=INTGRL(IC,STHD2)
STHET2=INTGRL(IC,THDTS2)
* SPEED BRAKE MOTOR 1
SMIN1=PSL1LM*D
STF1=SBTOR(TLP,TRP,THDTS1,GH,GMS,GS,EG)
SMSUM5=STSW1-STF1
STSW1=FCNSW(THDTS1,-TC,0.0,TC)
SMSUM6=SMSUM5+(THDTS1*BM)
STHD1=(SMIN1-SMSUM6)/JMS
THDTS1=INTGRL(IC,STHD1)
STHET1=INTGRL(IC,THDTS1)
* CONVERT FROM RADIANS/SEC TO RPM
SM1RPM=CRPM*THDTS1
SM2RPM=CRPM*THDTS2
SM3RPM=CRPM*THDTS3
* SPEED BRAKE MECHANICAL SUMMING DIFFERENTIALS

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```

DELTS1=STHET1/GS
DELTS2=STHET2/GS
DELTS3=STHET3/GS
SBSUM=DELTS3+DELTS2+DELTS1
DELTSB=HSTRSS(IC,-HS,HS,SBSUM)
SBFB=2.0*DELTSB/(GMS*GH)
OSVL1=SSAP1*(KPT/TDT)-SVL1/TDT
SVL1=INTGRL(IC,OSVL1)
SDDVW1=SVL1-(SDVW1*(2.0*LD*WD))-SVW1
SDVW1=INTGRL(IC,SDDVW1)
SVW1=(WD*WD)*INTGRL(IC,SDVW1)

```

*500-

500

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SDVK1=KC*DERIV(IC,SVW1)
SDVK11=SDVK1-(SVK1/TAUC)
SVK1=INTGRL(IC,SDVK11)
SLV1=SBFB*(KFBSB/TDT)-SLVDT1/TDT
SLVDT1=INTGRL(IC,SLV1)
SDDVZ1=SLVDT1-(SDVZ1*(2.0*LD*WD))-SVZ1
SDVZ1=INTGRL(IC,SDDVZ1)
SVZ1=(WD*WD)*INTGRL(IC,SDVZ1)
DSVL2=SSAP2*(KPT/TDT)-SVL2/TDT
SVL2=INTGRL(IC,DSVL2)
SDDVW2=SVL2-(SDVW2*(2.0*LD*WD))-SVW2
SDVW2=INTGRL(IC,SDDVW2)
SVW2=(WD*WD)*INTGRL(IC,SDVW2)
SDVK2=KC*DERIV(IC,SVW2)
SDVK12=SDVK2-(SVK2/TAUC)
SVK2=INTGRL(IC,SDVK12)
SLV2=SBFB*(KFBSB/TDT)-SLVDT2/TDT
SLVDT2=INTGRL(IC,SLV2)
SDDVZ2=SLVDT2-(SDVZ2*(2.0*LD*WD))-SVZ2
SDVZ2=INTGRL(IC,SDDVZ2)
SVZ2=(WD*WD)*INTGRL(IC,SDVZ2)
DSVL3=SSAP3*(KPT/TDT)-SVL3/TDT
SVL3=INTGRL(IC,DSVL3)
SDDVW3=SVL3-(SDVW3*(2.0*LD*WD))-SVW3
SDVW3=INTGRL(IC,SDDVW3)
SVW3=(WD*WD)*INTGRL(IC,SDVW3)
SDVK3=KC*DERIV(IC,SVW3)
SDVK13=SDVK3-(SVK3/TAUC)
SVK3=INTGRL(IC,SDVK13)
SLV3=SBFB*(KFBSB/TDT)-SLVDT3/TDT
SLVDT3=INTGRL(IC,SLV3)
SDDVZ3=SLVDT3-(SDVZ3*(2.0*LD*WD))-SVZ3
SDVZ3=INTGRL(IC,SDDVZ3)
SVZ3=(WD*WD)*INTGRL(IC,SDVZ3)
DSVL4=SSAP4*(KPT/TDT)-SVL4/TDT
SVL4=INTGRL(IC,DSVL4)
SDDVW4=SVL4-(SDVW4*(2.0*LD*WD))-SVW4
SDVW4=INTGRL(IC,SDDVW4)
SVW4=(WD*WD)*INTGRL(IC,SDVW4)
SDVK4=KC*DERIV(IC,SVW4)
SDVK14=SDVK4-(SVK4/TAUC)
SVK4=INTGRL(IC,SDVK14)
SLV4=SBFB*(KFBSB/TDT)-SLVDT4/TDT
SLVDT4=INTGRL(IC,SLV4)
SDDVZ4=SLVDT4-(SDVZ4*(2.0*LD*WD))-SVZ4
SDVZ4=INTGRL(IC,SDDVZ4)
SVZ4=(WD*WD)*INTGRL(IC,SDVZ4)

```

*RUDDER AND SPEEDBRAKE MIXER

RUD=DELTR/GMR
SB=DELT SB/GMS
LPX=RUD+SB

*530

550

RPX=RUD-SB
LPMIX=HSTRSS(IC,-HMR,HMR,LPX)
RPMIX=HSTRSS(IC,-HMSB,HMSB,RPX)
RPHIN=RPMIX/GH
LPHIN=LPMIX/GH
LPHMS=HSTRSS(IC,-HH,HH,LPHIN)
RPHMS=HSTRSS(IC,-HH,HH,RPHIN)
LPHS=LPHMS-DLP
RPHS=RPHMS-DRP
TLP=LPHS*KHC
TRP=RPHS*KHC
HMLP=AFGEN(HINGE,DSBRHL)
HMRP=-AFGEN(HINGE,DSBRHL)
HLP=HMLP+AFGEN(HINGE,DRRHL)*0.5*SIGN(1.0,DRRHL)
HRP=HMRP+AFGEN(HINGE,DRRHL)*0.5*SIGN(1.0,DRRHL)
LDL=(TLP+HRP-LTPC-LBP)/JP
DDL=(TRP+HRP-RTPC-RBP)/JP
DDL=INTGRL(IC,DDL)
DDR=INTGRL(IC,DDL)

*LEFT PANEL AND RIGHT PANEL DEFLECTION IN RHL PLANE

DLP=INTGRL(IC,DDL)
DRP=INTGRL(IC,DDR)
LTPC=FCNSW(DDL,-TPC,0.0,TPC)
RTPC=FCNSW(DDR,-TPC,0.0,TPC)
LBP=DDL*8PP
RBP=DDR*8PP
LPDEG=DLP*RAD
RPDEG=DRP*RAD
DRRHL=(LPDEG+RPDEG)/2.
DSBRHL=LPDEG-RPDEG

*CHANGING DEFLECTION TO FRL PLANE

DRFRL=DRRHL*KR
DSBRFL=DSBRHL*KSB
* LPACC= LEFT PANEL ACCELERATION (RHL IN DEG/SEC**2)
* LPACC=INTGRL(IC,LPDEG)
* RPACC= RIGHT PANEL ACCELERATION (RHL IN DEG/SEC**2)
* RPACC=INTGRL(IC,RPDEG)
* DSBACC= SPEEDBRAKE SEPARATION ANGLE ACCELERATION (RHL DEG/SEC**2)
* DSBACC=INTGRL(IC,DSBRHL)

TERMINAL

METHOD RKSPX

TIMER DELT=0.00005,OUTDEL=0.010,FINTIM=2.0

LABEL RUDDER RUN TO DEG/SEC COMMAND

TP 6.2(75

OUTPUT RINI,OL1GPM,OL2GPM,OL3GPM
OUTPUT RINI,RM1RPM,RM2RPM,RM3RPM
OUTPUT LPACC,RPACC,DSBACC

OUTPUT RINI,RPDEG
OUTPUT RINI,LPDEG

END
STOP

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APPENDIX B

STEP RESPONSE PLOTS

APPENDIX B FIGURES

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B-5 10-degree Speedbrake step command response	B-7
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B-7 Various Rudder step commands	B-9
B-8 Various Rudder step commands	B-10
B-9 Various Speedbrake step commands	B-11
B-10 Various Speedbrake step commands.	B-12
B-11 LPDEG magnified to observe oscillation.	B-13
B-12 DSBRLH magnified to observe oscillation	B-14

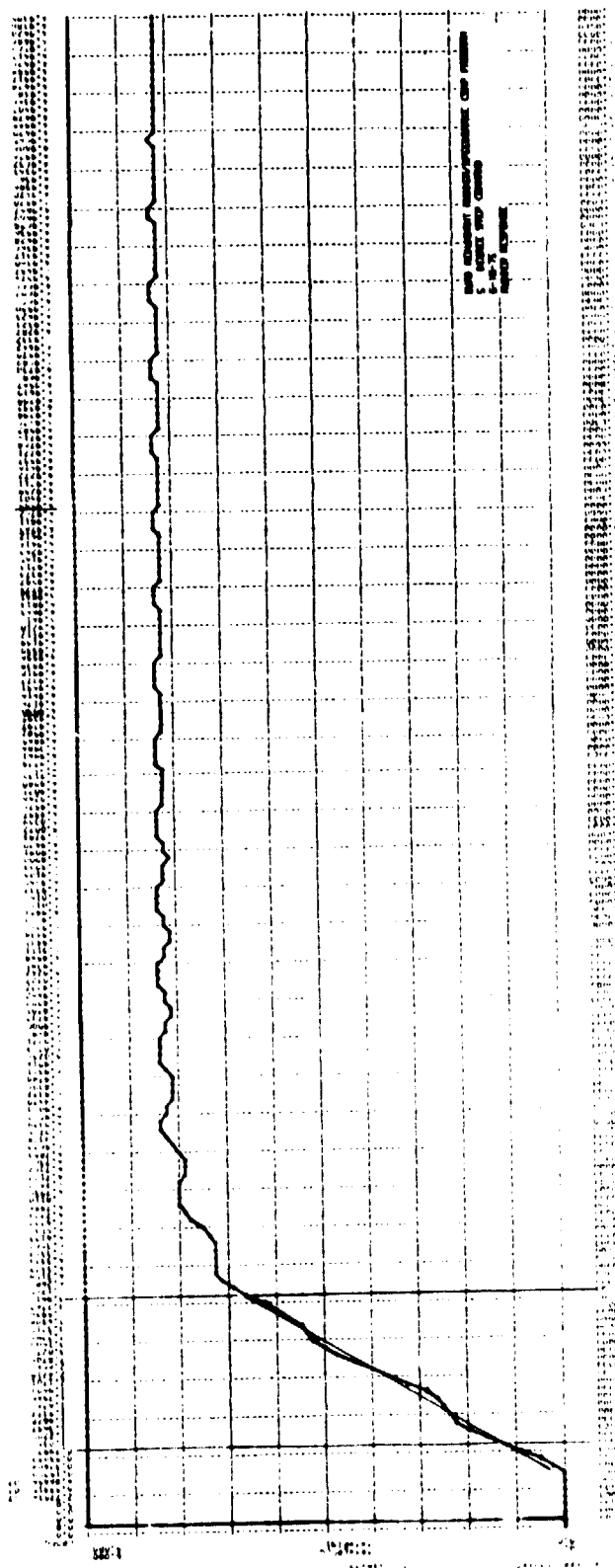


Figure B-1. - 5-degree Rudder step command response.

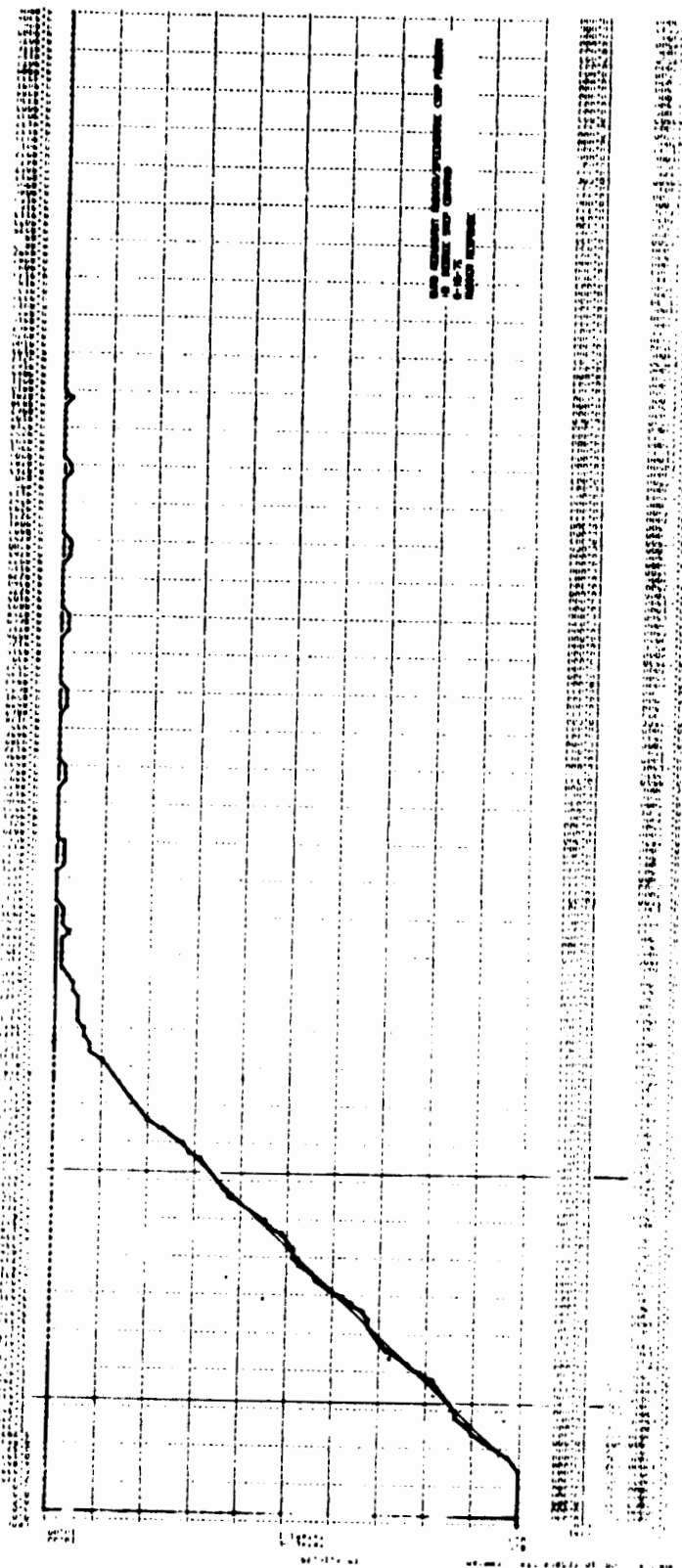


Figure B-2. - 10-degree Rudder step command response.

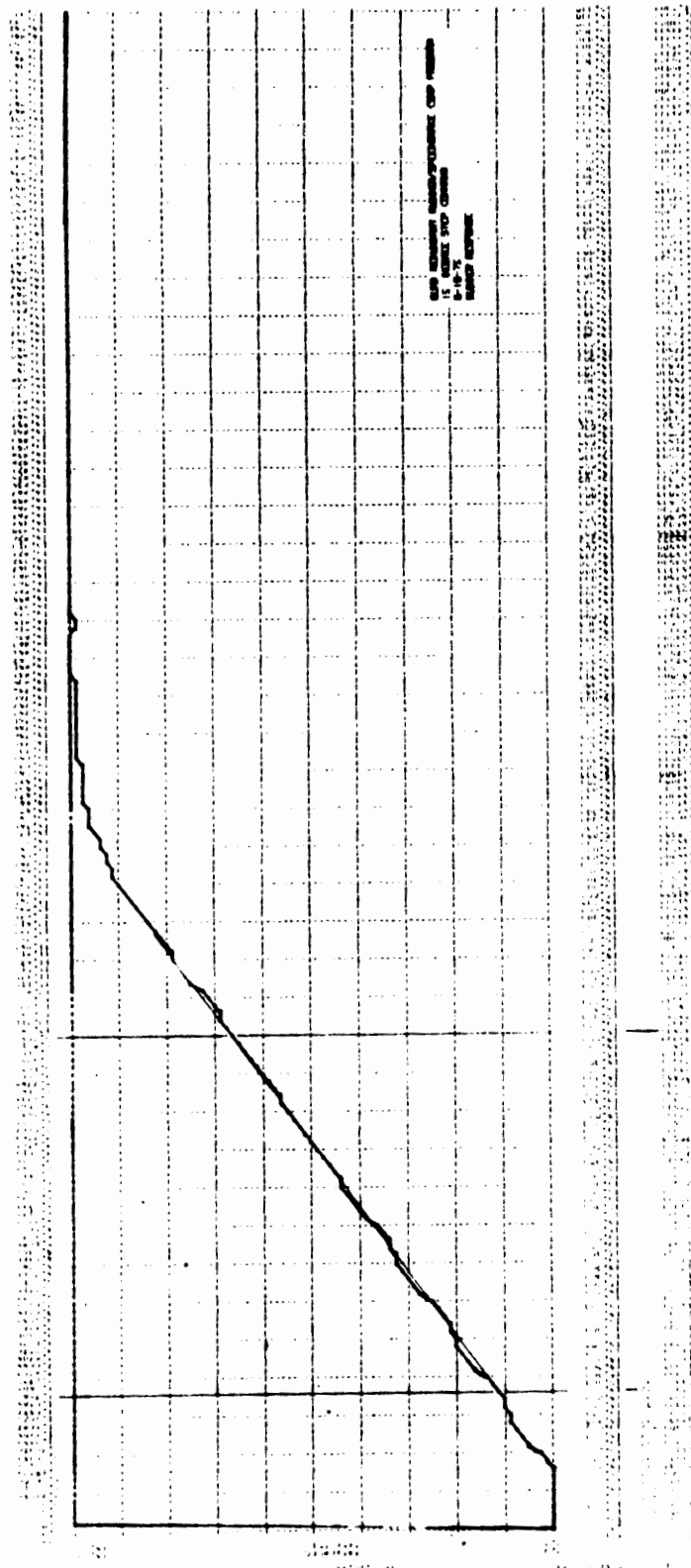


Figure B-3. - 15-degree Rudder step command response.

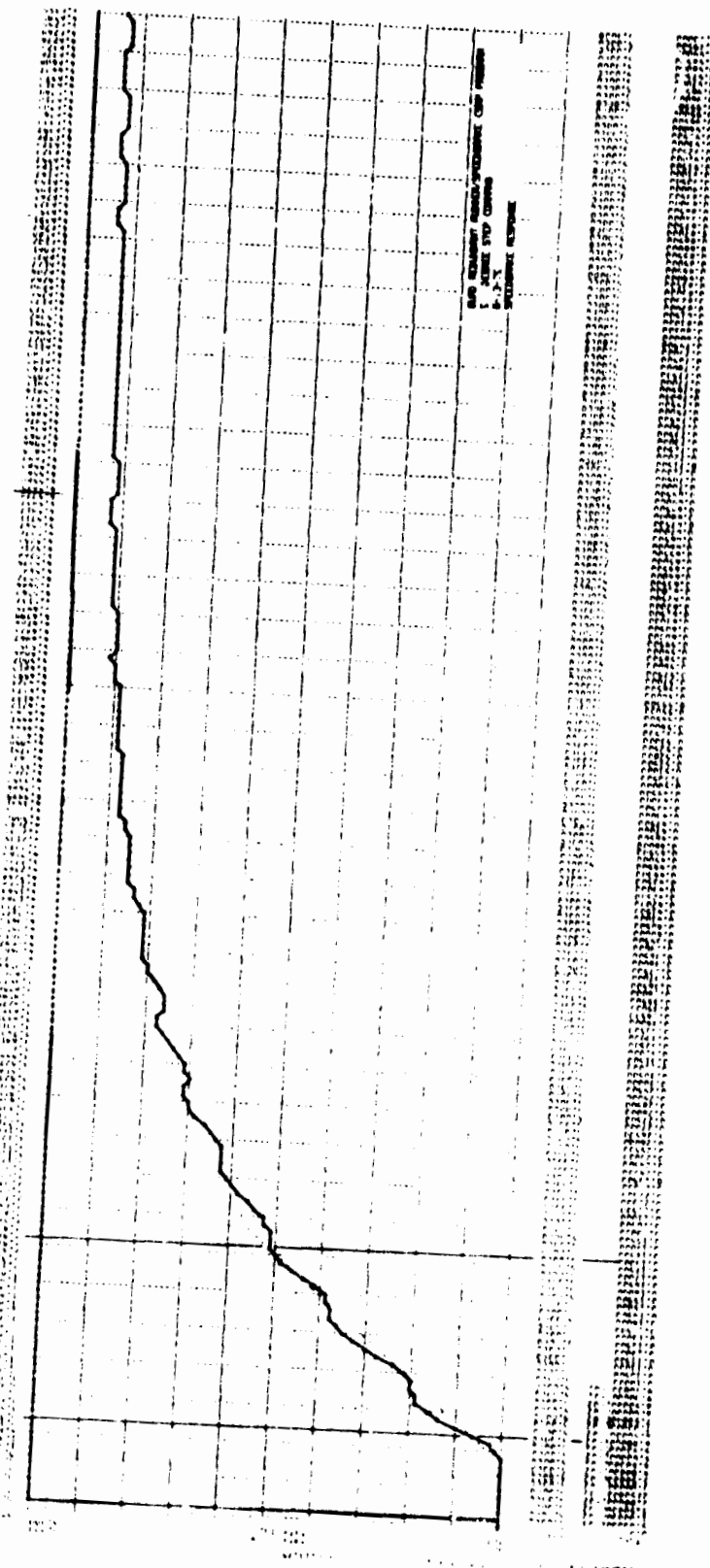


Figure B-4. - 5-degree Speedbrake step command response.

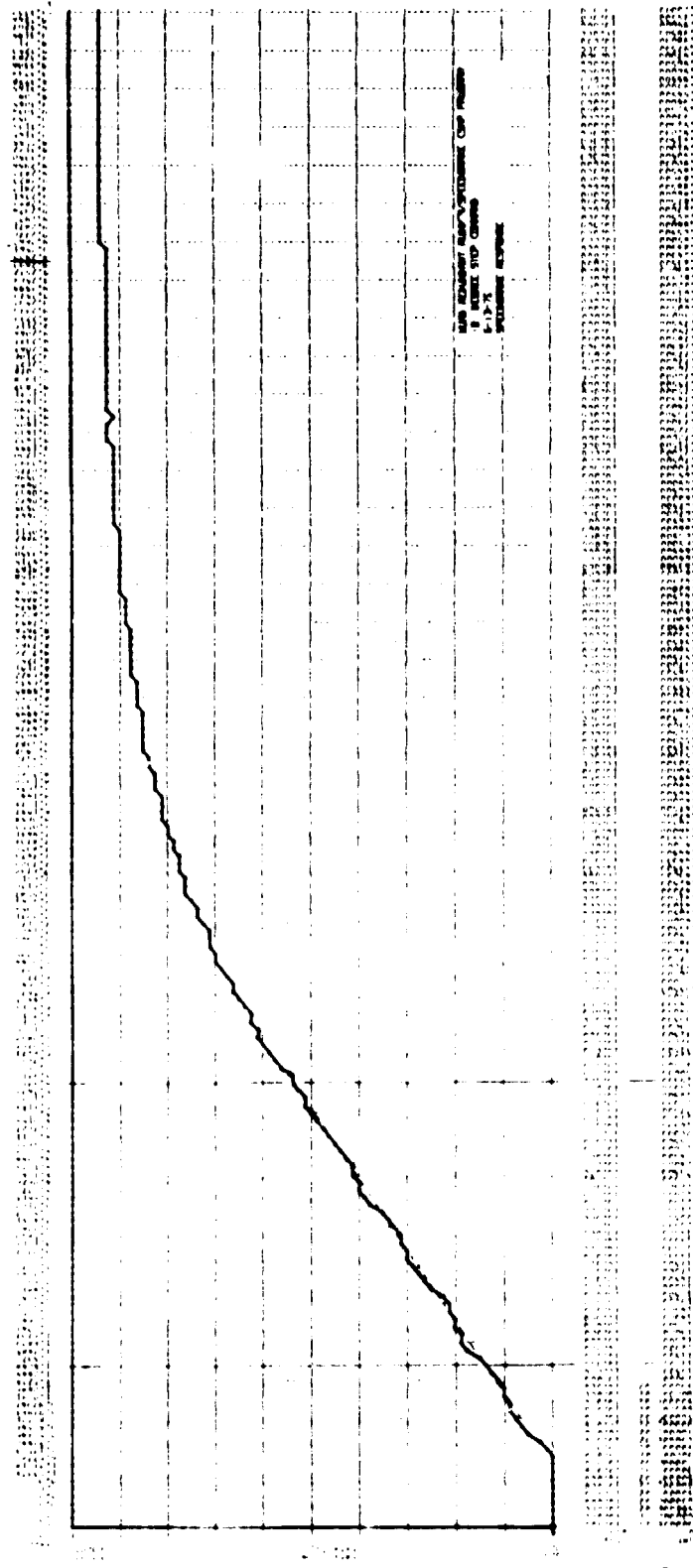


Figure B-5. - 10-degree Speedbrake step command response.

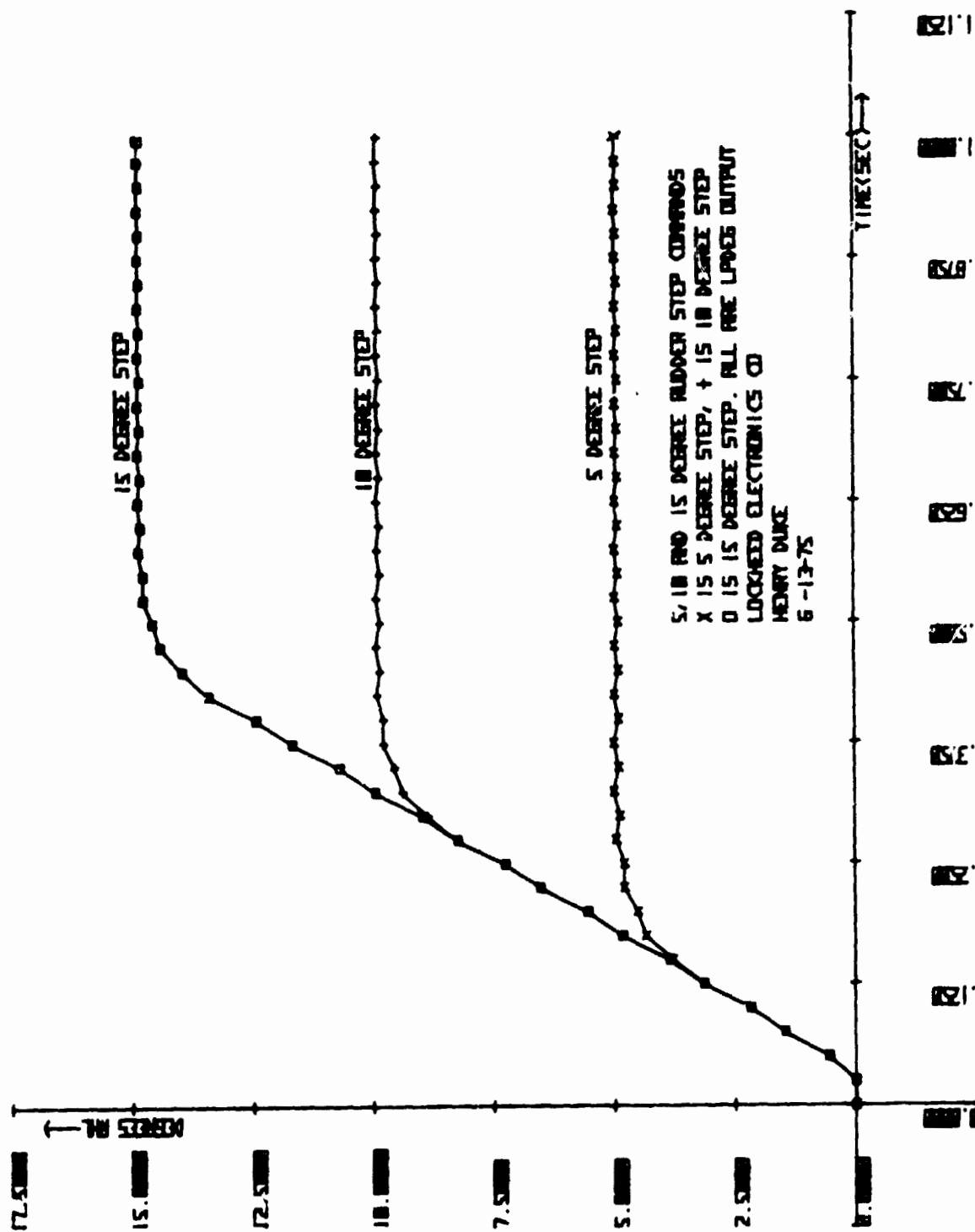
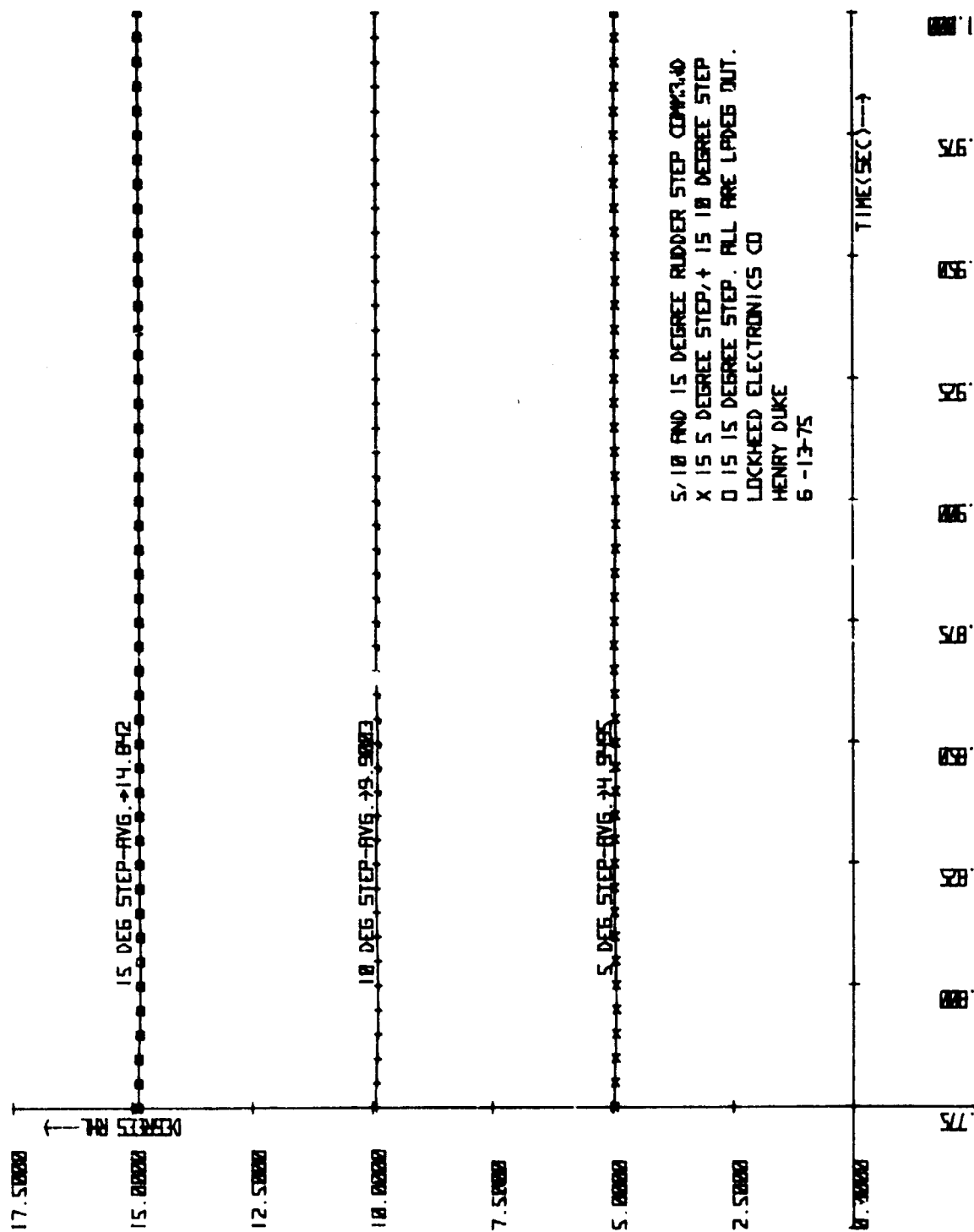


Figure B-7. - Various Rudder step commands.



5, 10 AND 15 DEGREE RUDDER STEP COMMANDS
 X 15 5 DEGREE STEP, + 15 10 DEGREE STEP
 0 15 15 DEGREE STEP. ALL ARE LAPDES OUT.
 LOCKHEED ELECTRONICS CO
 HENRY DUKE
 6-13-75

Figure B-8. - Various Rudder step commands.

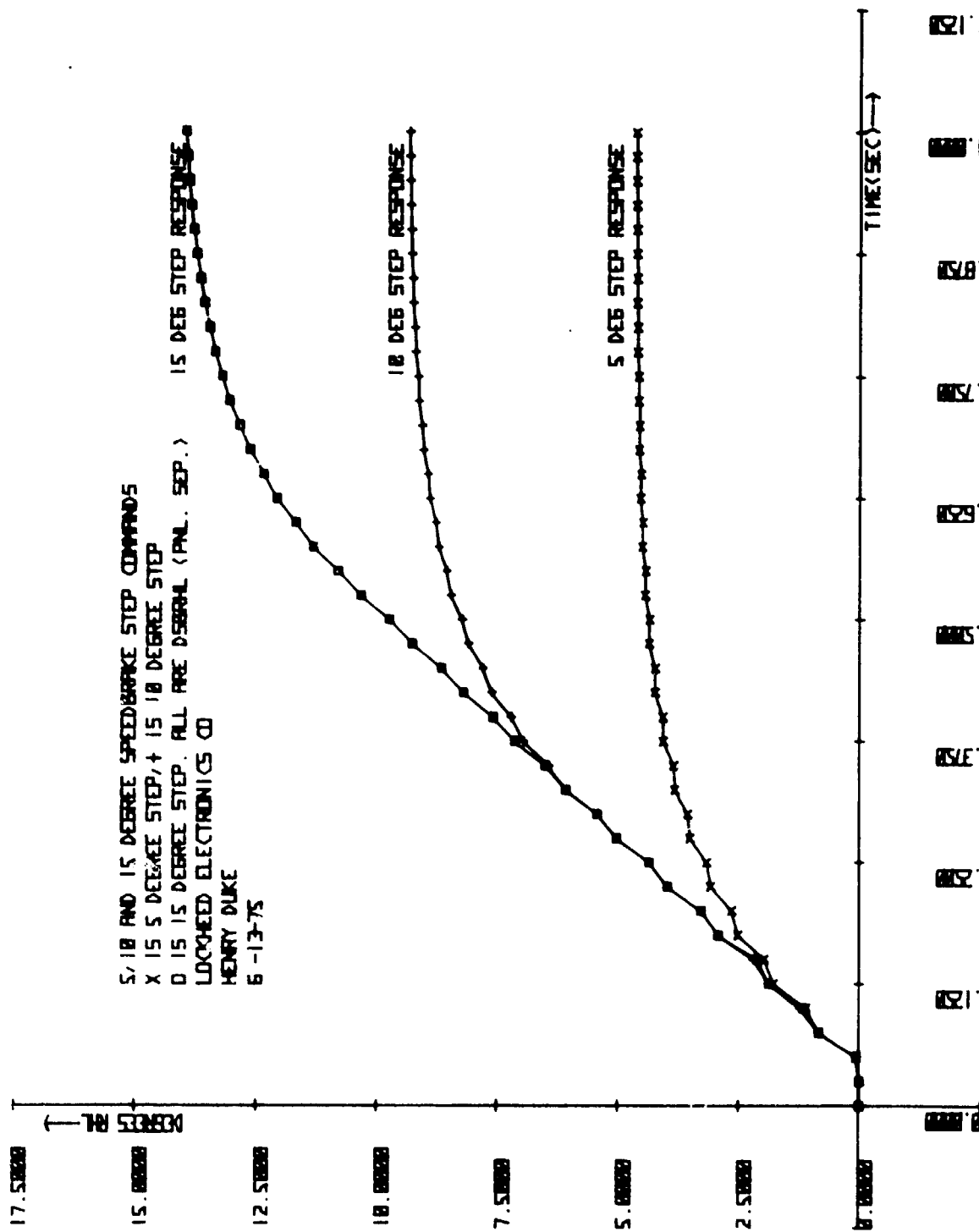
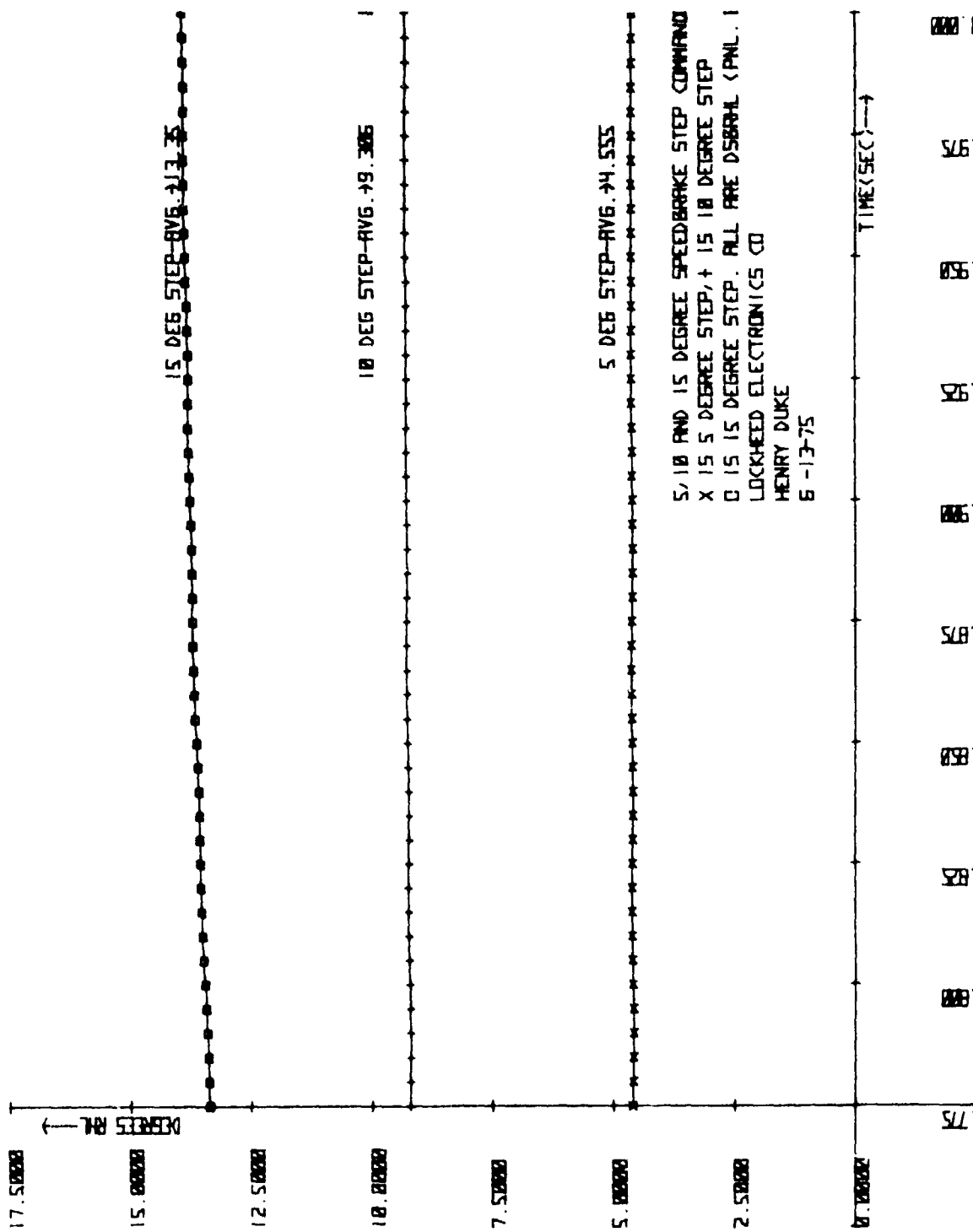


Figure B-9. - Various Speedbrake step commands.



5, 10 AND 15 DEGREE SPEEDBRAKE STEP COMMAND
X 15 5 DEGREE STEP, + 15 10 DEGREE STEP
O 15 15 DEGREE STEP. ALL ARE DSBRAL (PML. 1
LOCKHEED ELECTRONICS CO
HENRY DUKE
6-13-75

Figure B-10. - Various Speedbrake step commands.

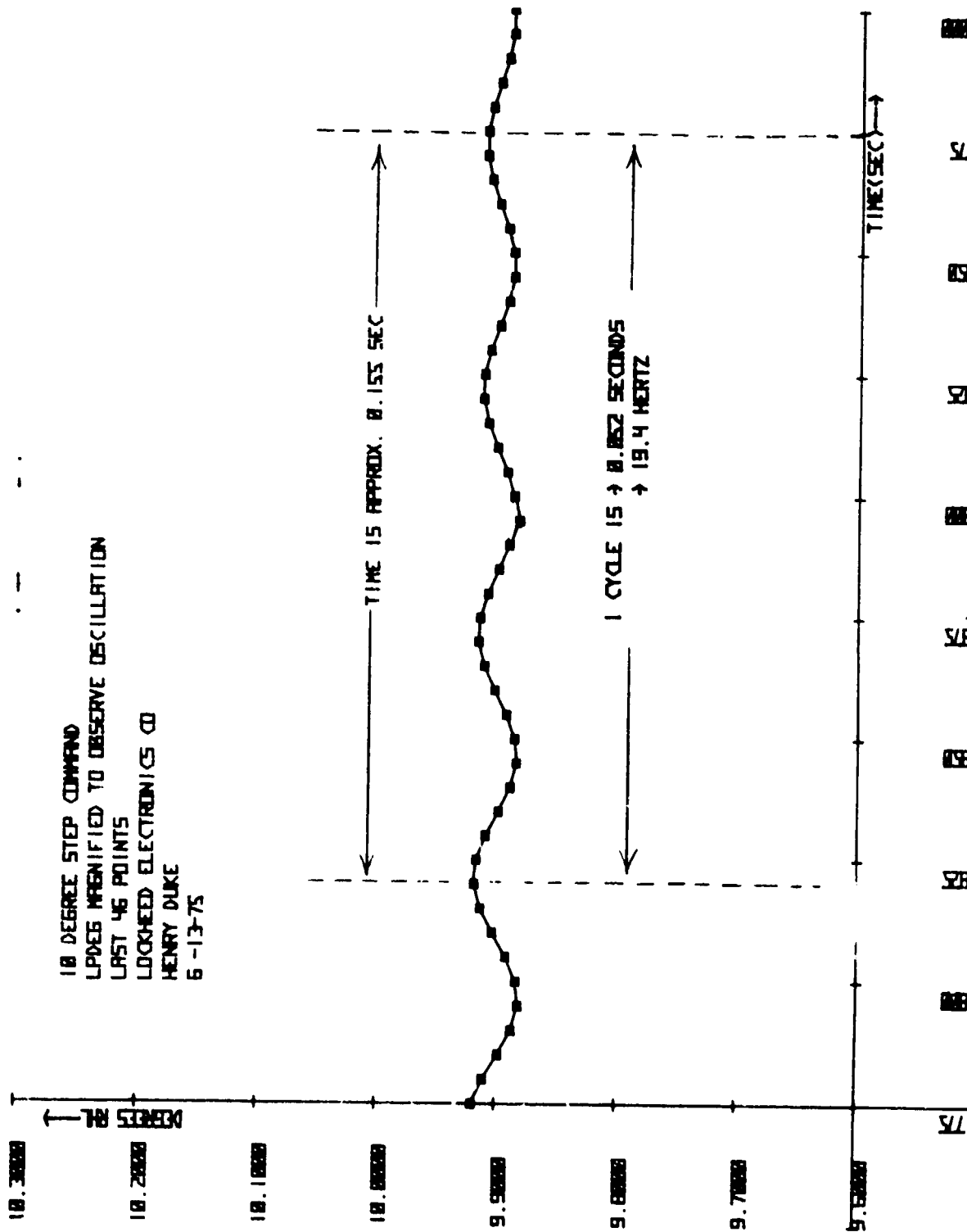


Figure B-11. - LPDEG magnified to observe oscillation.

APPENDIX C

INITIAL STARTING POINTS
(RESPONSE DELAY)
TO A STEP COMMAND

APPENDIX C FIGURES

Figure	Page
C-1 Initial starting point, 5-deg step	C-3
C-2 Initial starting point, 10-deg step.	C-4
C-3 Initial starting point, 15-deg step.	C-5
C-4 Initial starting point, 5-deg step	C-6
C-5 Initial starting point, 10-deg step.	C-7
C-6 Initial starting point, 15-deg step.	C-8
C-7 Initial starting point, 5-deg step	C-9

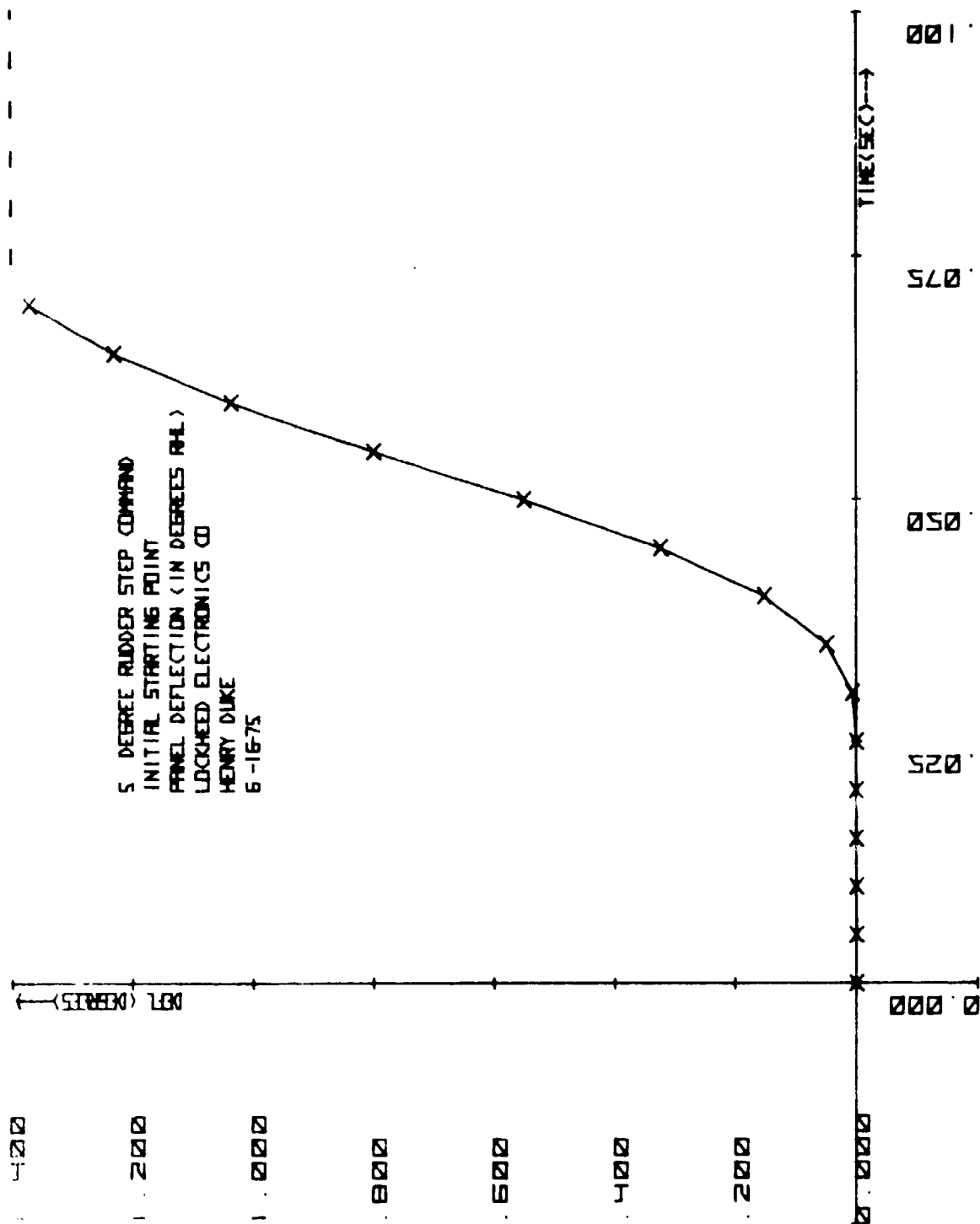


Figure C-1. - Initial starting point, 5-deg step.

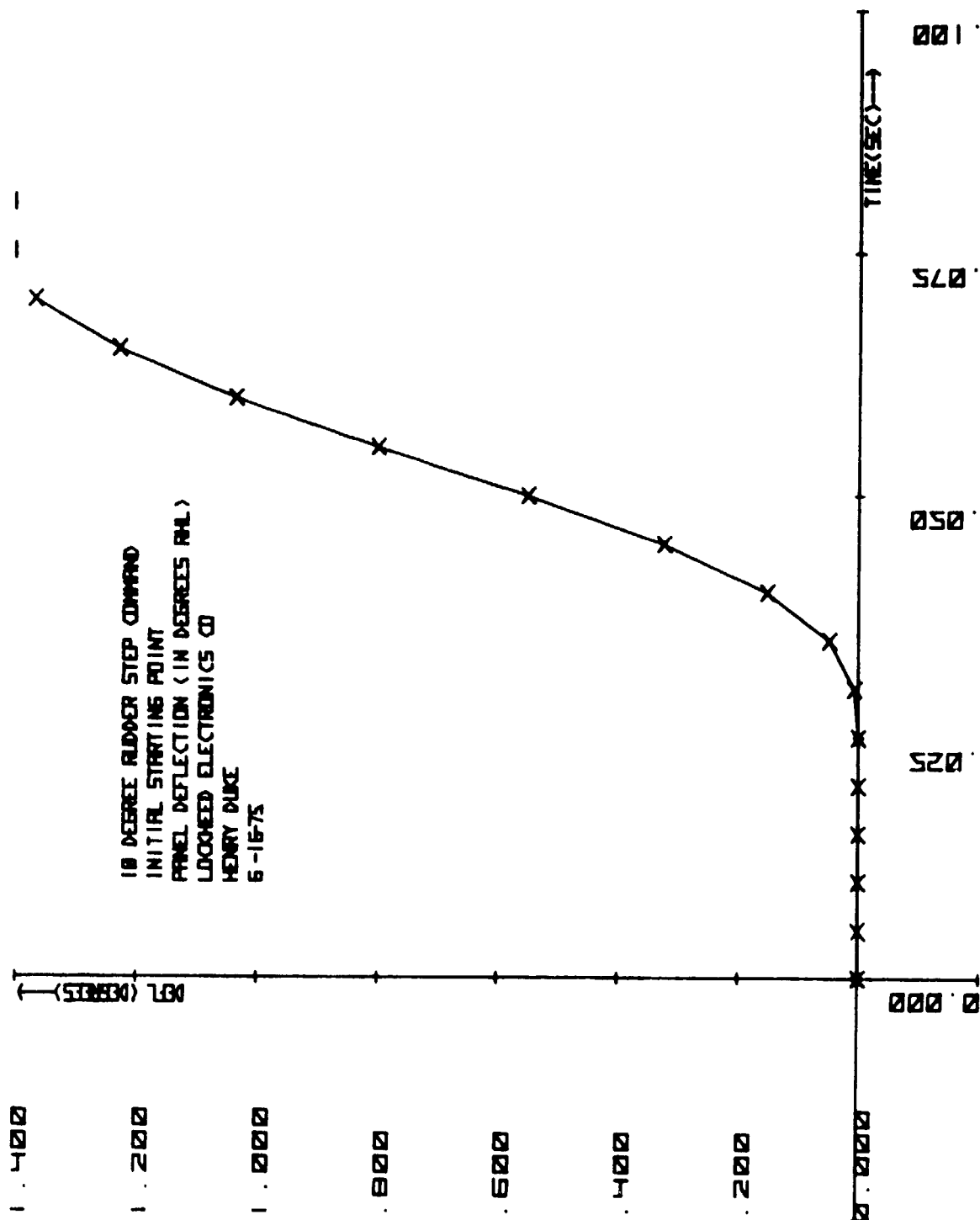


Figure C-2. - Initial starting point, 10-deg step.

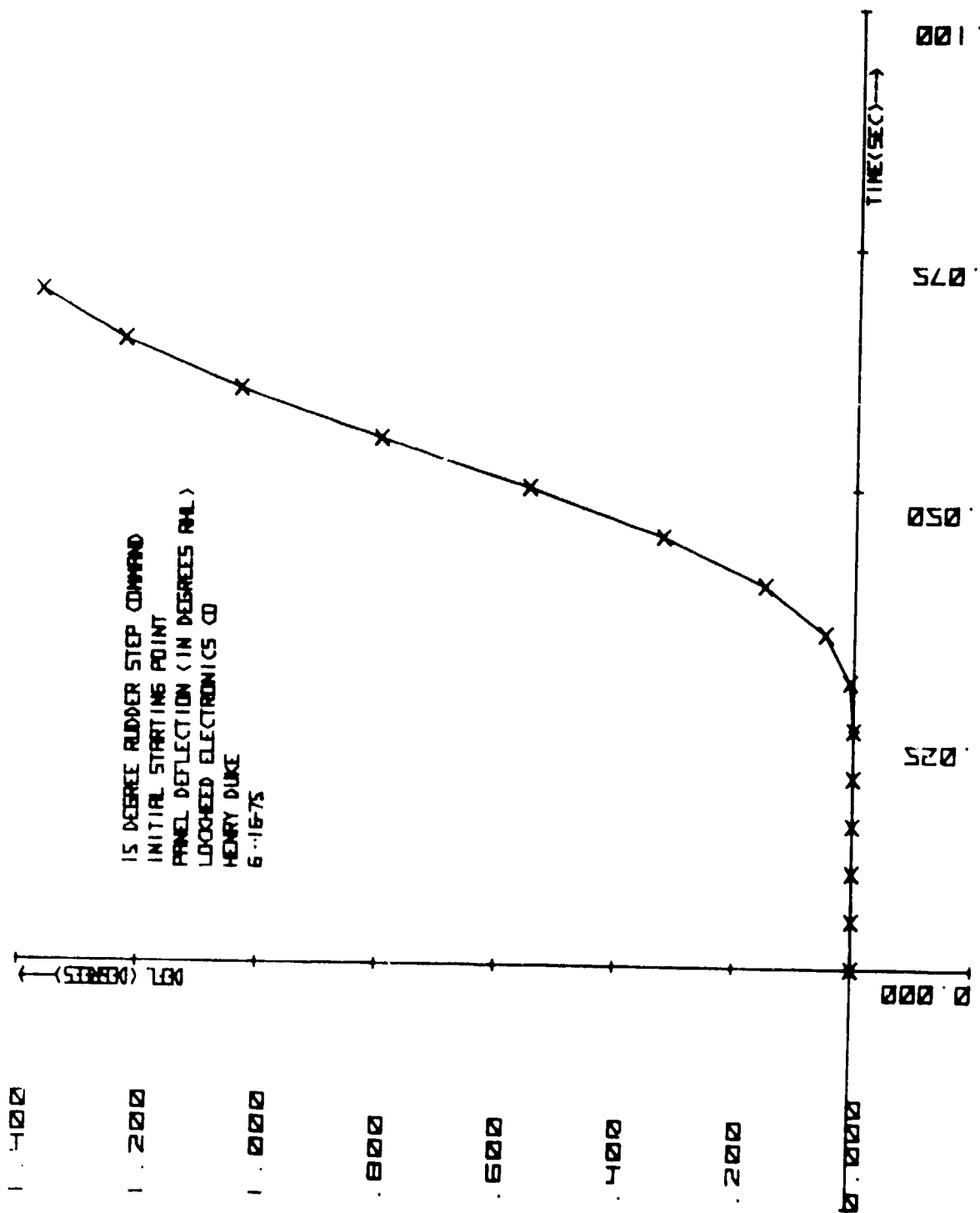


Figure C-3. - Initial starting point, 15-deg step.

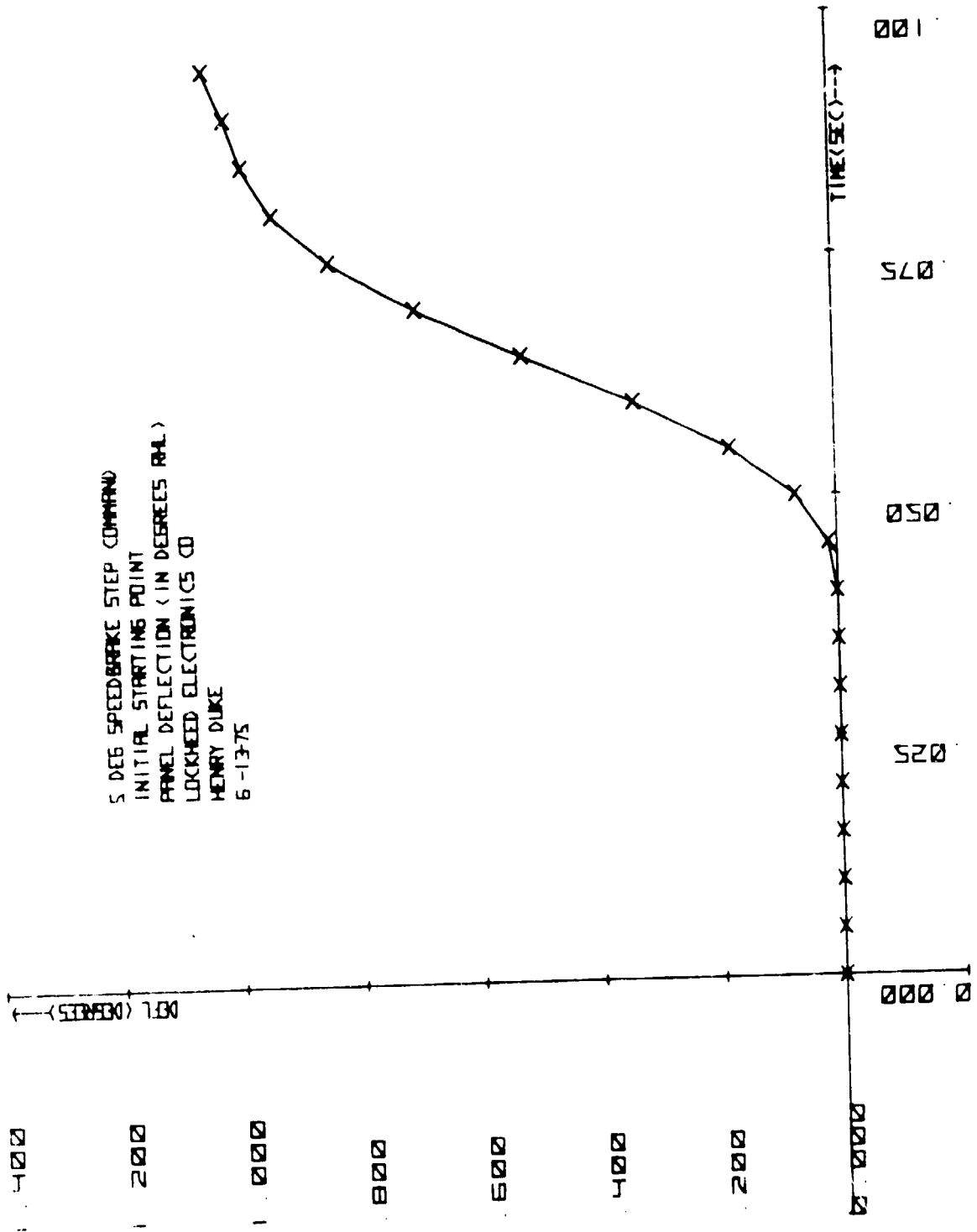


Figure C-4. - Initial starting point, 5-deg step.

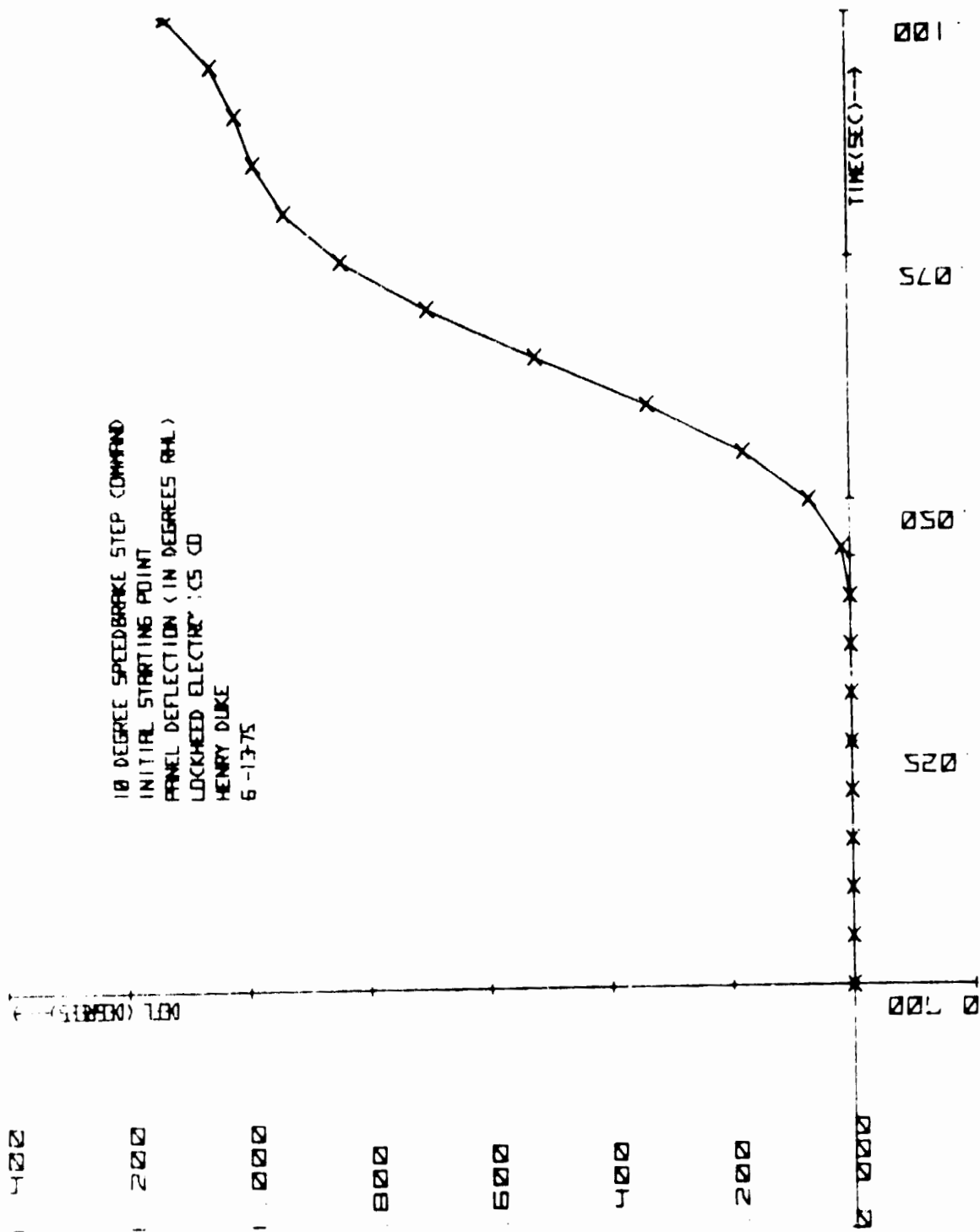


Figure C-5. - Initial starting point, 10-deg step.

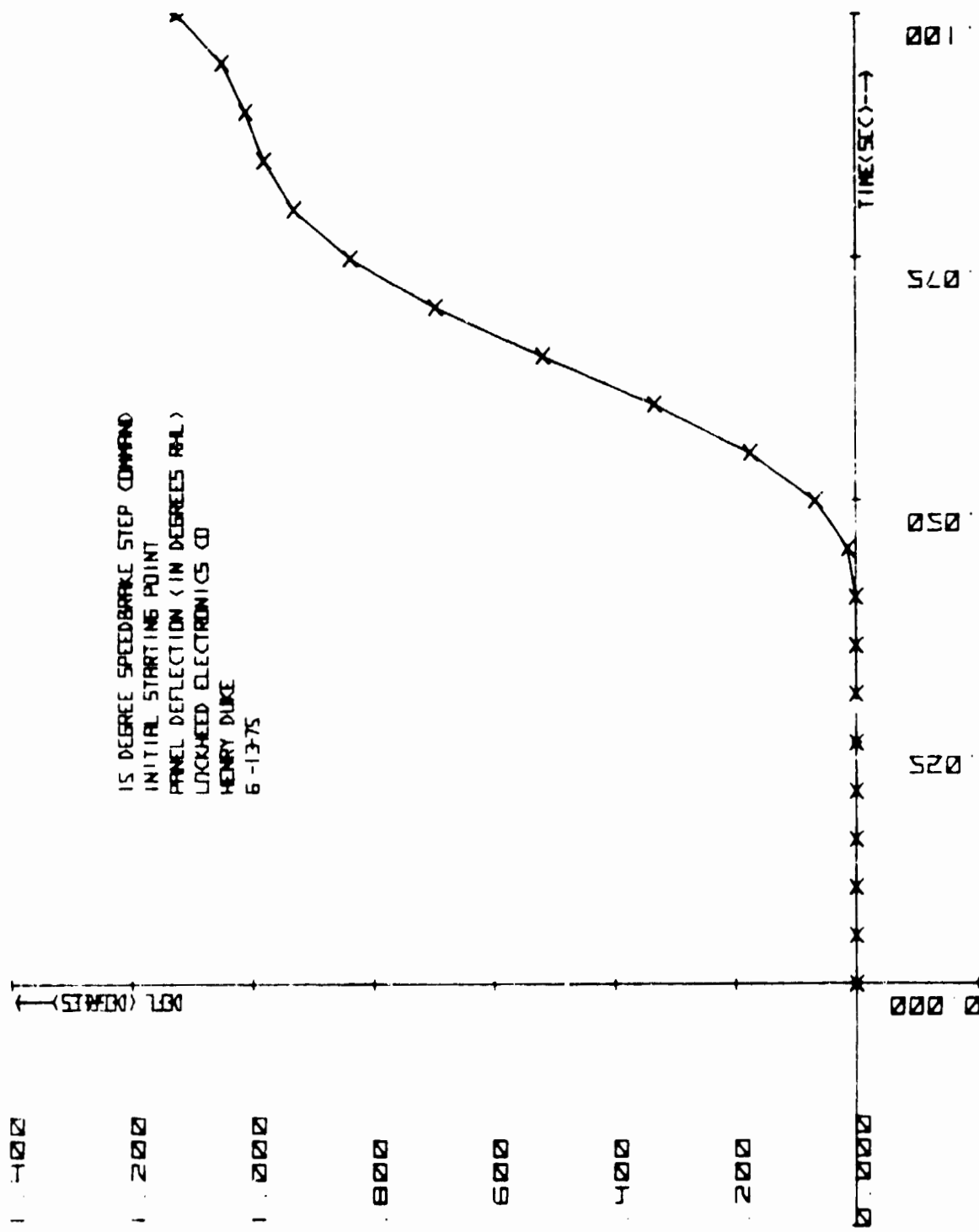


Figure C-6. - Initial starting point, 15-deg step.

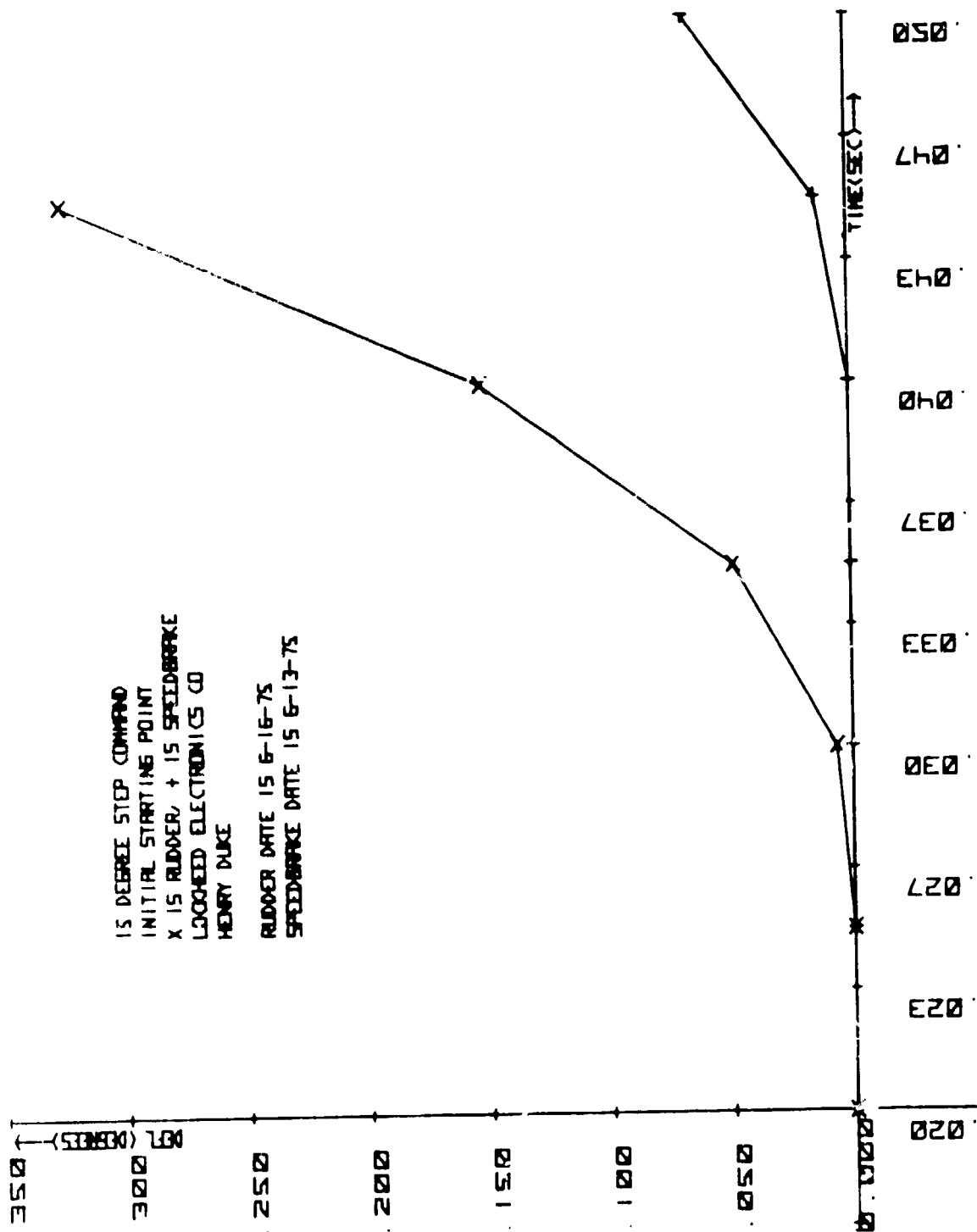


Figure C-7. - Initial starting point, 5-deg step.

APPENDIX D

RAMP RESPONSE CURVES

APPENDIX D FIGURES

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D-1 Motor speed of a single Rudder hydraulic motor for 10-deg/sec Rudder input	D-4
D-2 Motor speed of a single Rudder hydraulic motor for 10-deg/sec Rudder input	D-5
D-3 Motor speed of a single Rudder hydraulic motor for 10-deg/sec Rudder input	D-6
D-4 Rockwell motor speed of a single hydraulic motor for a 10.03-deg/sec Rudder input.	D-7
D-5 Left panel angular speed for a 10-deg/sec Rudder input.	D-8
D-6 Rockwell panel speed for a 10.03-deg/sec Rudder input.	D-9
D-7 Motor speed of a single Rudder hydraulic motor for a 6-deg/sec Speedbrake command.	D-10
D-8 Motor speed of a single Rudder hydraulic motor for a 6-deg/sec Speedbrake command.	D-11
D-9 Rockwell motor speed of a single hydraulic motor for a 10.03-deg/sec Speedbrake input.	D-12
D-10 Rate of Speedbrake opening for a 6-deg/sec Speedbrake command.	D-13
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D-13 Input to secondary actuator for a 10-deg/sec Rudder input.	D-16
D-14 Output force from a secondary actuator for a 10-deg/sec Rudder input	D-17
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D-19	Input hydraulic flow to a single Rudder hydraulic motor for a 10-deg/sec Rudder input . .	D-22

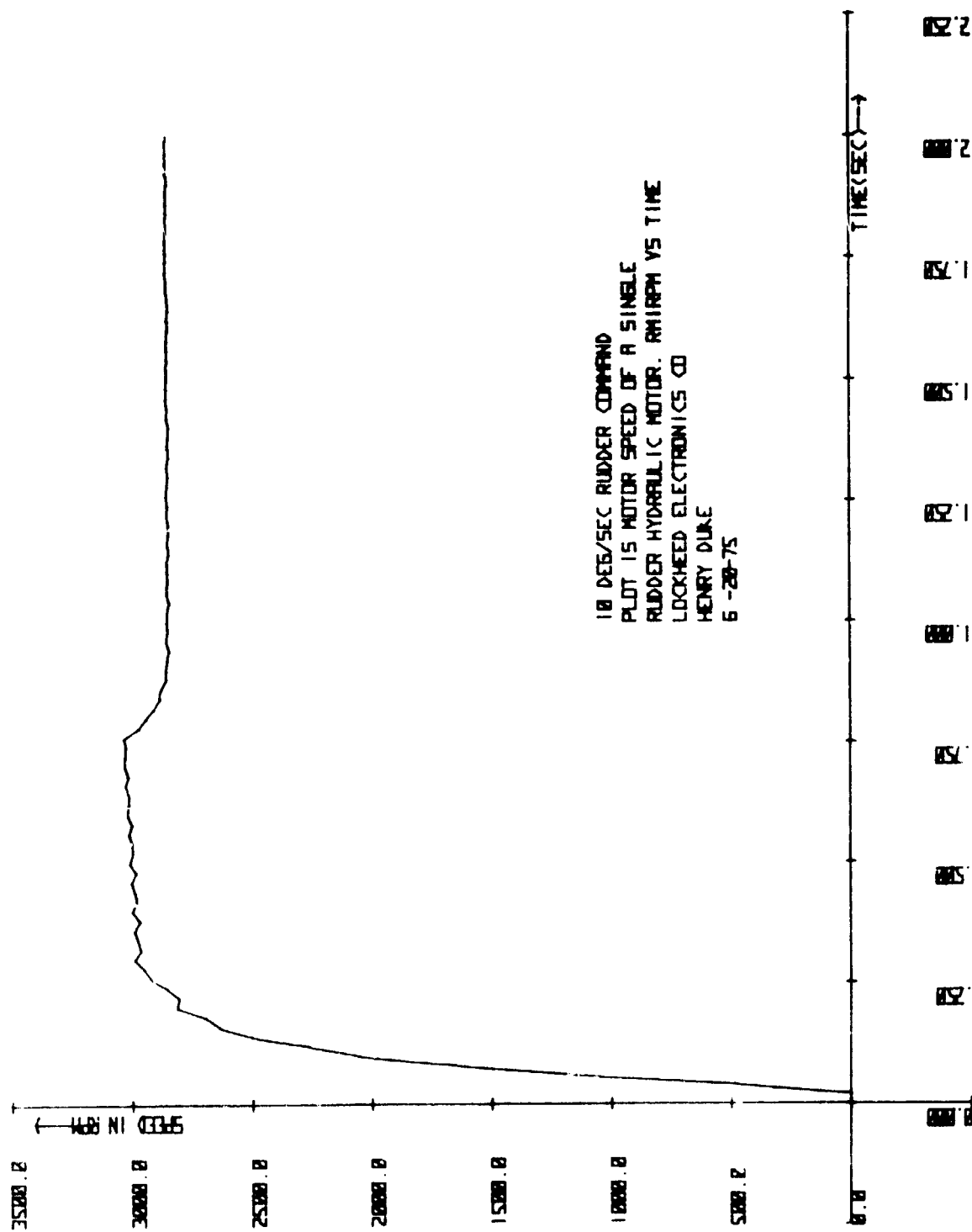


Figure D-1. - Motor speed of a single Rudder hydraulic motor for 10-deg/sec Rudder input.

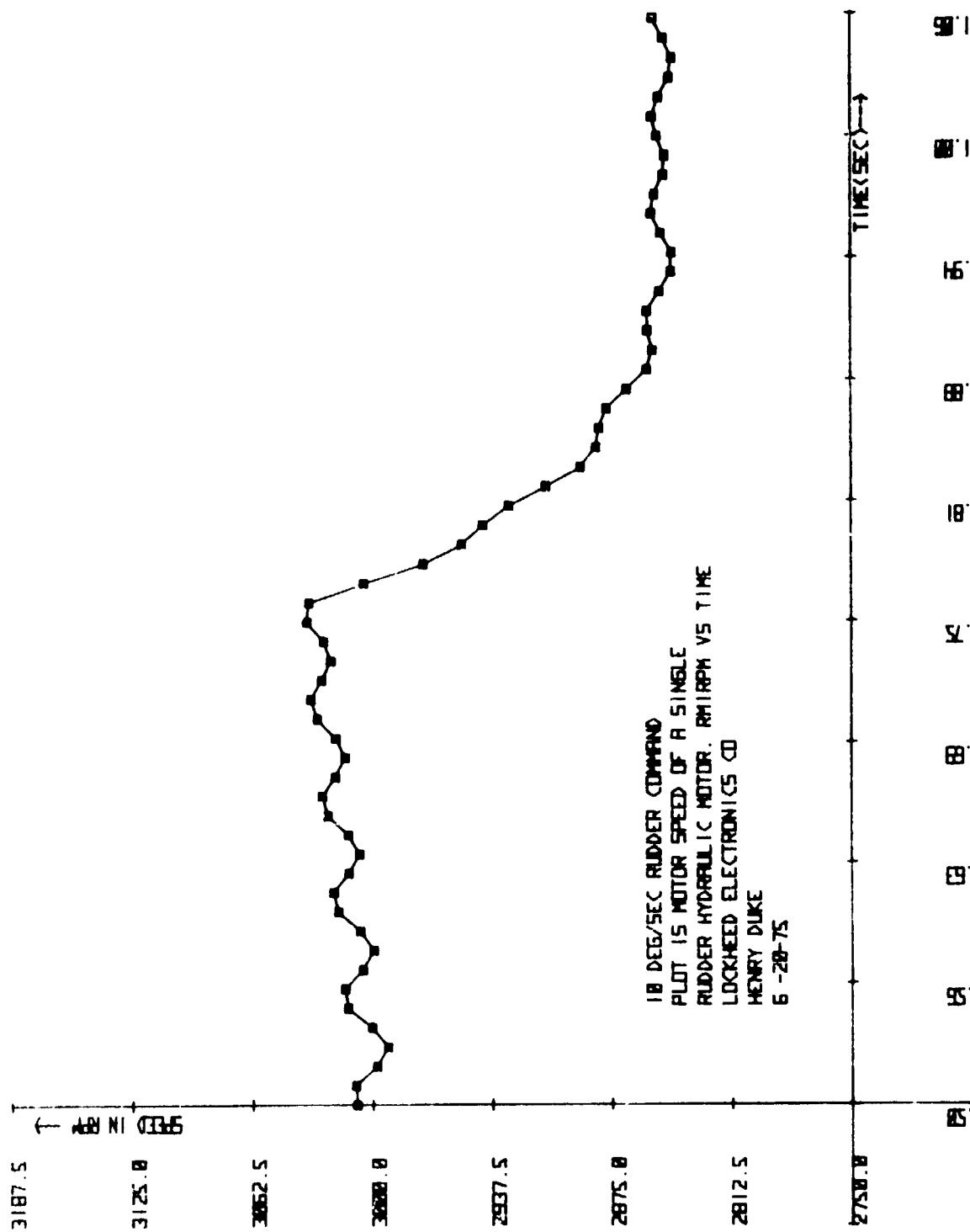


Figure D-2. -- Motor speed of a single Rudder hydraulic motor for 10-deg/sec Rudder input.

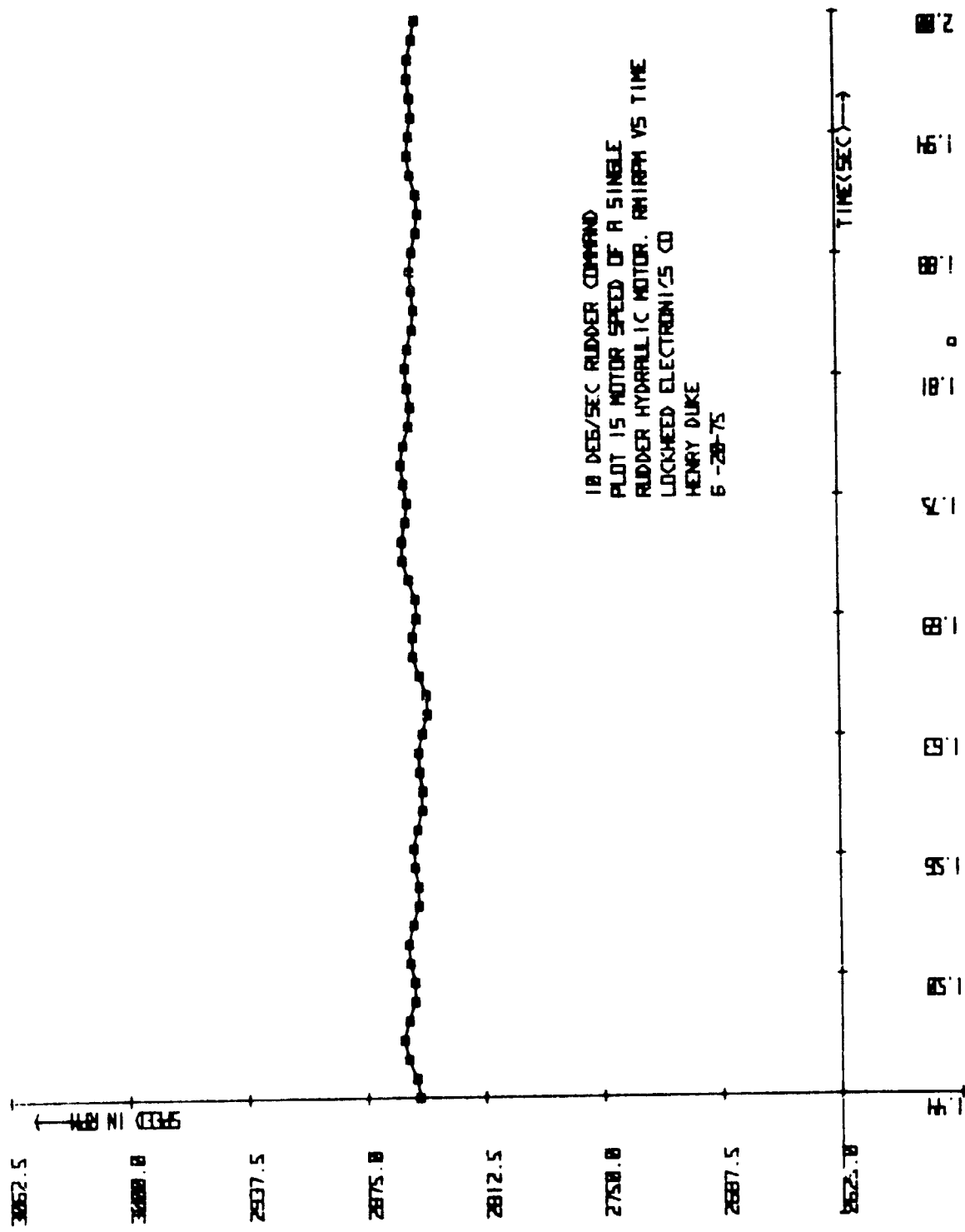
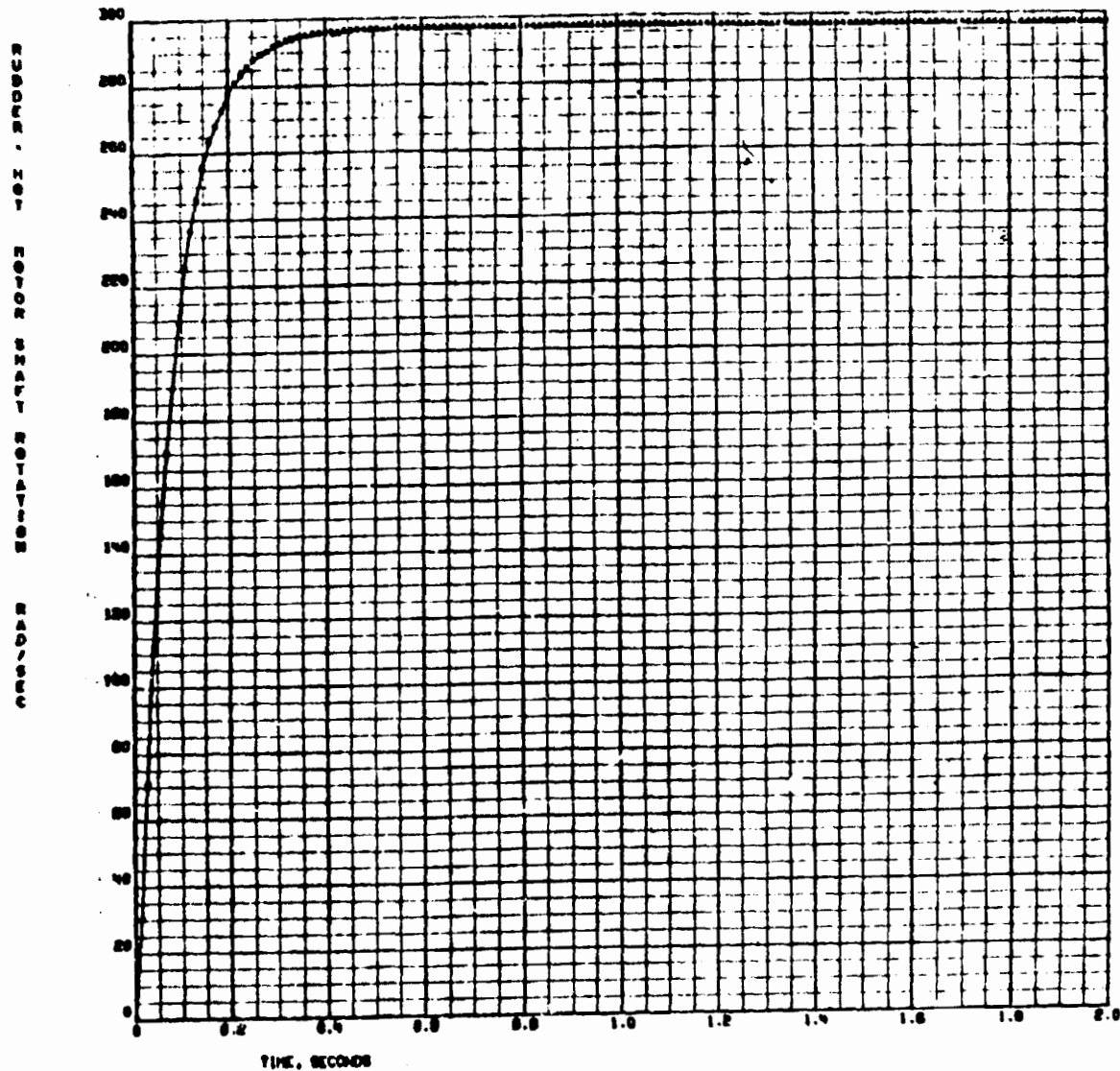


Figure D-3. - Motor speed of a single Rudder hydraulic motor for 10-deg/sec Rudder input.

LINEAR SINGLE STRING SYSTEM- RUDDER, NOT RAMP RESPONSE
 SOLUTION FOR UNKNOWN NO. 8 (DTHED) 04/30/75

04108123
 043075 0004



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Figure D-4. - Rockwell motor speed of a single hydraulic motor for a 10.03-deg/sec Rudder input.

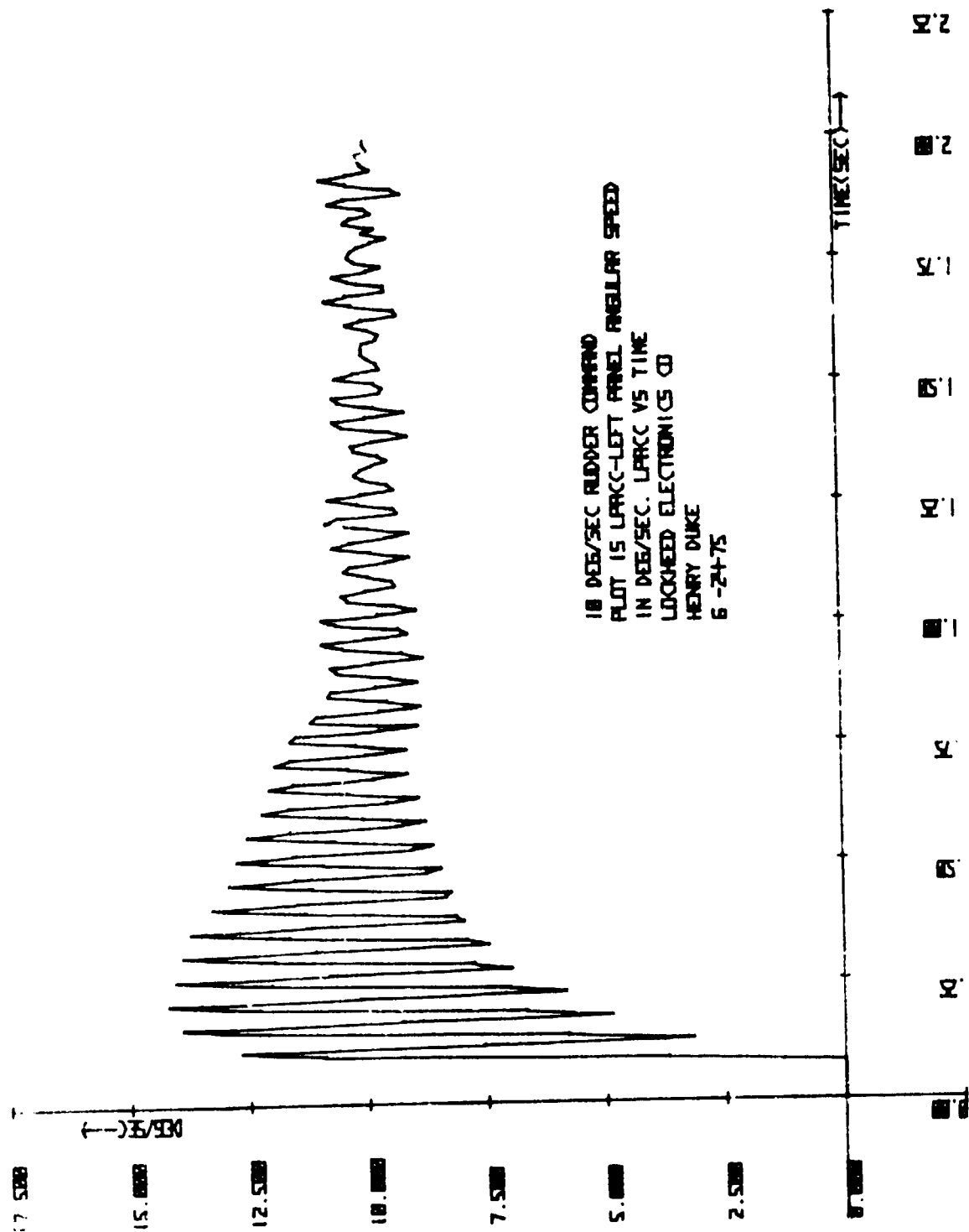
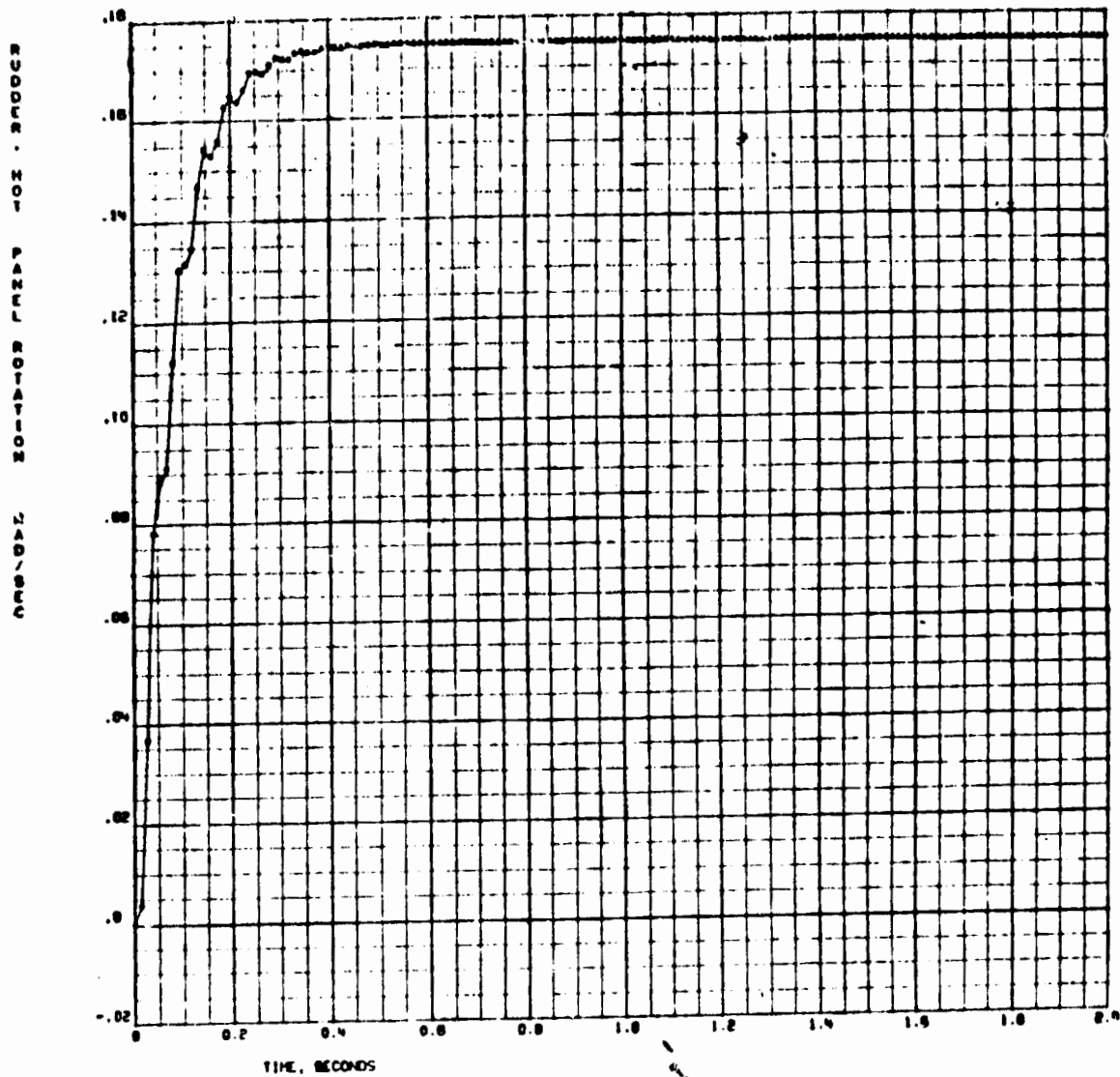


Figure D-5. - Left panel angular speed for a 10-deg/sec Rudder input.

LINEAR SINGLE STRING SYSTEM- RUDDER, NOT RAMP RESPONSE
SOLUTION FOR FREQUENCY NO. 2 (LOADYN) 04/30/75

04/30/75
04/30/75



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GUARANTEED

Figure D-6. - Rockwell panel speed for a 10.03-deg/sec Rudder input.

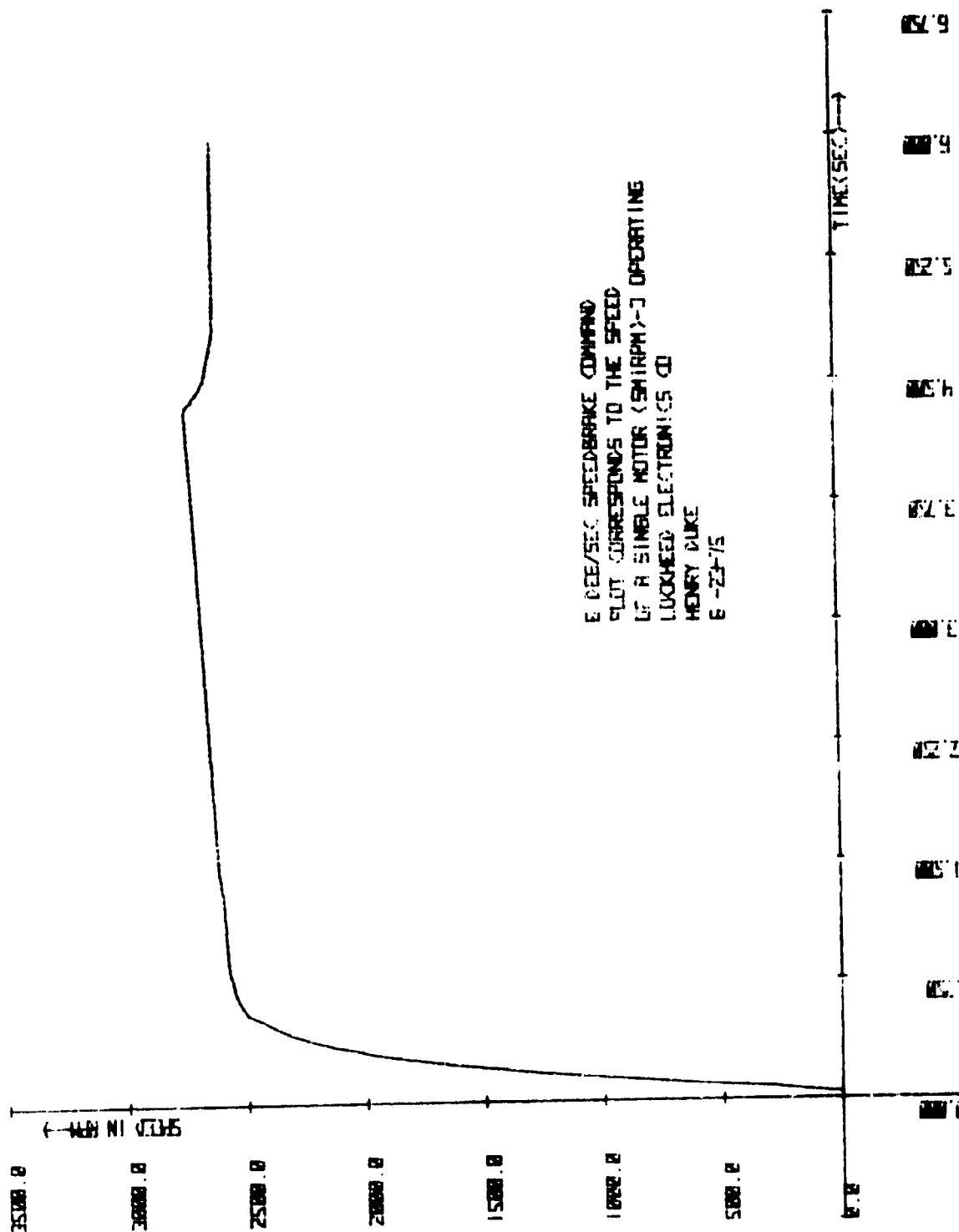


Figure D-7. - Motor speed of a single Rudder hydraulic motor for a 6-deg/sec Speedbrake command.

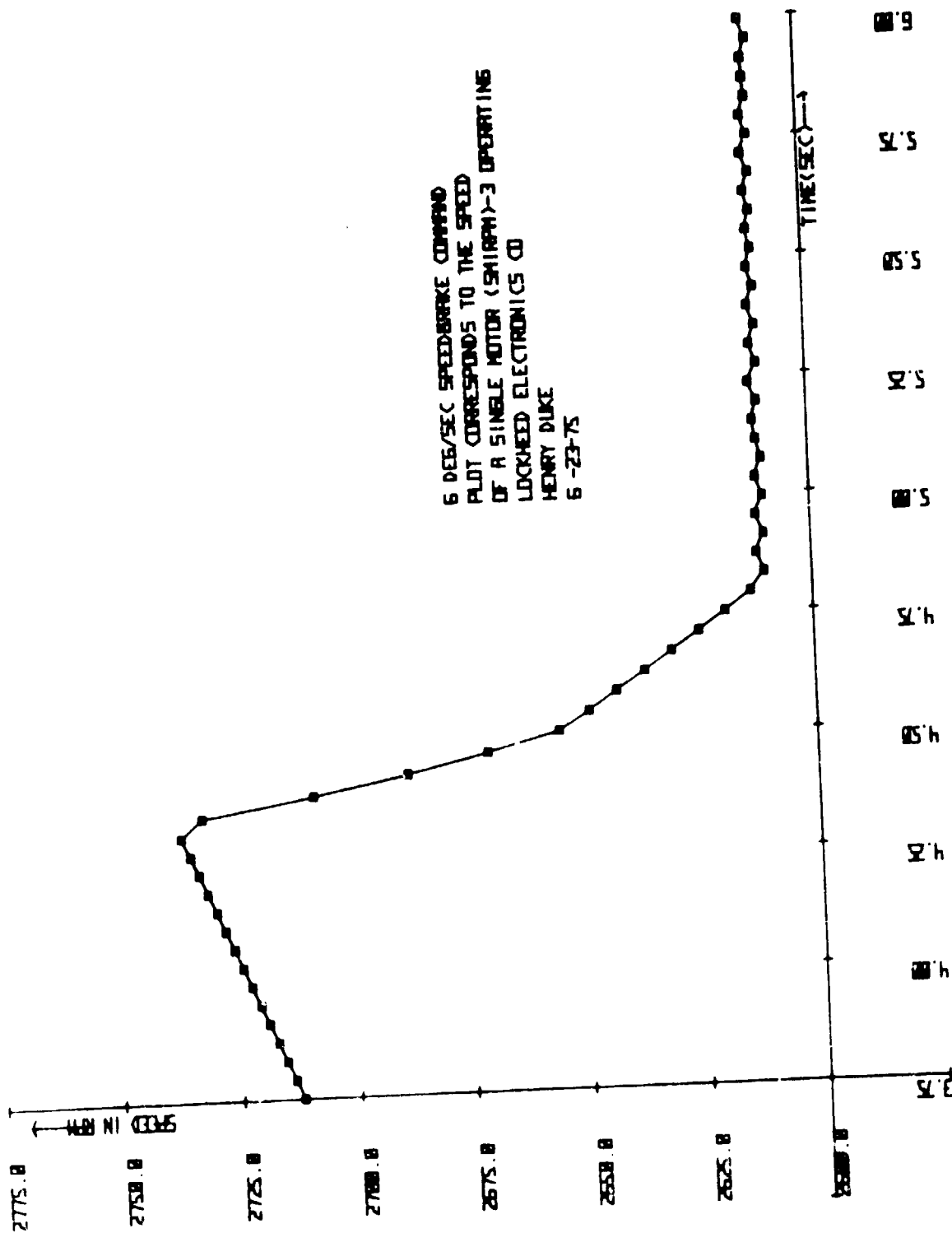


Figure D-8. - Motor speed of a single Rudder hydraulic motor for a 6-deg/sec Speedbrake command.

LINEAR SINGLE STRING SYSTEM- SPEEDBRAKE, HOT RAMP RESPONSE
 SOLUTION FOR UNKNOWN NO. 8 (DTHED) 08/02/78

04100123
 050275 000

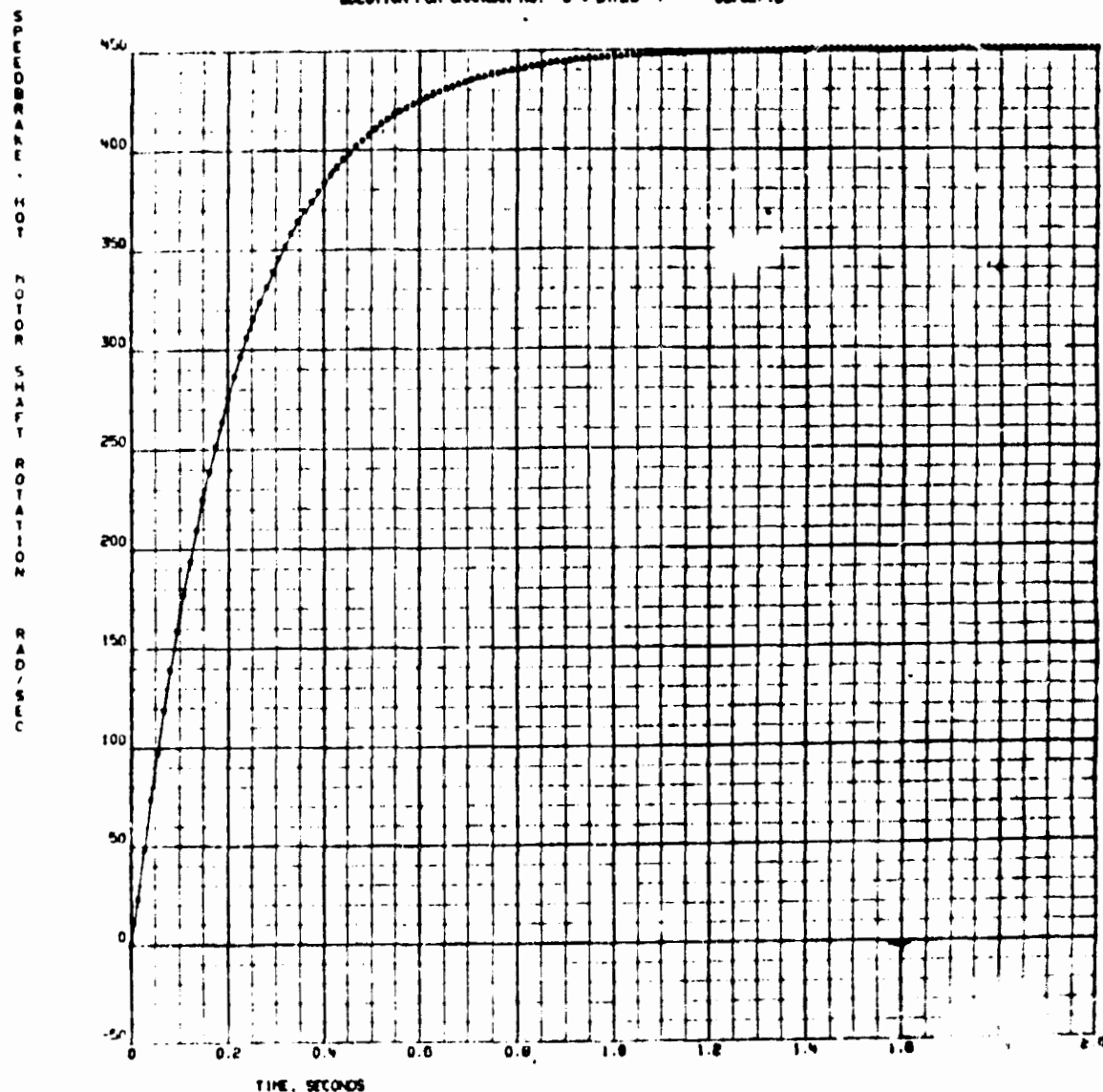


Figure D-9. - Rockwell motor speed of a single hydraulic motor for a 10.03-deg/sec Speedbrake input.

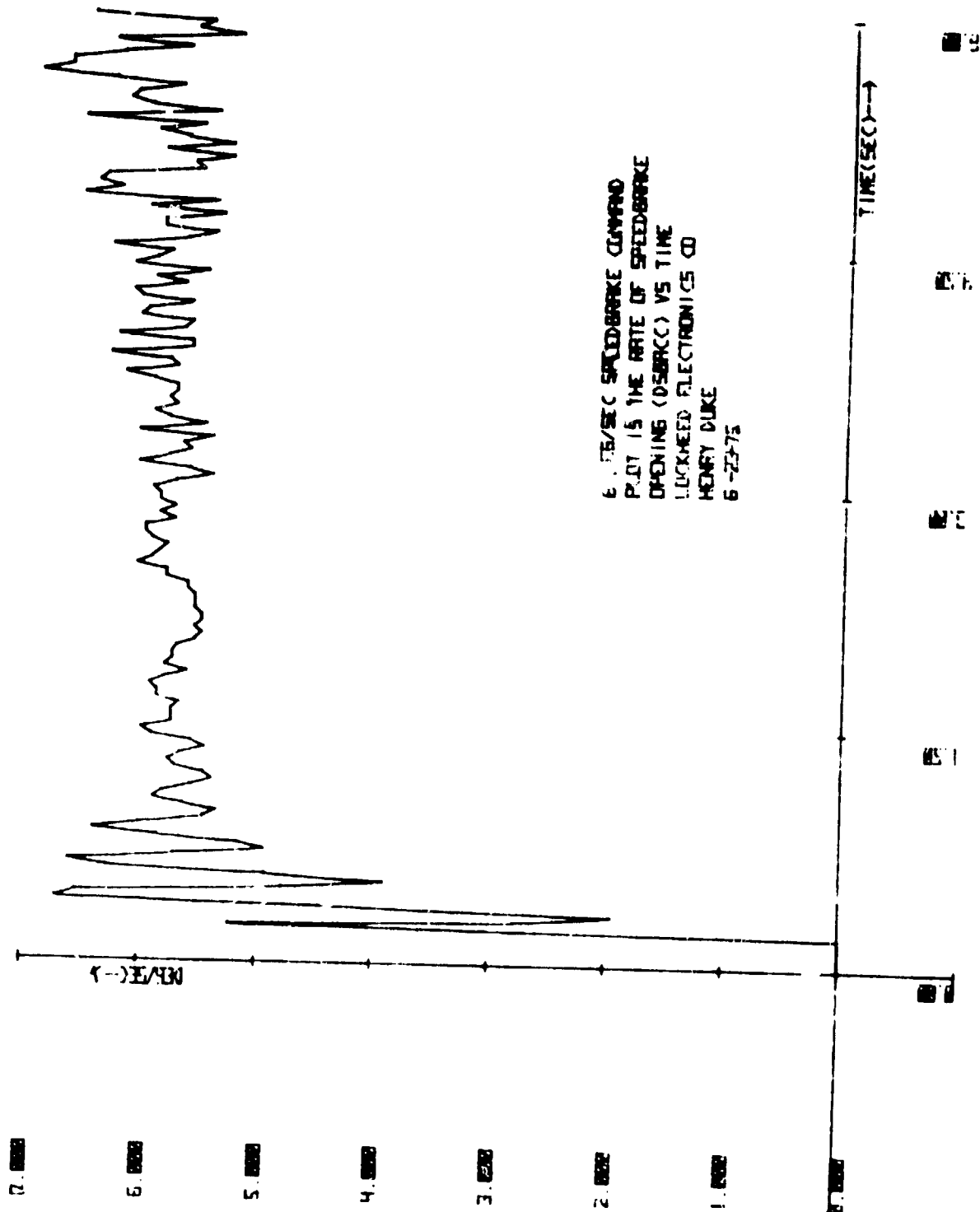


Figure D-10. - Rate of Speedbrake opening for a 6-deg/sec Speedbrake command.

LINEAR SINE SINE SYSTEM - SPEEDBRAKE, NOT RAMP RESPONSE
 SOLUTION FOR UNKNOWN NO. 2 (LOAD IN) 08/02/75

08/02/75
 08/02/75

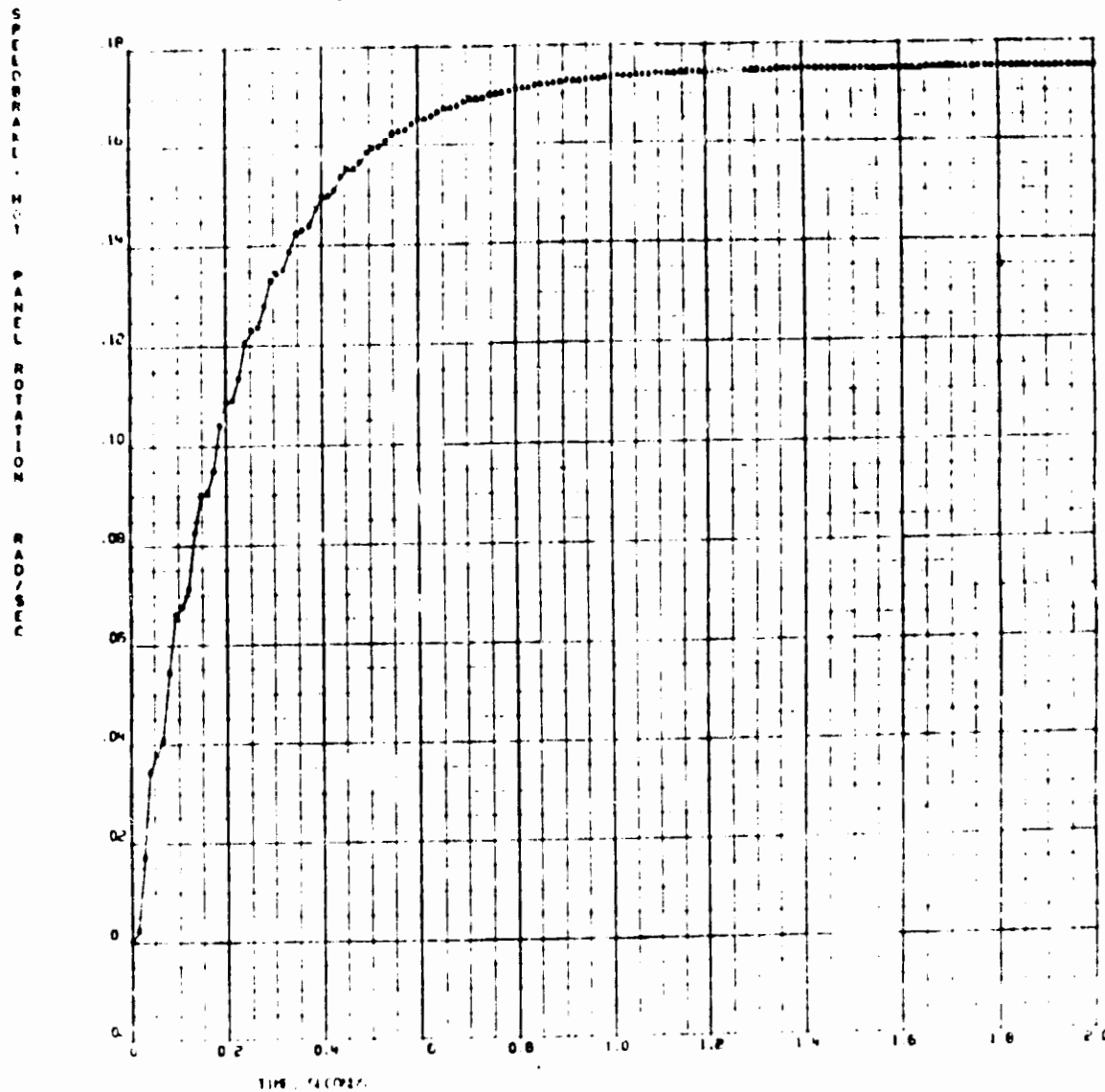


Figure D-11. - Rockwell panel opening for a 10.03-deg/sec Speedbrake input.

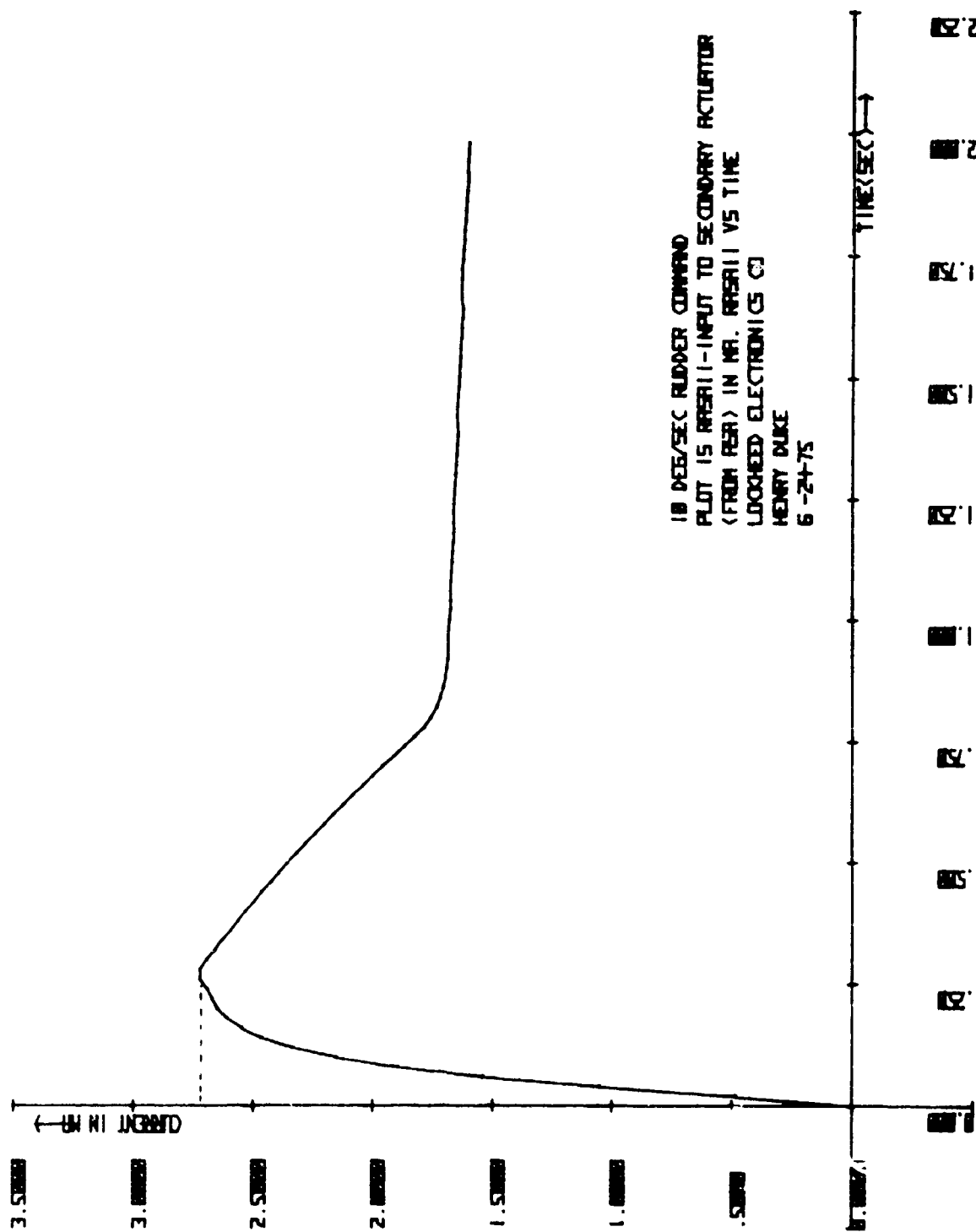


Figure D-12. - Input to secondary actuator for a 10-deg/sec Rudder input.

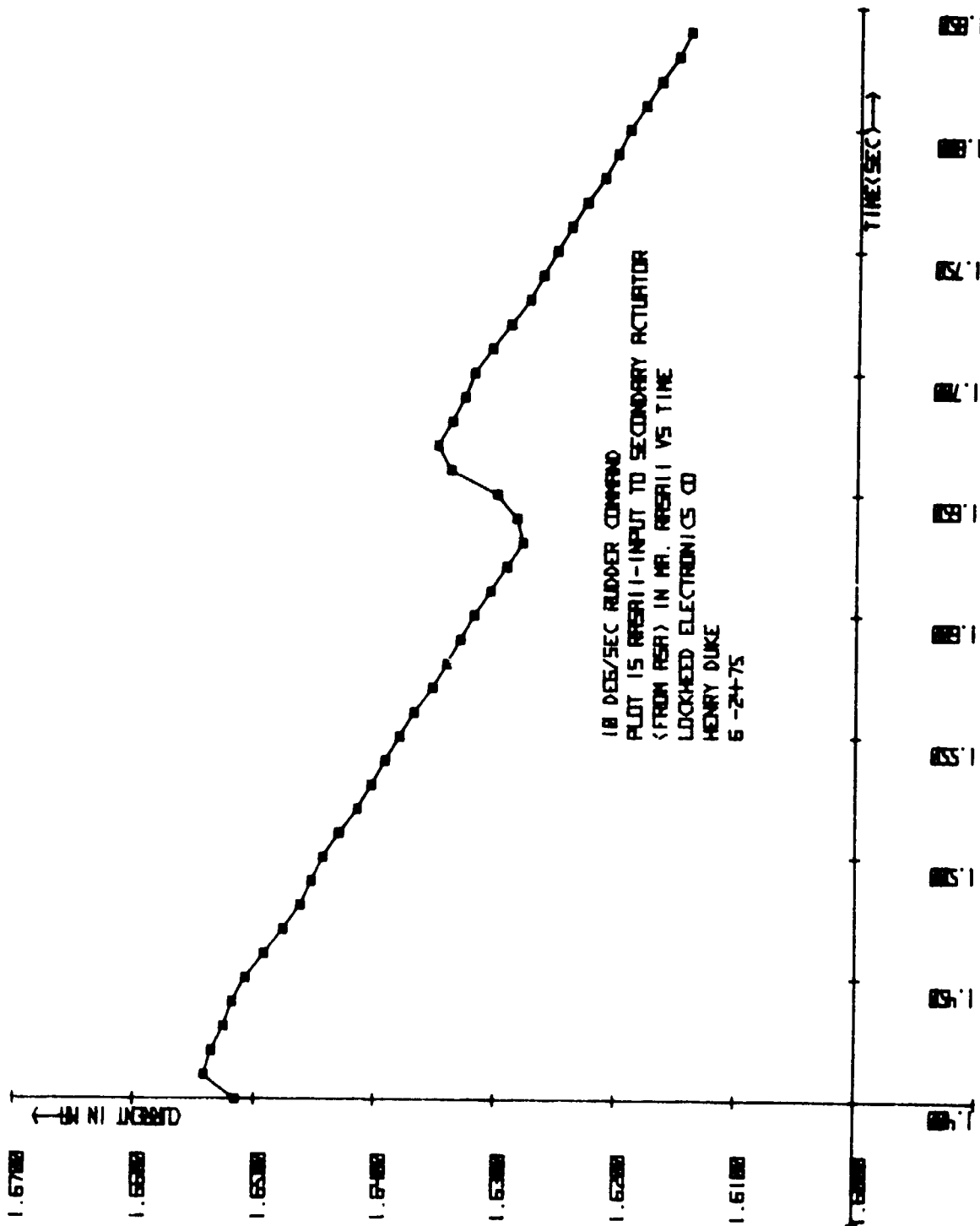


Figure D-13. - Input to secondary actuator for a 10-deg/sec Rudder input.

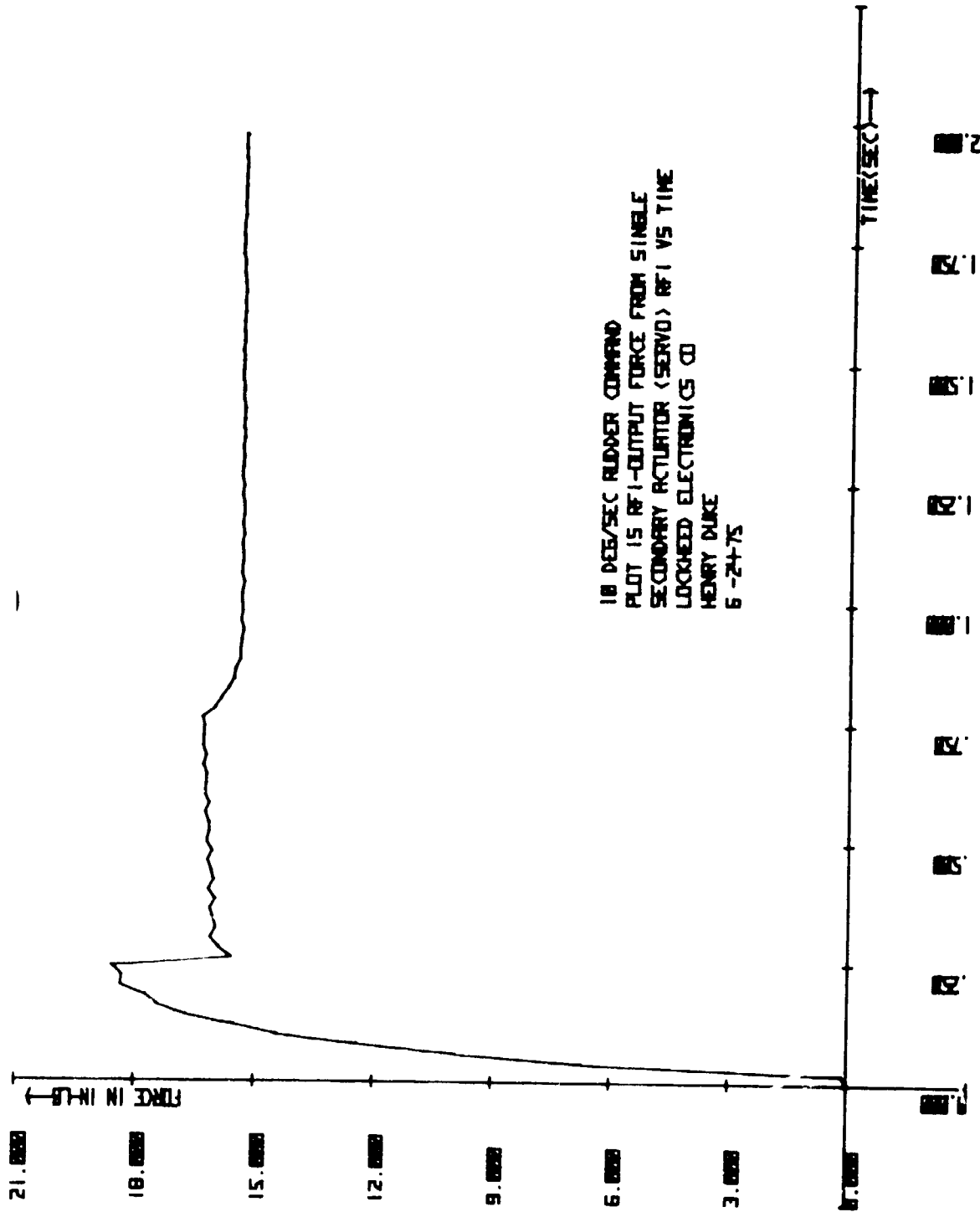


Figure D-14. - Output force from a secondary actuator for a 10-deg/sec Rudder input.

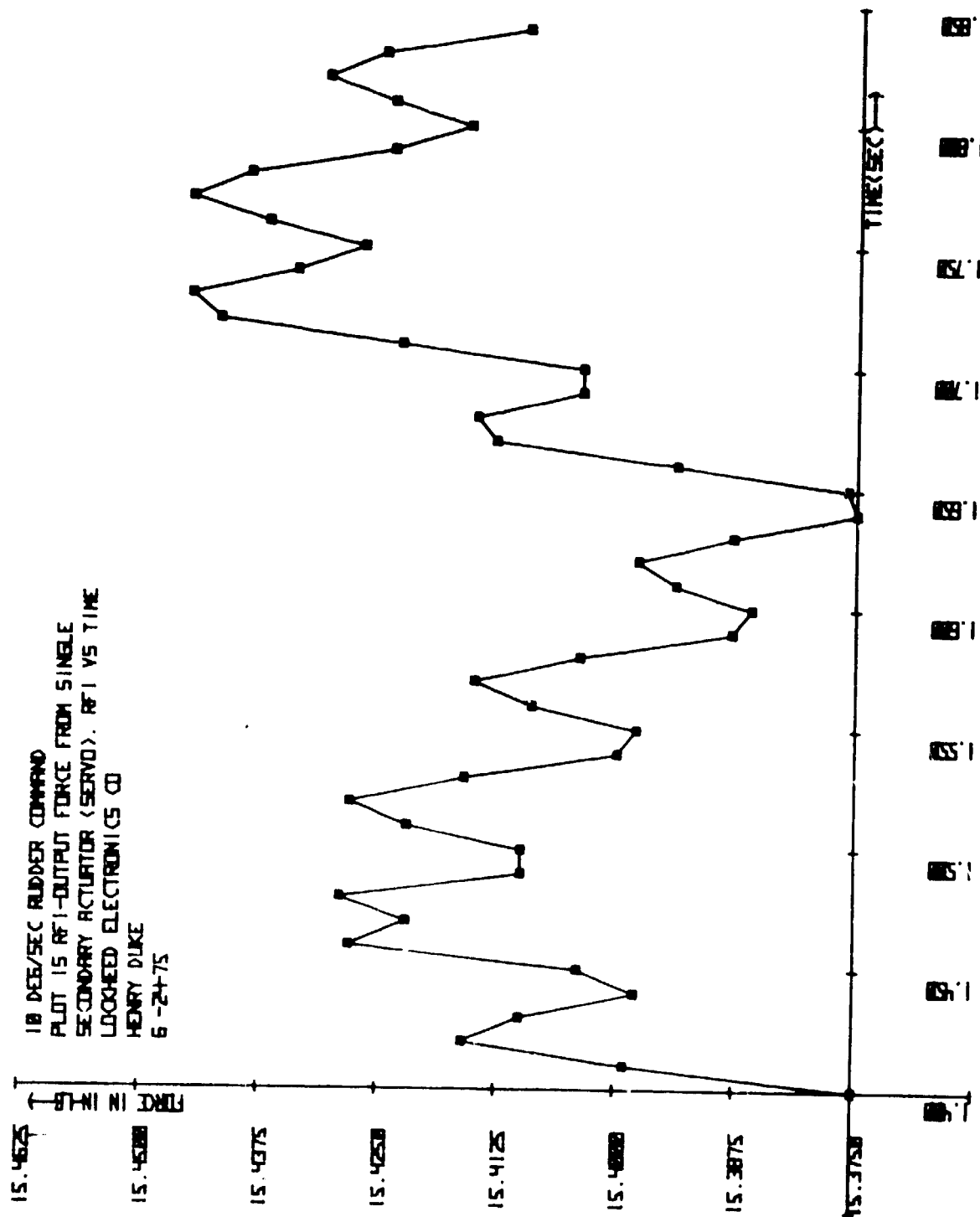


Figure D-15. - Output force from a secondary actuator for a 10-deg/sec Rudder input.

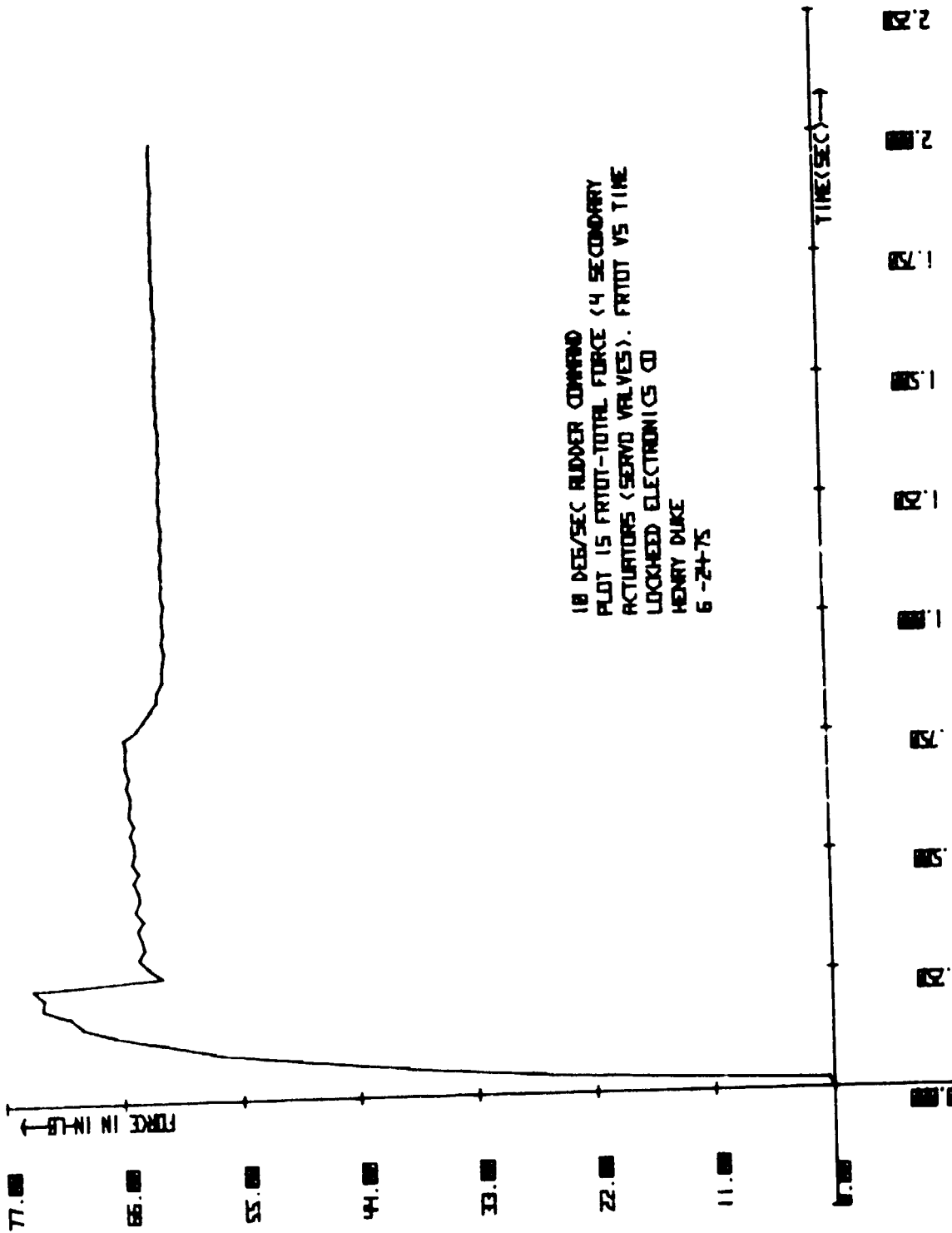


Figure D-16. - Total force output of 4 secondary actuators for a 10-deg/sec Rudder input.

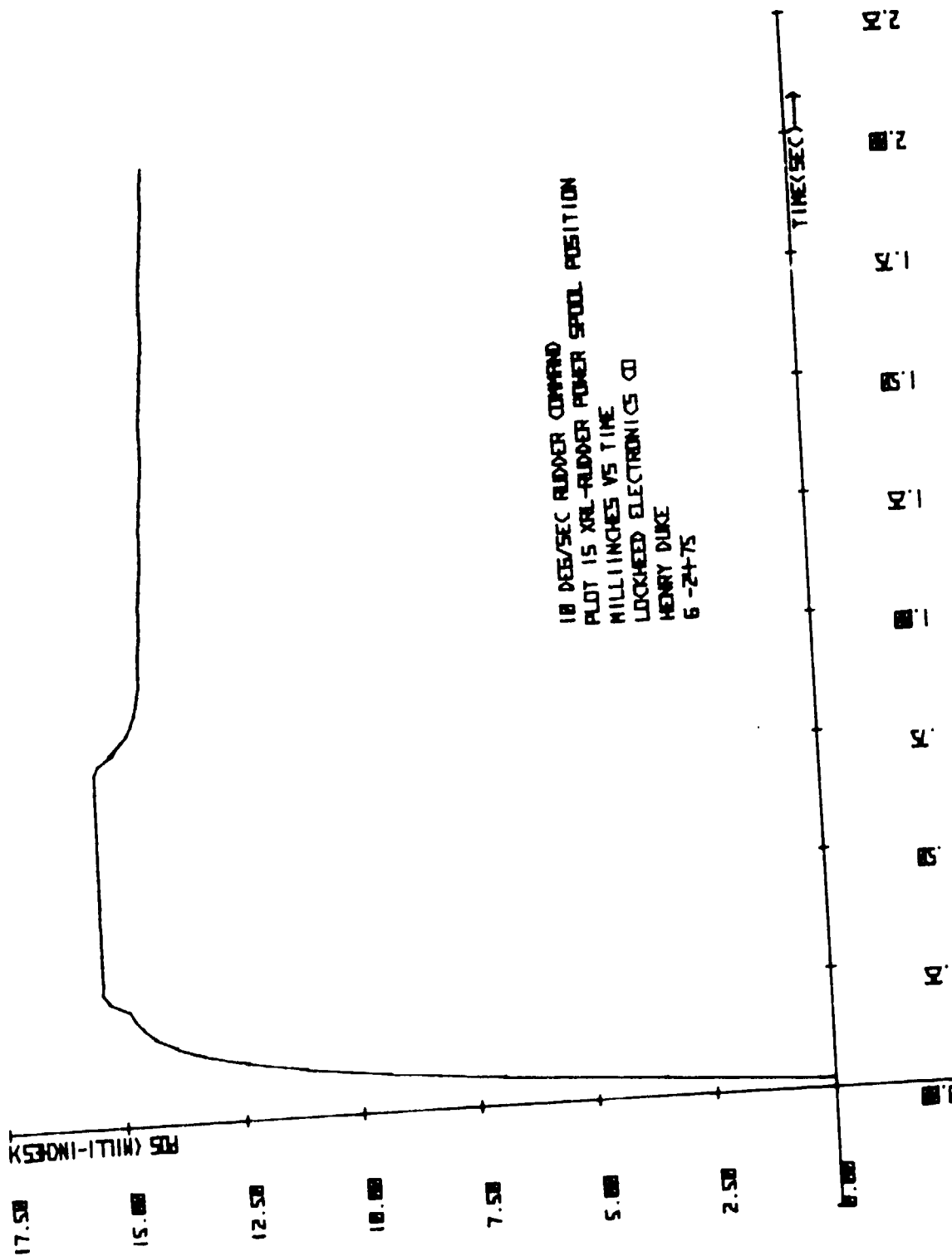


Figure D-17. - Rudder power spool position for a 10-deg/sec Rudder input.

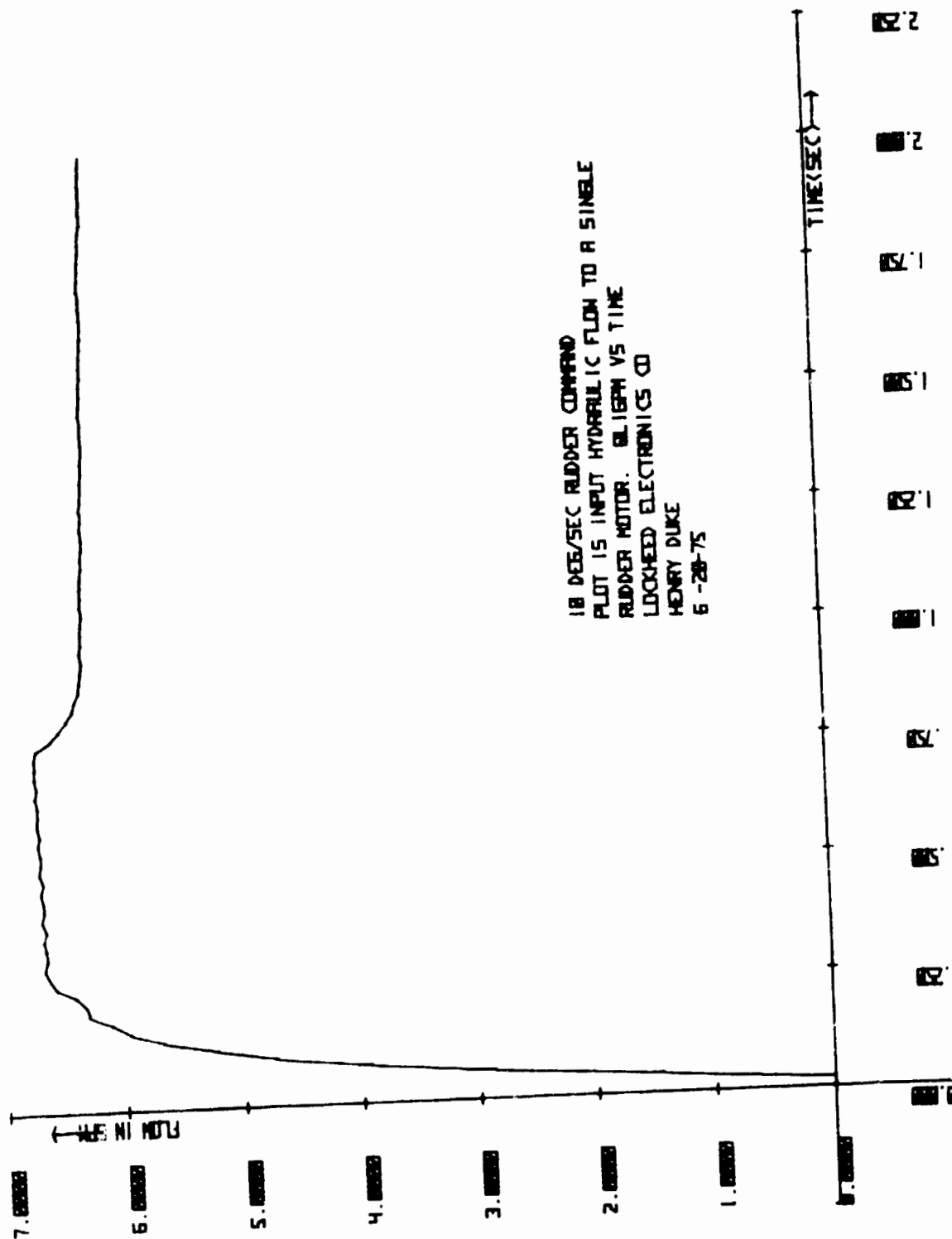


Figure D-18. - Input hydraulic flow to a single Rudder hydraulic motor for a 10-deg/sec Rudder input.

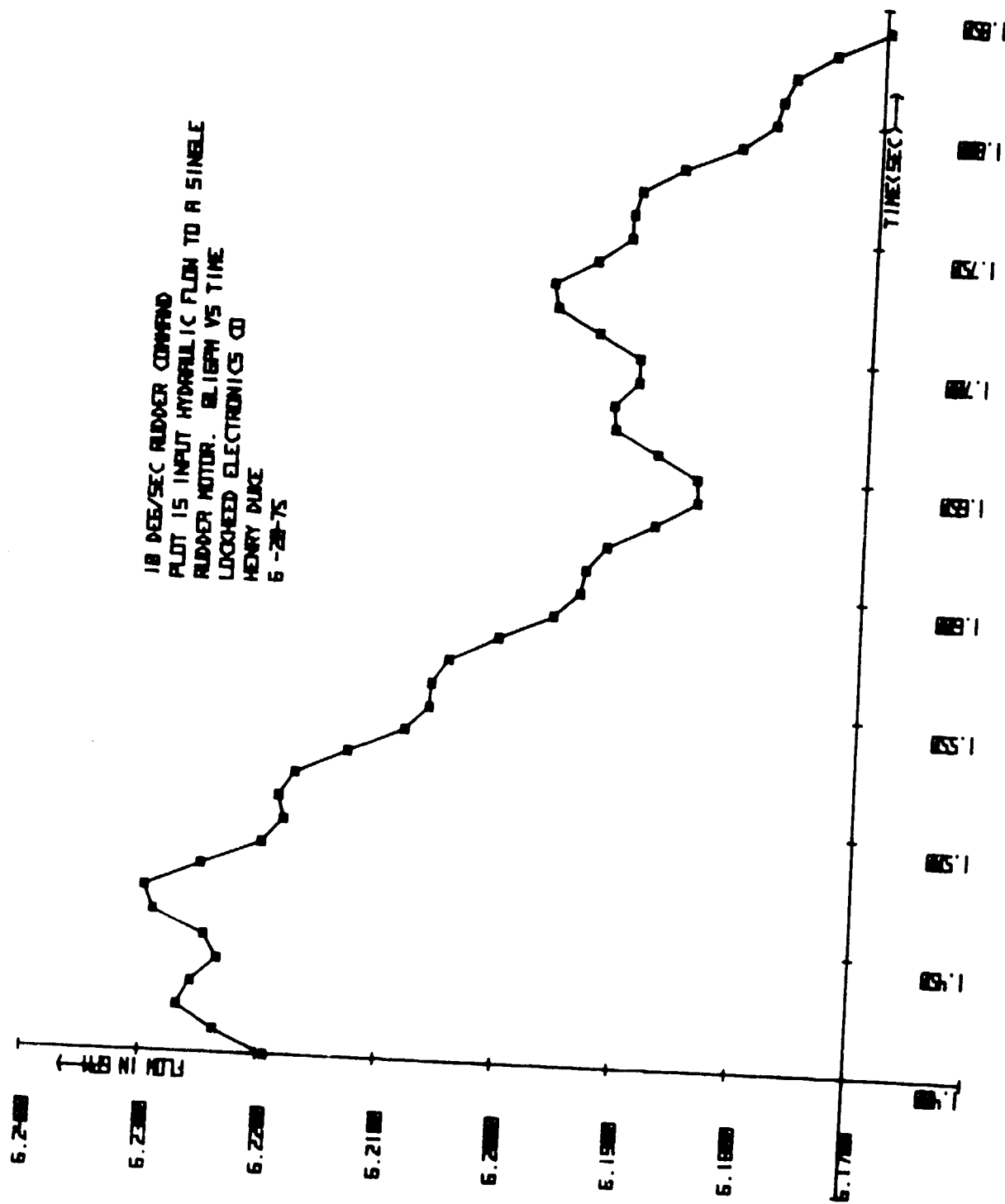


Figure D-19. - Input hydraulic flow to a single Rudder hydraulic motor for a 10-deg/sec Rudder input.

APPENDIX E

MOTOR HYDRAULIC INPUT VS MOTOR SPEED

APPENDIX E FIGURES

Figure	Page
E-1 Hydraulic input flow vs motor speed	E-4

From ref 5, page 65, the volumetric displacement is equal to:

$$D_m = \frac{Q_\ell}{\dot{\theta}_m} \frac{\text{gal}}{\text{rev}}$$

where:

D_m = volumetric displacement, $\frac{\text{gal}}{\text{rev}}$

Q_ℓ = flow through the motor, $\frac{\text{in}^3}{\text{rad}}$

$\dot{\theta}_m$ = shaft speed of motor, $\frac{\text{rad}}{\text{sec}}$

For a shaft speed of 2853 rpm with D_m given as 0.52 (ref 1), the flow is therefore:

$$\begin{aligned} Q_\ell &= D_m \dot{\theta}_\ell \text{ mass} \\ &= (0.52)(2853) \frac{\text{IN}^3}{\text{rev}} \frac{\text{rev}}{\text{min}} \times \frac{1 \text{ gal}}{231 \text{ IN}^3} \\ &= 6.42 \text{ gpm} \end{aligned}$$

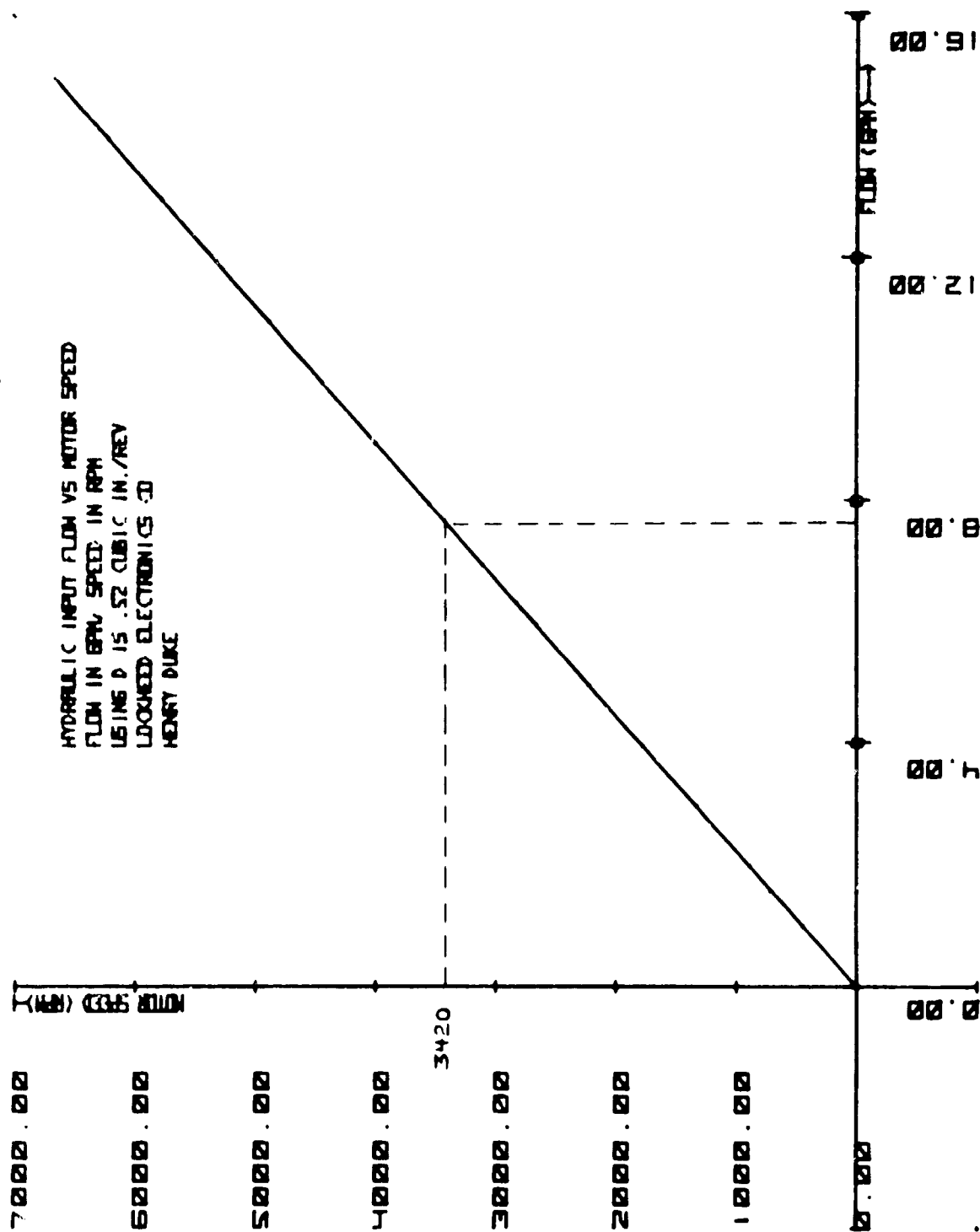


Figure E-1. - Hydraulic input flow vs motor speed.

APPENDIX F

LIST OF CSMP CONSTANTS AND VARIABLES

Signature	Type	Definition	Value	Units
AC	X	AREA OF ΔP PISTON	0.008975	IN ²
AP	X	AREA OF SECOND STAGE SPOOL	0.0276	IN ²
AS	X	AREA OF MOD PISTON	0.193	IN ²
BETA	X	FLUID BULK MODULUS	1.8E5	$\frac{LB}{IN^2}$
BM	X	MOTOR VISCOUS FRICTION COEFFICIENT	0.015	IN-LB- SEC
BP	X	DAMPING, SECOND STAGE SPOOL	0.0648	$\frac{LB-SEC}{IN}$

LEGEND: S = Speedbrake only
 RD = Rudder only
 X = Rudder and Speedbrake

L = Left Panel
 R = Right Panel
 M = Multiple use 1,2,3,4
 N = Multiple use 1,2,3

V = variable, no constant value

Signature	Type	Definition	Value	Units
BPP	X	VISCOUS DAMPING, EACH PANEL	15500	IN-LB- SEC
BR	X	DAMPING, TOTAL MOD PISTONS AND POWER SPOOL	1.386	LB-SEC IN
C1	X	FLOW DISPLACEMENT COEFFICIENT OF FLAPPER NOZZLE	185.2	IN ² LB-SEC
C2	X	FLOW PRESSURE CHARACTERISTIC OF NOZZLE	0.1096	IN ⁴ LB-SEC
C3	X	FLOW PRESSURE CHARACTERISTIC OF FIXED RESTRICTION	8.76E- 5	IN ⁵ LB-SEC
CONV	X	CONVERSION FACTOR FROM IN ³ TO GPM	0.2597	GPM-SEC IN ³

LEGEND: S = Speedbrake only L = Left Panel V = Variable, no
RD = Rudder only R = Right Panel constant value
X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
CP	X	FLOW PRESSURE CHARACTERISTIC FOR SECOND STAGE SPOOL	3.03E-5	$\frac{\text{IN}^5}{\text{LB} \cdot \text{SEC}}$
CQ	X	FLOW DISPLACEMENT CHARACTERISTIC FOR SPOOL VALUE	4.5866	$\frac{\text{IN}^3}{\text{SEC/LB}}$
CRPM		CONVERSION FACTOR FROM RAD/SEC TO RPM	9.5492	$\frac{\text{RPM}}{\text{RAD/SEC}}$
D	X	MOTOR VOLUMETRIC DISPLACEMENT	0.08276	$\frac{\text{IN}^3}{\text{RAD}}$
DDL P	L	RATE OF CHANGE OF PANEL MOVEMENT, PANEL SPEED	V	$\frac{\text{RAD}}{\text{SEC}}$
DDR P	R		V	$\frac{\text{RAD}}{\text{SEC}}$
DELTR	RD	SUMMER OUTPUT INCLUDING HYSTERESIS RADIAL POSITION	V	RAD
DELTS	S		V	RAD
DELTRN	RD,N	OUTPUT OF INDIVIDUAL MOTORS ACROSS SUMMER GEAR RATIO, RADIAL POSITION	V	RAD
DELTSN	S,N		V	RAD

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, no
 RD = Rudder Only R = Right Panel constant value
 X = Rudder and Speedbrake M = Multiple use 1,2,3,4
 N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
DELTXN	RD,N	POWER SPOOL OFFSET	0.0	IN.
DLP	L	LEFT AND RIGHT PANEL POSITION IN RADIANS	V	RADIANS
DRP	R			
DRFRL	RD	ANGLE BETWEEN LEFT AND RIGHT HAND PANELS IN FRL	V	DEGREES (FRL)
DSBFRL	S			
DRRHL	RD	ANGLE BETWEEN LEFT AND RIGHT HAND PANELS IN RHL	V	DEGREES (RHL)
DSBRHL	S			

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
 RD = Rudder Only R = Right Panel Constant value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

LEGEND:	S = Speedbrake Only	L = Left Panel	V = Variable, No
	RD = Rudder Only	R = Right Panel	Constant Value
	X = Rudder and Speedbrake	M = Multiple Use 1,2,3,4	
		N = Multiple Use 1,2,3	

Signature	Type	Definition	Value	Units
FR	RD	POWER SPOOL POSITION ERROR	V	LB
PS	S			
FRC	X	POWER SPOOL STATIC AND COULOMB STARTING FRICTION	13.0	LB.
FRL	X	FUSILAGE REFERENCE LINE	--	--
FRTOT	RD	TOTAL OUTPUT FORCE FROM 4 SERVO VALUES	V	LB.
FSTOT	S			
G1	X	FLOW PRESSURE CHARACTERISTIC USED IN SPOOL VALUE TO REPRESENT SUM OF FIXED RESTRICTION AND NOZZLE FLOW PRESSURE CHARACTERISTICS	59.24	$\frac{IN^5}{LB-SEC}$
G2	X	SECOND STAGE SERVO VALUE VOLUMETRIC REACTION TO FIRST STAGE PRESSURE DRIVE	$1.72E^{-6}$	$\frac{IN^3}{P.S.I.}$
GH	X	POWER HINGE GEAR RATIO	473.92L:1	--

LEGEND: S - Speedbrake Only L - Left Panel V = Variable, No
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
GMR	RD	MIXER GEAR TRAIN RATIO	1.47:1	--
GMS	S		4.46:1	--
HH	X	POWER HINGE HYSTERESIS	2.9088-E-3	RAD
HINGE	X	HINGE MOMENT RAMP FUNCTION	V	IN.LB DEG
HLP	L	HINGE MOMENT	V	IN.LB.
HRP	R			
HMLP	L	HINGE MOMENT	V	IN.LB.
HMRP	R			
HMR	RD	MIXER GEAR TRAIN HYSTERESIS	0.0208	RAD
HMSB	S		0.0208	RAD
IL	X	CURRENT LIMIT OF FLAPPER TORQUE MOTOR IN SERVO VALVE	8.0	MA

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
JMR	RD	MOTOR MOMENT OF INERTIA	0.00636	IN-LB- SEC ²
JMS	S		0.00565	IN-LB- SEC ²
JP	X	MOMENT OF INERTIA OF INDIVIDUAL PANEL	2579.0	IN-LB- SEC ²
KAMP	X	ASA GAIN	17.5	$\frac{mA}{VOLT}$
KB	X	BERNOULLI FORCE COEFFICIENT	0.52	IN.
KC	X	DYNAMIC LOAD DAMPING GAIN	1.91617	$\frac{mA}{VOLT}$
KFB	RD	ACTUATOR POSITION TRANSDUCER GAIN	10.571177	$\frac{VOLTS}{RADIAN}$
KFBSB	S		5.81093	$\frac{VOLTS}{RADIAN}$

LEGEND: S = Speedbrake Only L=Left Panel V= Variable, No
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

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Signature	Type	Definition	Value	Units
KFL	X	FLAPPER VALUE LIMITS OF SECONDARY ACTUATOR	0.0011	IN.
KHC	X	SPRING CONSTANT OF ROTARY ACTUATORS	0.397E+8	$\frac{\text{IN-LB}}{\text{RAD}}$
KHST	X	HYSTERESIS CONSTANT FOR TORQUE MOTOR-COIL HYSTERESIS	0.02	IN-LB
KN	X	NOZZLE PRESSURE FEEDBACK CONSTANT	1.38E-4	IN ³
KP	X	TOTAL SPRING RATE OF SECOND STAGE SPOOL	1200.0	$\frac{\text{LB.}}{\text{IN.}}$
KPQ	X	MOTOR LEAKAGE COEFFICIENT	0.004	$\frac{\text{IN}^5}{\text{LB-SEC}}$

LEGEND: S = Speedbrake Only
 RD = Rudder Only
 X = Rudder and Speedbrake

L = Left Panel
 R = Right Panel
 M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

V = Variable, NO Constant Value

Signature	Type	Definition	Value	Units
KPT	X	PRESSURE TRANSDUCER GAIN	0.00167	$\frac{\text{VOLTS}}{\text{PSI}}$
KQP	X	RUDDER POWER SPOOL FLOW GAIN	33.2	$\frac{\text{IN}^2}{\text{SEC/LB}}$
KR	RD	CONSTANT TO CONVERT RHL TO FR' FOR PANEL ANGLE (RUDDER) OR INCLUDED ANGLE (SPEEDBRAKE)	0.841328	--
KSB	S		0.88438	--
KFBL	X	FEEDBACK GAIN FROM POWER SPOOL POSITION	6.22	$\frac{\text{IN-LB}}{\text{IN.}}$
KRV	RD	CHANGE DEGREES INPUT COMMAND TO VOLTS	0.1845	$\frac{\text{VOLTS}}{\text{DEGREE}}$
KSBV	S		0.10142	$\frac{\text{VOLTS}}{\text{DEGREE}}$
KTM	X	CHANGE INPUT CURRENT TO TORQUE. TRANSLA- TIONAL GAIN OF TORQUE MOTOR IN SERVO	0.045	$\frac{\text{IN-LB}}{\text{mA}}$

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
RD = Rudder Only R = Right Panel Constant Value
X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
LBP	L	POWER HINGE DAMPED FEEDBACK	V	IN-LB RADIANS
RBP	R		V	IN-LB RADIANS
LD	X	DEMULATOR FILTER DAMPING FACTOR (TIMES 2)	0.707	--
LDL	L	POWER HINGE DYNAMICS POSITION ERROR	V	IN-LB- RADIANS
RDL	R		V	IN-LB- RADIANS
LN	X	FLAPPER POSITION GAIN	0.01693	IN. IN-LB.
LPACC	L	PANEL SPEED (RHL)	V	DEG SEC
RPACC	R			
LPDEG	L	PANEL ANGULAR POSITION (RHL)	V	DEG
RPDEG	R			

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
MP	X	SECOND STAGE SPOOL MASS IN SERVO VALVE	6.83E-5	LB-SEC ² IN
MR	X	TOTAL MASS MOD PISTONS IN SERVO VALVE	2.07E-3	LB-SEC ² IN
PDU	X	POWER DRIVE UNIT	--	--
PLN	RD,N	PRESSURE OUTPUT FROM POWER SPOOL TO MOTOR. PRESSURE DROP ACROSS MOTOR	V	PSI
PSLN	S,N		V	PSI
PLNLIM	RD,N	MOTOR INPUT PRESSURE LIMITER	V	PSI
PSNLIM	S,N		V	PSI
PSN	X,N	HYDRAULIC INPUT PRESSURE TO RUDDER/SPEED-BRAKE SUBSYSTEM	2800	PSI
R1XM	RD,M	SERVO 2ND STAGE FLOW FEEDBACK FROM SERVO MOD PISTON	V	IN ³ /SEC
S1XM	S,M		V	IN ³ /SEC

LEGEND: S = Speedbrake Only L = Lift Panel V = Variable, No
RD = Rudder Only R = Right Panel Constant Value
X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
LPHHS	L	OUTPUT OF POWER HINGE HYSTERESIS. UNLOADED PDU POSITION	V	RADIANS
RPHHS	R		V	RADIANS
LPHIN	L	INPUT TO POWER HINGE HYSTERESIS	V	RADIANS
RPHIN	R		V	RADIANS
LPHS	L	ERROR BETWEEN DRIVE POSITION LPHHS OR RPHHS AND ACTUAL POSITION FEEDBACK DLP AND DRP	V	RADIANS
RPHS	R		V	RADIANS
LPNIX	L	OUTPUT OF MIXER GEAR HYSTERESIS	V	RADIANS
RPNIX	R		V	RADIANS
LPX	L	INPUT TO MIXER GEAR HYSTERESIS	V	RADIANS
RPX	R		V	RADIANS
LTPC	L	FEEDBACK FROM LOAD DYNAMICS STICTION SWITCH	V	IN-LB- RADIANS
RTPC	R		V	IN-LB- RADIANS

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, Mo
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
RDDSPM	RD,M	SECOND DERIVATIVE OF INTERNAL FORCE OF SERVO 2ND STAGE SPOOL	V	--
SDDSPM	S,M		"	--
RDDVWM	RD,M	SECOND DERIVATIVE OF RUDDER (OR SPEED- BRAKE) SECONDARY DELTA P FEEDBACK	V	--
SDDVWM	S,M		V	--
RDDVZM	RD,M	SECOND DERIVATIVE OF RUDDER (OR SPEED- BRAKE) POSITION FEEDBACK	V	--
SC VZM	S,M		V	--
RDEGM	RD,M	INPUT CONSTANT - USED AS CONSTANT STEP COMMAND	AR	--
SDEGM	S,M		AR	--
RDSPM	RD,M	SECOND DERIVATIVE OF RUDDER (OR SPEED- BRAKE) SERVO MOD PISTON FORCE INPUT	V	--
SDSPM	S,M		V	--
RDVWM	RD,M	1ST DERIVATIVE OF RUDDER (OR SPEEDBRAKE) SECONDARY DELTA P FEEDBACK	V	--
SDVWM	S,M		V	--

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
RD = Rudder Only R = Right panel Constant Value
X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
N = Multiple Use 1,2,3

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Signature	Type	Definition	Value	Units
RDVKM	RD,M	1ST DERIVATIVE OF RUDDER (OR SPEEDBRAKE) POSITION FEEDBACK	V	--
SDVKM	S,M		V	--
RDVKIM	RD,M	DERIVATIVE OF DELTA P DEMOD OUTPUT RVKM	V	--
SDVKIM	S,M		V	--
RDVZM	RD,M	DERIVATIVE OF POSITION FEEDBACK DEMOD OUTPUT RVZM	V	--
SDVZM	S,M		V	--
RFMFLW	RD,M	HYDRAULIC FLOW INTO SERVO VALVE 2ND STAGE SPOOL	V	IN ³ /SEC
SFMFLW	S,M		V	IN ³ /SEC
RFMFT	RD,M	FLOW-FORCE INPUT FOR SERVO VALVE MOD PISTON	V	IN ⁵ SEC
SFMFT	S,M		V	IN ⁵ SEC
RDXSAM	RD,M	DERIVATIVE OF FLOW PER UNIT AREA INTO SERVO VALVE MOD PISTON RXSAM, SXSAM	V	--
SDXSAM	S,M		V	--

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
RD = Rudder Only R = Right Panel Constant Value
X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
RPN	RD,N	FORCE OUTPUT OF SERVO VALUE SECONDARY ACTUATOR	V	LB
SPN	S,N		V	LB
RBF	RD	TOTAL BERNOUILLI FORCE FEEDBACK TO INPUT OF POWER SPOOLS	V	LB
SBF	S		V	LB
RFTM	RD,M	FORCE FEEDBACK TO FLAPPER VALVE (1ST STAGE VALUE) IN SERVO SECONDARY ACTUATOR	V	IN-LB
SFTM	S,M		V	IN-LB
RFXM	RD,M	FLAPPER POSITION IN SERVO SECONDARY ACTUATOR	V	IN
SFXM	S,M		V	IN
RFXLMM	RD,M	FLAPPER VALVE LIMITER	V	IN
SFXLMM	S,M		V	IN
RPPM	RD,M	FLUID PRESSURE INTO SERVO SECONDARY ACTUATOR SECOND STAGE	V	PSI
SPPM	S,M		V	PSI

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

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Signature	Type	Definition	Value	Units
RHL	X	RUDDER HINGE LINE	--	--
QLN	RD,N	LOAD FLOW INTO HYDRAULIC MOTORS	V	IN ³ /SEC
QSLN	S,N		V	IN ³ /SEC
QLNGPM	RD,N	LOAD FLOW INTO HYDRAULIC MOTORS	V	GPM
QSLNGM	S,N		V	GPM
RILM	RD,M	LIMITED CURRENT INTO SECONDARY ACTUATOR TORQUE MOTORS	V	MA
SILM	S,M		V	MA
RINM	RD,M	INPUT COMMAND TO RUDDER/SPEEDBRAKE SUBSYSTEM	V	DEG
SINM	S,M		V	DEG
RLVM	RD,M	DERIVATIVE OF POSITION FEEDBACK RVDT OUTPUT	V	Vrms
SLVM	S,M		V	Vrms

LEGEND: S = Speedbrake Only
 RD = Rudder Only
 X = Rudder and Speedbrake

L = Left Panel
 R = Right Panel
 M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

V = Variable, No Constant Value

Signature	Type	Definition	Value	Units
RLVDTM	RD, M	DERIVATIVE OF RVDT OUTPUT	V	--
SLVDTM	S, M		V	--
PMINN	RD, N	FORCE INPUT TO MOTOR TO CAUSE ONE RADIAN OF ROTATION	V	$\frac{\text{IN-LB}}{\text{RAD}}$
SMINN	S, N		V	$\frac{\text{IN-LB}}{\text{RAD}}$
RMNRPM	RD, N	MOTOR SPEED	V	RPM
SMNRPM	S, N		V	RPM
RMSUMP	RD, P	MOTOR DYNAMICS FEEDBACK. 1, 3 AND 5 ARE TORQUE FEEDBACKS FROM PANELS WHERE 2, 4 AND 6 ARE MOTOR DYNAMICS FEEDBACK	V	--
SMSUMP	S, P		V	--
RPVSN	RD, N	FEEDBACK CONSTANT OF SECONDARY ACTUATOR SERVO MOD PISTON	V	PSI
SPVSN	S, N		V	PSI
RQXSAM	RD, M	FLOW INTO SECONDARY ACTUATOR SERVO MOD PISTON	V	IN^3/SEC
SQXSAM	S, M		V	IN^3/SEC

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
RD = Rudder Only R = Right Panel Constant Value
X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
RSAPM	PD,M	INTERNAL FLOW IS SECONDARY ACTUATOR MOD PISTON	V	IN^3/SEC
SSAPM	S,M		V	IN^3/SEC
RSPM	RD,M	SECONDARY ACTUATOR MOD PISTON DYNAMICS FEEDBACK	V	--
SSPM	S,M		V	--
RSVPF	RD,M	FUNCTION SWITCH FOR POWER SPOOL INPUT RATE OF CHANGE	V	LB
SSVPF	S,M		V	LB
RSVIM	RD,M	POWER SPOOL FLUID VOLUME	V	IN^3
SSVIM	S,M		V	IN^3
RSVMIM	RD,M	POWER SPOOL OUTPUT FLOW	V	IN^3/SEC
SSVMIM	S,M		V	IN^3/SEC
RTM	RD,M	TORQUE INPUT TO SERVO SECONDARY ACTUATOR FLAPPER	V	IN-LB
STM	S,M		V	IN-LB

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
RTHETN	RD,N	MOTOR SHAFT POSITION	V	RADIANS
STHETN	S,N		V	RADIANS
RTPN	RD,N	TORQUE FEEDBACK TO MOTOR FROM PANELS	V	IN-LB
STPN	S,N		V	IN-LB
RTINM	RD,M	TORQUE INPUT TO SECONDARY ACTUATOR FLAPPER VALVE IN SERVO	V	IN-LB
STINM	S,M		V	IN-LB
RUDFB	RD	PANEL POSITION FEEDBACK TO RVDT'S	V	RADIANS
SBFB	S		V	RADIANS
RUDSUM	RD	OUTPUT OF SUMMER DIFFERENTIALS WITHOUT HYSTERESIS	V	RADIANS
SBSUM	S		V	RADIANS
RVINM	RD,M	VOLTAGE INPUT COMMAND TO ASA FROM MDM. LIMITED TO ± 5.12 VOLTS DC.	V	VOLTS
SVINM	S,M		V	VOLTS

Legend: S = Speedbrake Only L = Left Panel V = Variable, No
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
RVKM	RD, M	DELTA P FEEDBACK CURRENT	V	mA
SVKM	S, M		V	mA
RVLM	RD, M	DELTA P PRESSURE TRANSDUCER FEEDBACK IN VOLTS	V	VOLTS
SVLM	S, M		V	VOLTS
RVZM	RD, M	POSITION FEEDBACK VOLTAGE	V	VOLTS
SVZM	S, M		V	VOLTS
RUD	RD	RADIAL POSITION MIDWAY DOWN MIXER GEAR TRAIN	V	RADIANS
SB	S		V	RADIANS
RXAM	RD, M	MOVEMENT ERROR OF SECONDARY ACTUATOR SERVO MOD PISTON	V	IN/SEC
SXAM	S, M		V	IN/SEC
RXSAM	RD, M	MOVEMENT OF SECONDARY ACTUATOR SERVO MOD PISTON	V	IN/SEC
SXSAM	S, M		V	IN/SEC

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
<u>RXM</u>	RD,M	POWER SPOOL POSITION	V	IN
<u>SXM</u>	S,M		V	IN
<u>RXSSM</u>	RD,M	SERVO SECONDARY ACTUATOR 2ND STAGE SPOOL POSITION	V	IN
<u>SXSSM</u>	S,M		V	IN
<u>STHDN</u>	S,	SPEED OF INDIVIDUAL MOTORS	V	$\frac{\text{RADIANS}}{\text{SEC}}$
<u>THDN</u>	RD		V	$\frac{\text{RADIANS}}{\text{SEC}}$
<u>RPS</u>	X	SUPPLY PRESSURE IN SERVO SEC. ACTUATOR	2800	PSI

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, NO
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
TAUC	X	DYNAMIC LOAD DAMPING CONSTANT	0.1	SEC
TC		STATIC AND COULOMB FRICTION SWITCH IN MOTOR DYNAMICS	9.0	$\frac{\text{RAD}}{\text{SEC}}$
TDT		RVDT TIME CONSTANT	0.004	SEC
THDOTN	RD, N	ANGULAR POSITION OF INDIVIDUAL MOTORS	V	RADIANS
THDTSN	S, N		V	RADIANS
TLP	L	PANEL OUTPUT TORQUE	V	IN-LB
TRP	R		V	IN-LB
TPC	X	STATIC AND COULOMB FRICTION SWITCH IN LOAD DYNAMICS	0.0	DEG

LEGEND: S = Speedbrake Only
 RD = Rudder Only
 X = Rudder and Speedbrake
 L = Left Panel
 R = Right Panel
 M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3
 V = Variable, No Constant Value

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Signature	Type	Definition	Value	Units
VM	X	MOTOR FLUID VOLUME (INCLUDES FLUID LINES)	3.88	IN ³
VT	X	SPOOL AND NOZZLE FLUID VOLUME OF SERVO VALVE	0.0838	IN ³
VT2	X	MOD PISTON VOLUME OF SERVO VALVE	0.62	IN ³
WD	X	NATURAL FREQUENCY OF DEMODULATOR FILTER	314.0	$\frac{\text{RAD}}{\text{SEC}}$
XO	X	INITIAL FLAPPER DISPLACEMENT IN SERVO VALVE	0.00185	IN
XR	RD	POWER SPOOL POSITION	V	IN
XS	S		V	IN

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, NO
RD = Rudder Only R = Right Panel Constant Value
X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
N = Multiple Use 1,2,3

Signature	Type	Definition	Value	Units
XRD	RD	POWER SPOOL POSITION CHANGE (SPEED)	V	IN/SEC
XSD	S		V	IN/SEC
XRDDOT	RD	POWER SPOOL ACCELERATION	V	IN/SEC
XSDDOT	S		V	IN/SEC
XRDPB	RD	POWER SPOOL SPEED FEEDBACK FROM SPOOL DYNAMICS	V	IN/SEC
XSDPB	S		V	IN/SEC
XRDOT	RD	POWER SPOOL DYNAMICS SWITCH	V	--
XSDOT	S		V	--
XRL	RD	POWER SPOOL POSITION LIMITED TO ±XRM	V	IN
XSL	S		V	IN
XRM	RD	POWER SPOOL POSITION LIMIT	0.065	IN
XSM	S		0.065	IN

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
RD = Rudder Only R = Right Panel Constant Value
X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
N = Multiple USE 1,2,3

Signature	Type	Definition	Value	Units
Y1	RD	POWER SPOOL LIMITING CONSTANT	1.0	LB
Z1	S		1.0	LB
Y2	RD	POWER SPOOL LIMITING CONSTANT	1.0	LB
Z2	S		1.0	LB

LEGEND: S = Speedbrake Only L = Left Panel V = Variable, No
 RD = Rudder Only R = Right Panel Constant Value
 X = Rudder and Speedbrake M = Multiple Use 1,2,3,4
 N = Multiple Use 1,2,3

APPENDIX G

DISCUSSION OF USING HP9820 FOR PLOTTING

In the preparation of this report, the author has made extensive use of the Hewlett Packard HP9820 calculator and HP9866A Calculator plotter. This was accomplished to remove data from the IBM 360/75 printer plots and to clarify this information. The line printer can plot 10 characters per inch, but cannot plot between these characters whose plotted position is approximate. The HP9820 plots with such accuracy that the error is virtually undetectable and hence, a more accurate plot is produced from which information can be extracted. Figure G-1 (page G-3) is the basic listing of the HP9820 plotting program.

Another useful function of the HP9820 was to determine the average value of a number of points. This listing appears as figure G-2 (page G-7). Both programs have been loaded onto cassette magnetic tape for convenience.

```

01: REW 1SPC 4:PRT " FILE 5", "PLOTING PROGRAM"10 TO
    X TO R19 TO Z11 TO R24 [
11: SPC 1:ENT "X MAX",R1,"X MIN",R2 [
21: ENT "NUMBER POINTS",R27,"Y MAX",R5,"Y MIN",R6 [
31: FXD 0:PRT "TOTAL POINTS =",R27:FXD 08:PRT "+--++",
    "X MAX",R1,"X MIN",R2 [
41: PRT "Y MAX",R5,"Y MIN",R6:SPC 3 [
51: ENT "DELTA X TIC",R3:ENT "DELTA Y TIC",R7:ENT "X S
    TEP =",R18 [
61: ENT "NO DECIMALS X",R11:ENT "NO DECIMALS Y",R12 [
71: PRT "CHAR CODE", "1 PLOT X", "2 PLOT +" [
81: PRT "3 PLOT 0", "4 PLOT D", "5 PLOT C", "6 PLOT Y" [
91: PRT "7 PLOT U", "8 PLOT .", "9 PRINT LINE 54":SPC 1 [
101: (R1-R2)/R3 TO R9:(R5-R6)/R7 TO R10 [
111: (R1-R2)/10 TO A:(R5-R6)/10 TO B:0 TO C TO R0 [
121: 1+C TO C:JMP CR3>A [
131: 1+R0 TO R0:JMP R0R7>B [
141: R2-CR3 TO R4:R6-R0R7 TO R8 [
151: SCL R4,R1,R8,R5 [
161: AXE R2,R6,R3,R7:.0032(R1-R4) TO R0:0 TO B:FXD R11 [
171: LTR R2+BR3-2R0,R8,212:PLT R2+BR3:JMP (1+B TO B)>R9 [
181: .0032(R5-R8) TO A:0 TO B:FXD R12 [
191: LTR R4,R6+BR7-4A,211:PLT R6+BR7:JMP (1+B TO B)>R10 [
201: SPC 2:PRT "FOR COMPLETE", "LIST OF X,Y", "DATA, DATA
    =1":SPC 1:ENT "DATA?",R23 [
211: ENT "CHAR CODE",C:0 TO B TO R14:IF C=9:GTO 51 [
221: ENT "1 TO CONNECT PTS",B:PEN [
231: ENT "PRINT DECIMALS",R13:PRT "DATA" [

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24: CFG 13: X+R19R18+R2R24 TO X: IF (Z+1 TO Z)>R27: GTO 51 [
25: 0 TO R24: IF R23=0: JMP 2 [
26: FXD 0: PRT "NO. ", Z: FXD R13: PRT "X=", X: ENT "Y=", Y: P
    RT "Y=", Y: SPC 1: 1 TO R19: JMP 2 [
27: ENT "Y=", Y: 1 TO R19: FXD 0: PRT Z: FXD R13 [
28: IF R14=0: JMP 3 [
29: IF B=0: JMP 2 [
30: PLT R15, R16 [
31: PLT X, Y: PEN [
32: LTR X-R0, Y-A, 111 [
33: IF C>1: JMP 2 [
34: PLT "X": GTO 49 [
35: IF C>2: JMP 2 [
36: PLT "+": GTO 49 [
37: IF C>3: JMP 2 [
38: PLT "0": GTO 49 [
39: IF C>4: JMP 2 [
40: PLT "D": GTO 49 [
41: IF C>5: JMP 2 [
42: PLT "C": GTO 49 [
43: IF C>6: JMP 2 [
44: PLT "Y": GTO 49 [
45: IF C>7: JMP 2 [
46: PLT "U" [
47: IF C>8: GTO 54 [
48: PLT " " [
49: PEN : LTR X, Y: PEN : IF Z+1=R27: DSP "LAST POINT", " ",
    "LAST POINT", " " [

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50: R14+1 TO R14;X TO R15;Y TO R16;GTO 24 [
 51: SPC 3;DSP "CONT","CONT"," INUE","CONTINUE";ENT
 "ERRORS?";C;IF C # 1;GTO 66 [
 52: ENT "ERROR X LOCATION",R25,"ERROR Y LOCATION",R26;
 LTR R25,R26,211 [
 53: PLT "PLOT ERROR--";JMP -2 [
 54: LTR R2+.8(R1-R2),R6-.02(R5-R6),211;PLT "TIME(SEC)-
 -*" [
 55: LTR R2-.01(R1-R2),R6+.8(R5-R6),212;PLT "RESPONSE--
 --*" [
 56: ENT "START LABEL X",A,"START LABEL Y",B [
 57: LTR A,B,211;PLT "10 DEG/SEC RUDDER COMMAND" [
 58: LTR A,B-.03(R5-R6),211;PLT "X IS" [
 59: LTR A,B-.06(R5-R6);PLT "+ IS" [
 60: LTR A,B-.09(R5-R6);PLT "LOCKHEED ELECTRONICS CO" [
 61: LTR A,B-.12(R5-R6);PLT "HENRY DUKE" [
 62: PRT "PRINT DATE =1";SPC 2;ENT "DATE ?",R28;FXD 0;I
 F R28=0;GTO 68 [
 63: ENT "MONTH",R29,"DAY",R30,"YEAR",R31;R5-R6 TO R33;LT
 R A,B-.15R33;PLT R29 [
 64: .0096(R1-R2) TO R32;LTR A+2R32,B-.15R33;PLT "-";LTR
 A+3R32,B-.15R33 [
 65: PLT R30;LTR A+5R32,B-.15R33;PLT "-";LTR A+6R32,B-.
 15R33;PLT R31 [
 66: PRT "END =0","NEW PLOT =1","LABEL =2";SPC
 2 [
 67: ENT "DECISION?",R22;IF R22=0;PRT "DECISION END";GT
 0 74 [

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68: IF R22=1:PRT "NEW PLOT":GTO 71 [
69: IF R22=2:PRT "LABEL":GTO 54 [
70: IF R22>2:PRT "DECISION ERROR":GTO 67 [
71: ENT "NUMBER POINTS",R27 [
72: 0 TO X TO R19 TO Z:1 TO R24:ENT "X STEP=",R18 [
73: .0032(R1-R4) TO R0:0 TO B:FXD R11:.0032(R5-R8) TO A:0 TO B
    :FXD R12:GTO 21 [
74: PRT "+-+-+-+-+";SPC 8 [

75: END [

```

```

0:  REW ;PRT "-+-+-+-+", "PROGRAM FOR", "CONTINUO
    US", "AVERAGE" [
1:  PRT "-----";SPC 2 [
2:  0 TO R0;0 TO X;0 TO Y;ENT "DECIMALS",B;FXD B [
3:  R0+1 TO R0 [
4:  ENT "VALUE",RR0;IF FLG 13;JMP 3 [
5:  RR0+Y TO Y [
6:  X+1 TO X;JMP -3 [
7:  Y/X TO Z;PRT "NO POINTS=",X,"AVERAGE=",Z [
8:  PRT "-----END-----";CFG 13 [
9:  PRT "NEW PLOT=1","CONTINUE=2","END=3";ENT "DECISIO
    N?",A [
10: IF A=1;JMP -8 [
11: IF A=2;JMP -8 [
12: PRT "-----STOP-----";SPC 6 [
13: END [

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APPENDIX H

DEADSPACE PLOTS

APPENDIX H FIGURES

Figure		Page
H-1	10-deg/sec Rudder servo valve hysteresis	H-3
H-2	10-deg/sec Rudder summer valve hysteresis. . . .	H-4
H-3	10-deg/sec Rudder mixer valve hysteresis	H-5
H-4	10-deg/sec Rudder PDU valve hysteresis	H-6
H-5	6-deg/sec Speedbrake servo valve hysteresis. . .	H-7
H-6	6-deg/sec Speedbrake summer valve hysteresis . .	H-8
H-7	6-deg/sec Speedbrake mixer valve hysteresis. . .	H-9
H-8	6-deg/sec Speedbrake PDU valve hysteresis. . . .	H-10

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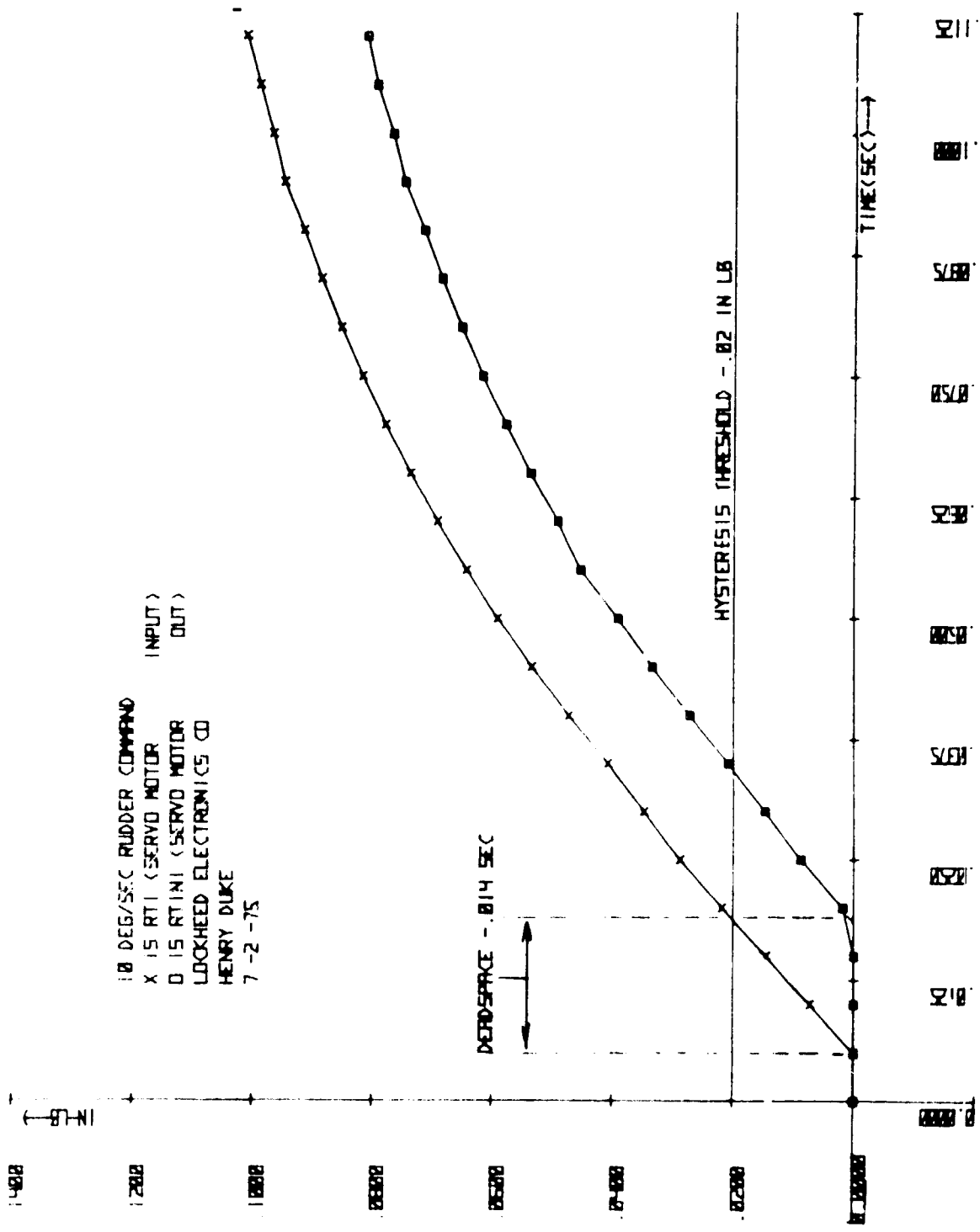


Figure H-1. - 10-deg/sec Rudder servo valve hysteresis.

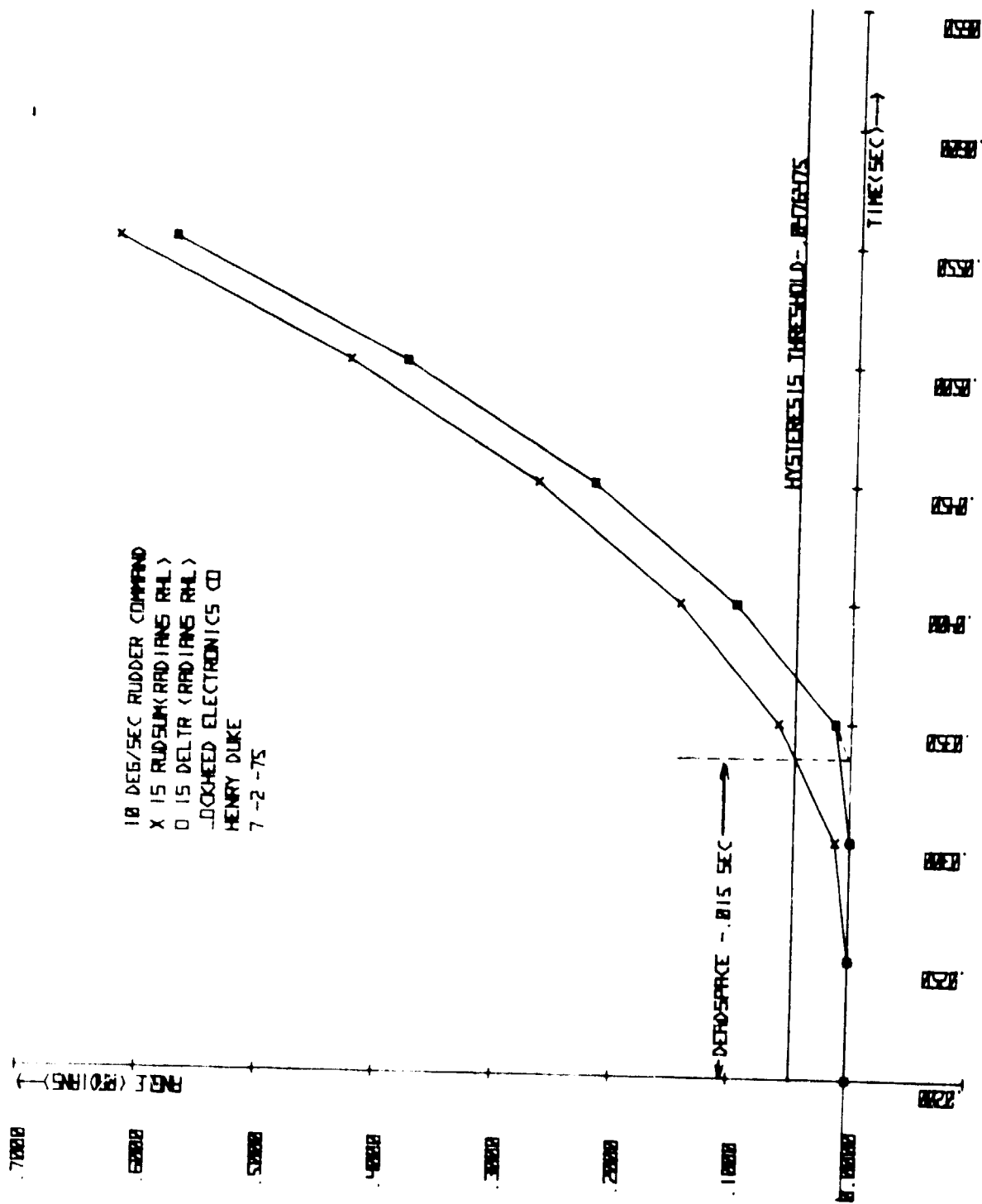


Figure H-2. - 10-deg/sec Rudder summer valve hysteresis.

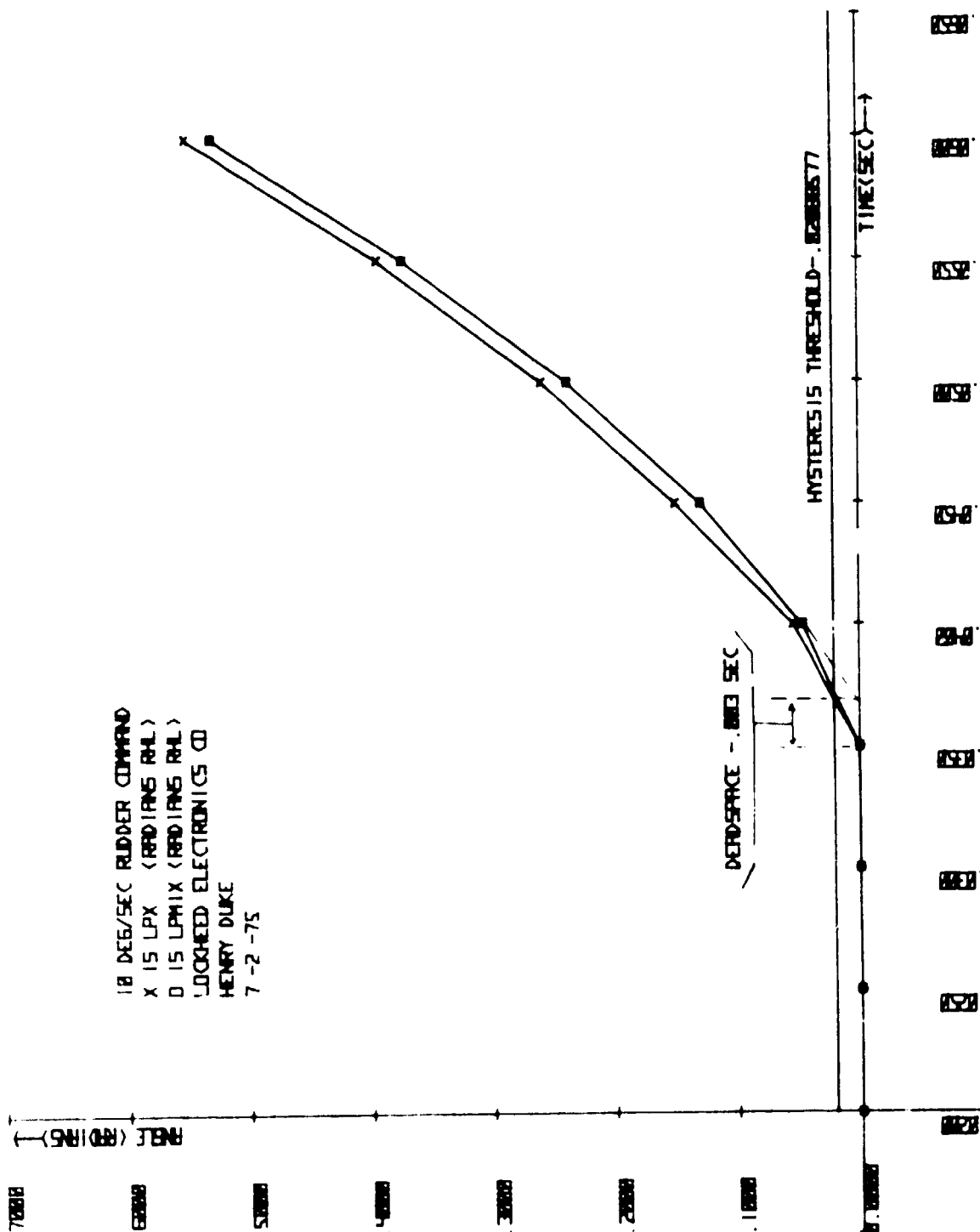


Figure H-3. - 10-deg/sec Rudder mixer valve hysteresis.

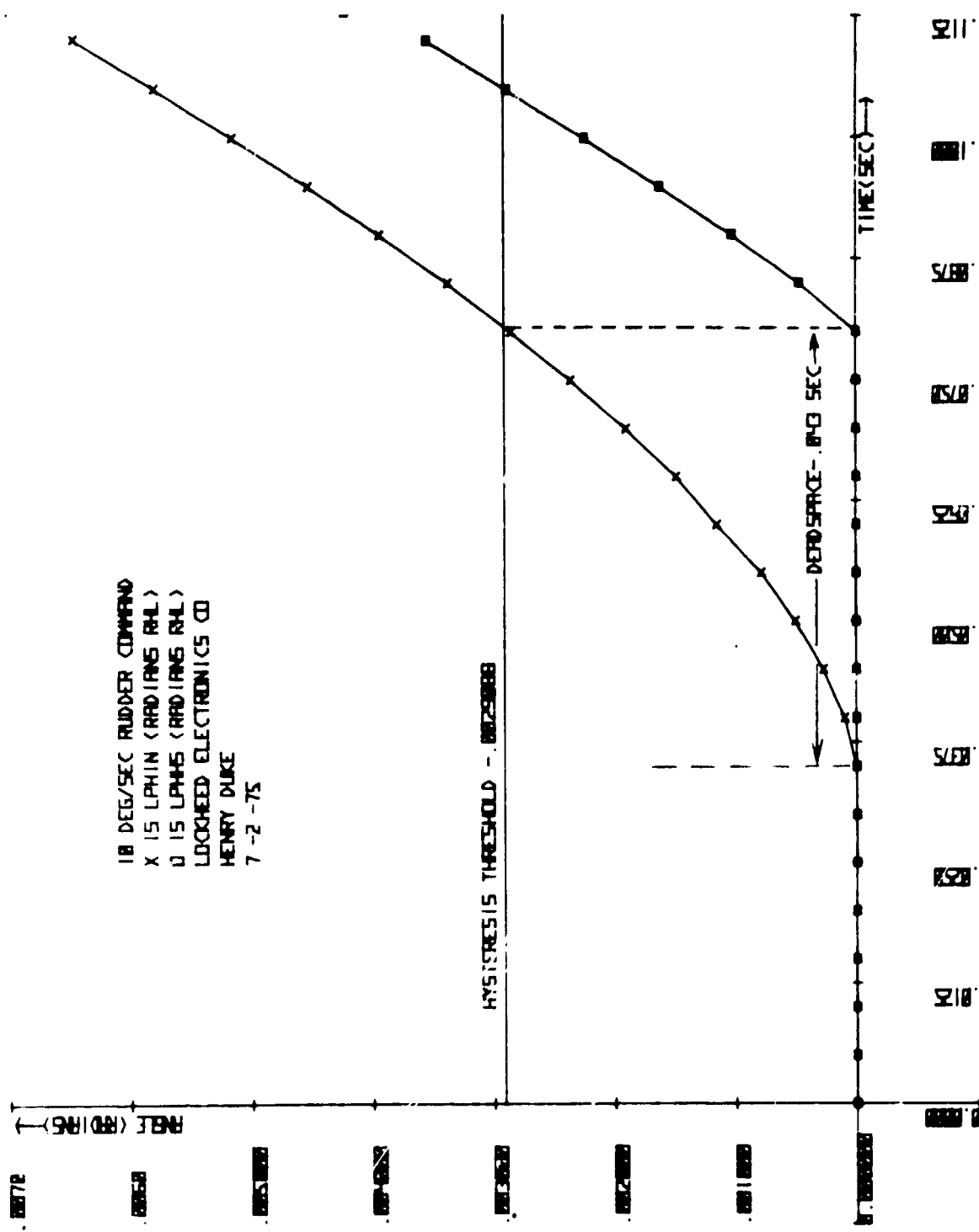


Figure H-4. - 10-deg/sec Rudder PDU valve hysteresis.

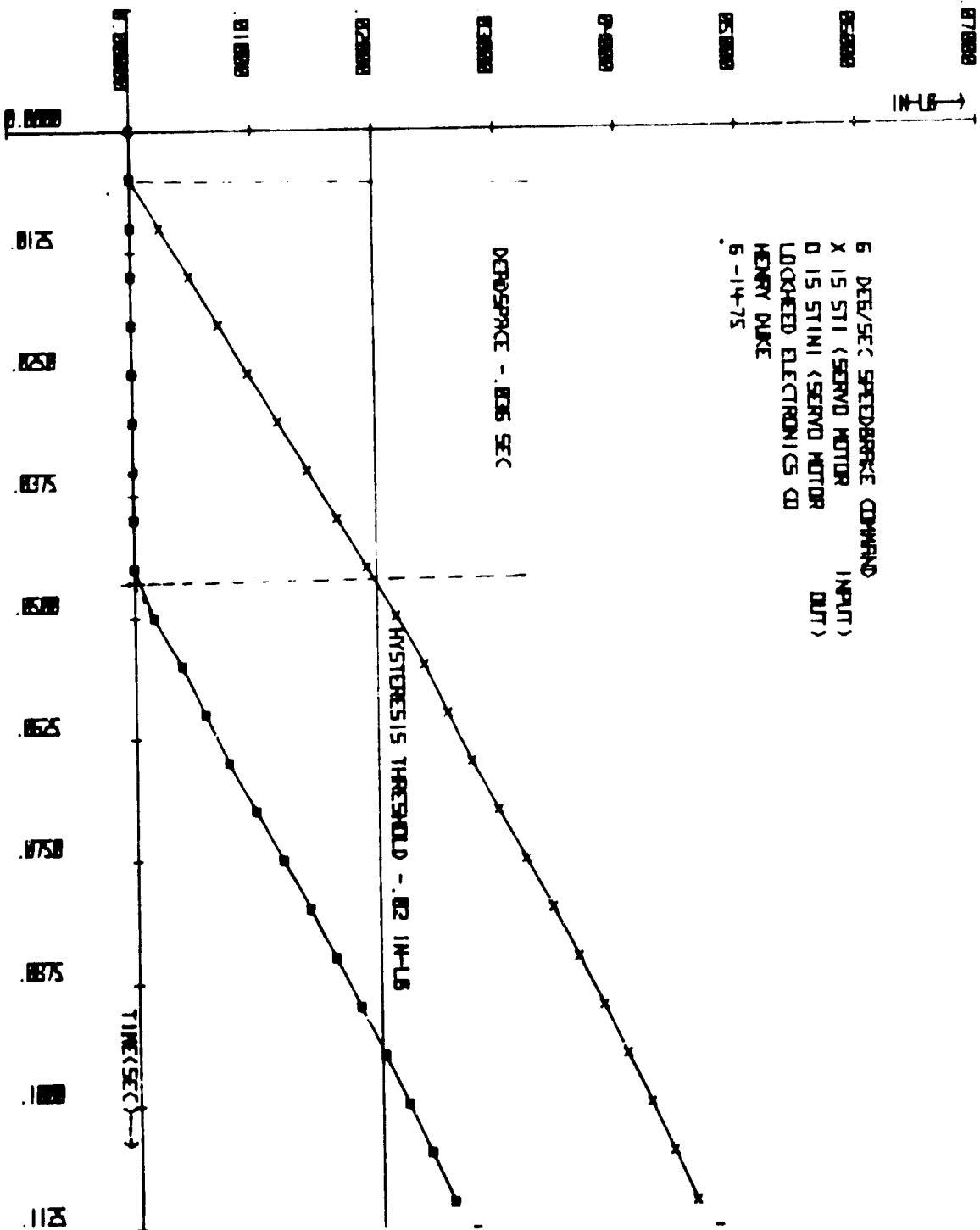


Figure H-5. - 6-deg/sec Speedbrake servo valve hysteresis.

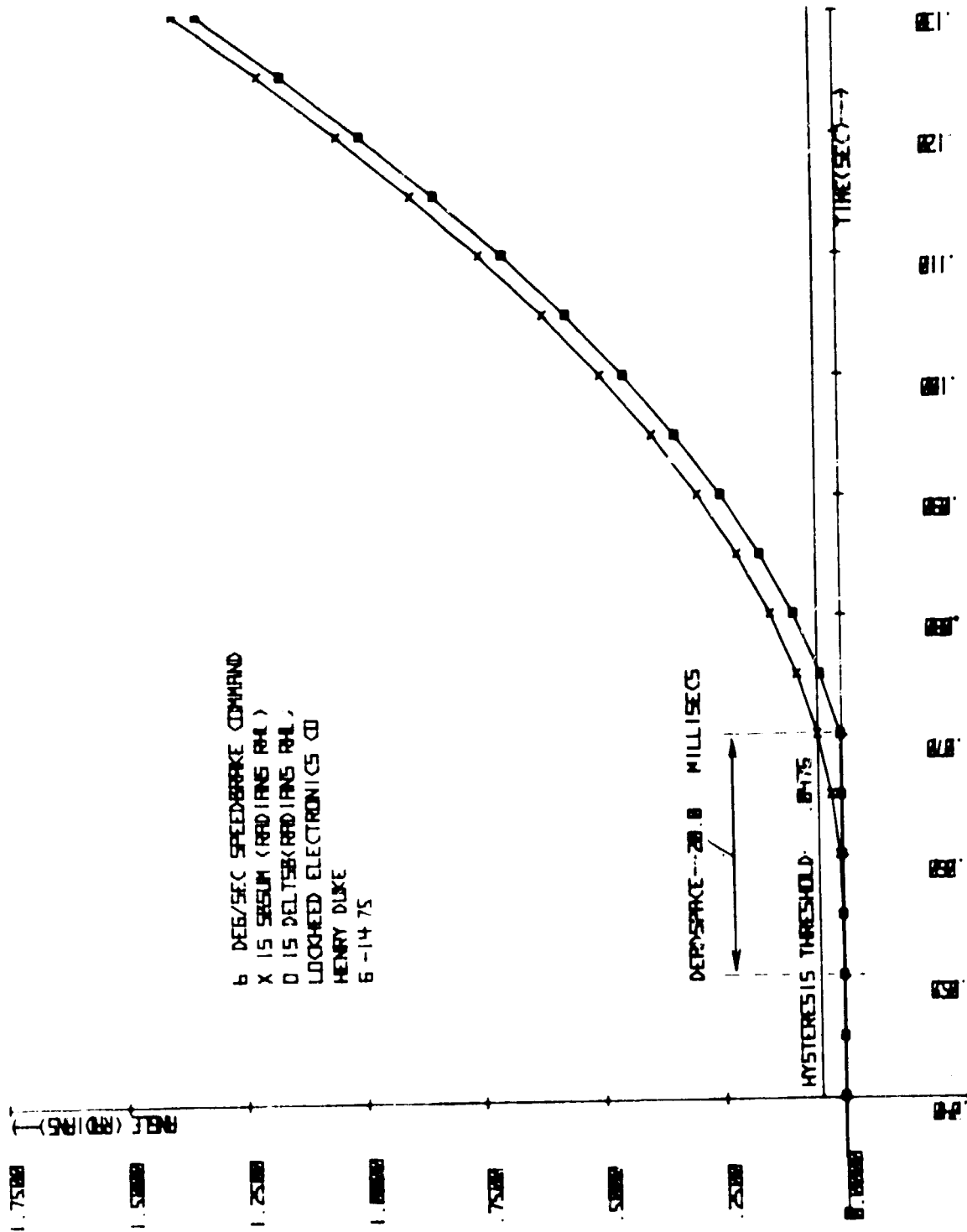


Figure H-6. - 6-deg/sec Speedbrake summer valve hysteresis.

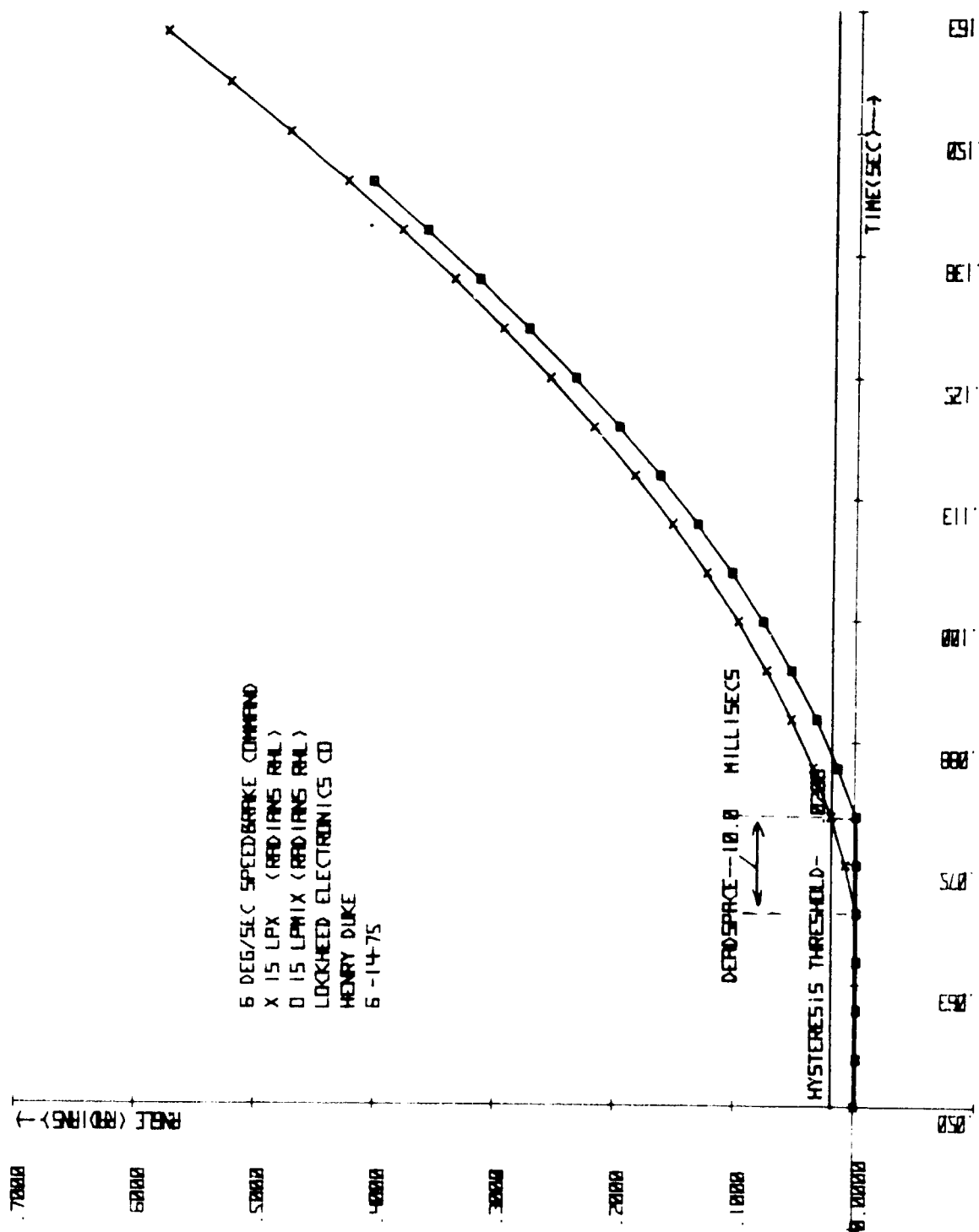


Figure H-7. - 6-deg/sec Speedbrake mixer valve hysteresis.

APPENDIX I

DISCUSSION OF INPUT COMMANDS

APPENDIX I FIGURES

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I-1	Rudder/Speedbrake command channels	I-5
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The input command from the pilot will probably be given as a quick transition from one state to the next in a very short period of time. This is best represented by a step command. This analog step will be sampled by the MDM (MDM #1 as shown in figure I-1) at a 40-millisecond refresh rate and addressed through the IOP to the FCS computer. The computer will then determine the necessary commands to the subsystem to respond to the pilot's requested command.

Since the computer, through software, sets up both position and rate limits, it will determine the magnitude of each step of the staircase commanded to the subsystem. Position commands are set at 27.1 degrees (Rudder) and 49.3 degrees (Speedbrake) with rate limits presently set at 12.1 degrees/second (Rudder), 6.1 degrees/second (Speedbrake opening), and 10.85 degrees/second (Speedbrake closing).

A digital command is then sent through the IOP to MDM #2 where it is converted to the staircase analog command which is addressed into the Rudder/Speedbrake subsystem through the ASA.

The value shown for the staircase voltage will vary from a 10 mV minimum to a maximum set by FCS computer rate limiting. The computer furnishes 10 bits to the MDM of which the highest represents the sign. This leaves 2^9 bits or 512 possible absolute combinations. The weight of each combination is 10 mV, therefore the maximum voltage output will be 5.12 volts. The computer has set a position limit of 27.1 degrees for the Rudder and 49.3 degrees for the Speedbrake, both of which are defined as 5.00 volts (not 5.12 volts). Therefore the actual maximum binary count is 500, not 512.

The refresh rate of the signal applied to the MDM is 40 ms, therefore the time required for each step is 40 ms as set by the FCS computer.

At rate limiting, the slew rate required for each panel is:

$$\text{Rudder} = \frac{(5.0 \text{ volts})(12.1 \text{ degrees})}{(27.1 \text{ degrees})(1 \text{ second})} = 2.232 \frac{\text{volts}}{\text{second}}$$

$$\text{Speedbrake (opening)} = \frac{(5.0 \text{ volts})(6.1 \text{ degrees})}{(49.3 \text{ degrees})(1 \text{ second})} = 0.619 \frac{\text{volts}}{\text{second}}$$

$$\text{Speedbrake (closing)} = \frac{(5.0 \text{ volts})(10.85 \text{ degrees})}{(49.3 \text{ degrees})(1 \text{ second})} = 1.100 \frac{\text{volts}}{\text{second}}$$

The maximum voltage* step in 40 ms is therefore:

$$\text{Rudder} = \frac{(2.232 \text{ volts})(0.04 \text{ seconds})}{(1 \text{ second})} \approx 0.09 \text{ volts}$$

$$\text{Speedbrake (opening)} = \frac{(0.619 \text{ volts})(0.04 \text{ seconds})}{(1 \text{ second})} \approx 0.02 \text{ volts}$$

$$\text{Speedbrake (closing)} = \frac{(1.100 \text{ volts})(0.04 \text{ second})}{(1 \text{ second})} \approx 0.04 \text{ volts}$$

*Rounded to closest multiple of 10 mV.

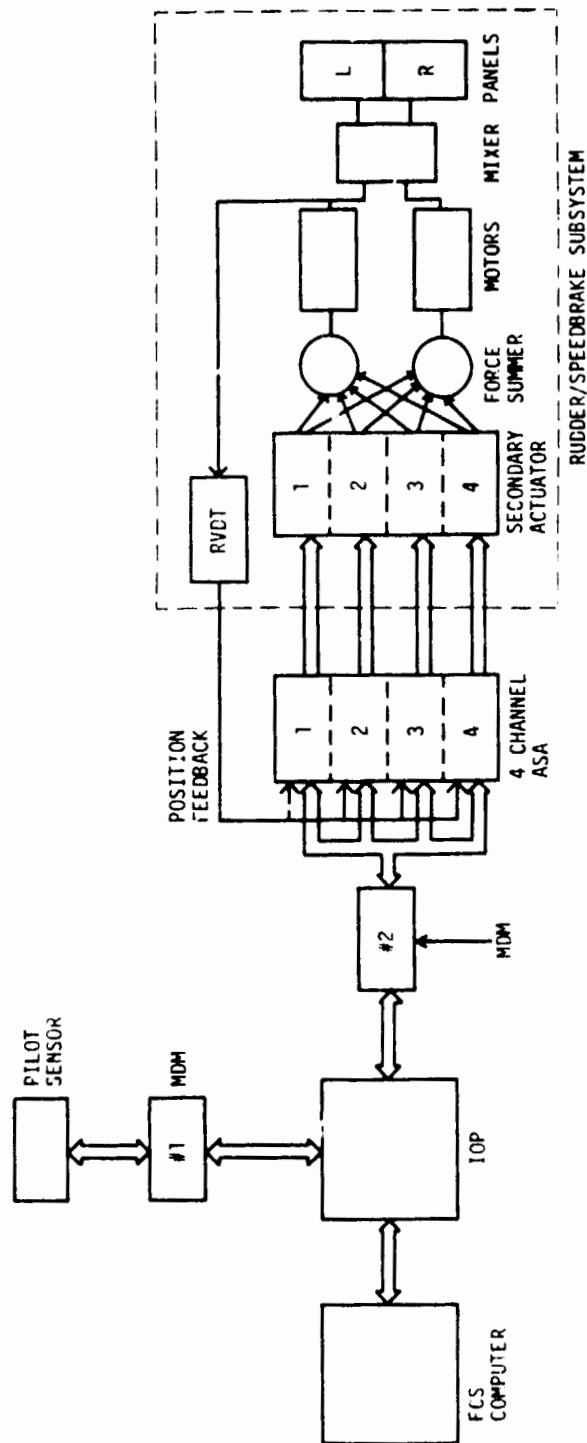


Figure I-1. - Rudder/Speedbrake command channels.

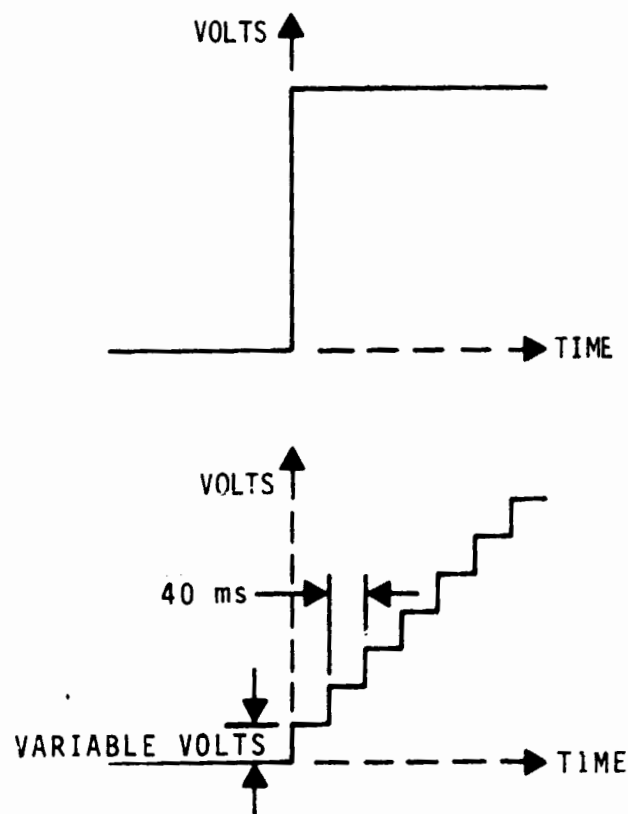


Figure I-2. — Stairstep for Rudder/Speedbrake input.